

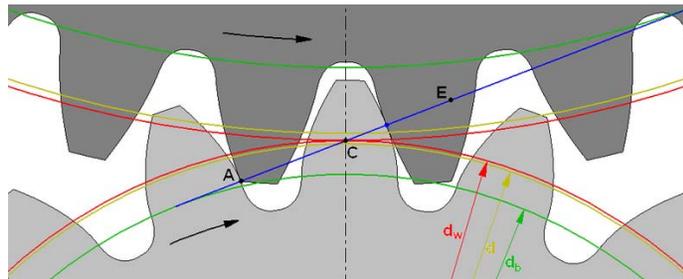
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2.007 Design and Manufacturing I  
Spring 2009

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2.007 –Design and Manufacturing I

# Gears: Terminology, Geometry, Gear Trains, Strength



Presented by Dan Frey on 17 MAR 2009

# Today's Agenda

- Distribute homework #3
- Gears
  - Applications
  - Types
  - Terminology / nomenclature
  - Congugate action
  - Involute curve
  - Analysis & design

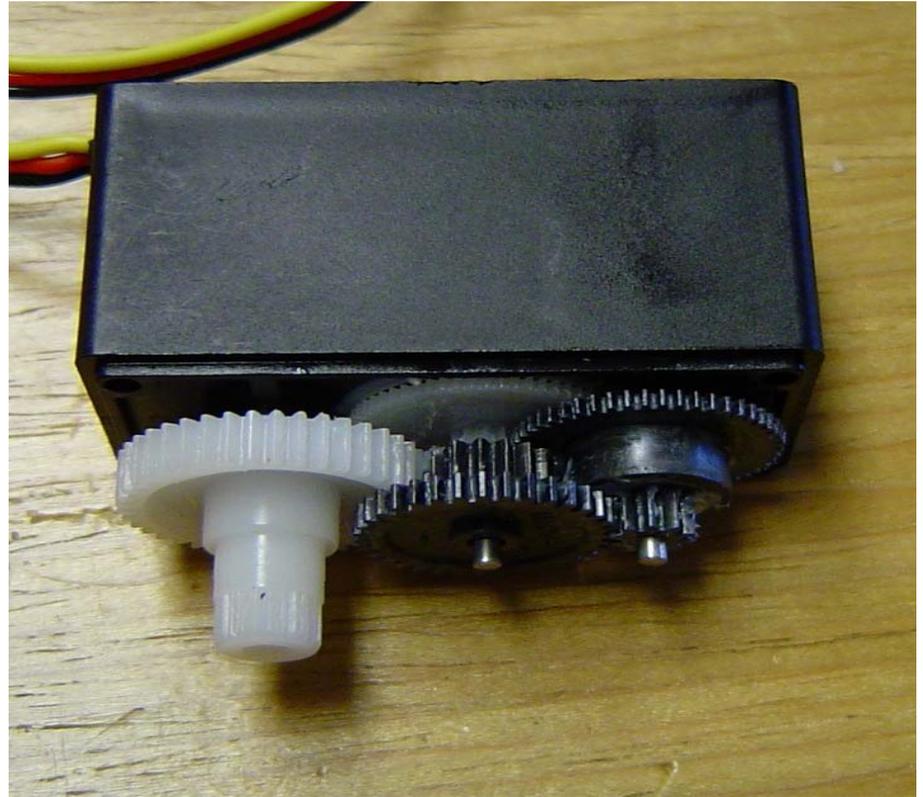
# Applications of Gears



Images removed due to copyright restrictions. Please see  
<http://mossmotors.com/Graphics/Products/Schematics/SPM-027.gif>  
<http://www.nmm.ac.uk/collections/displayRepro.cfm?reproID=A6263>

# Spur Gears

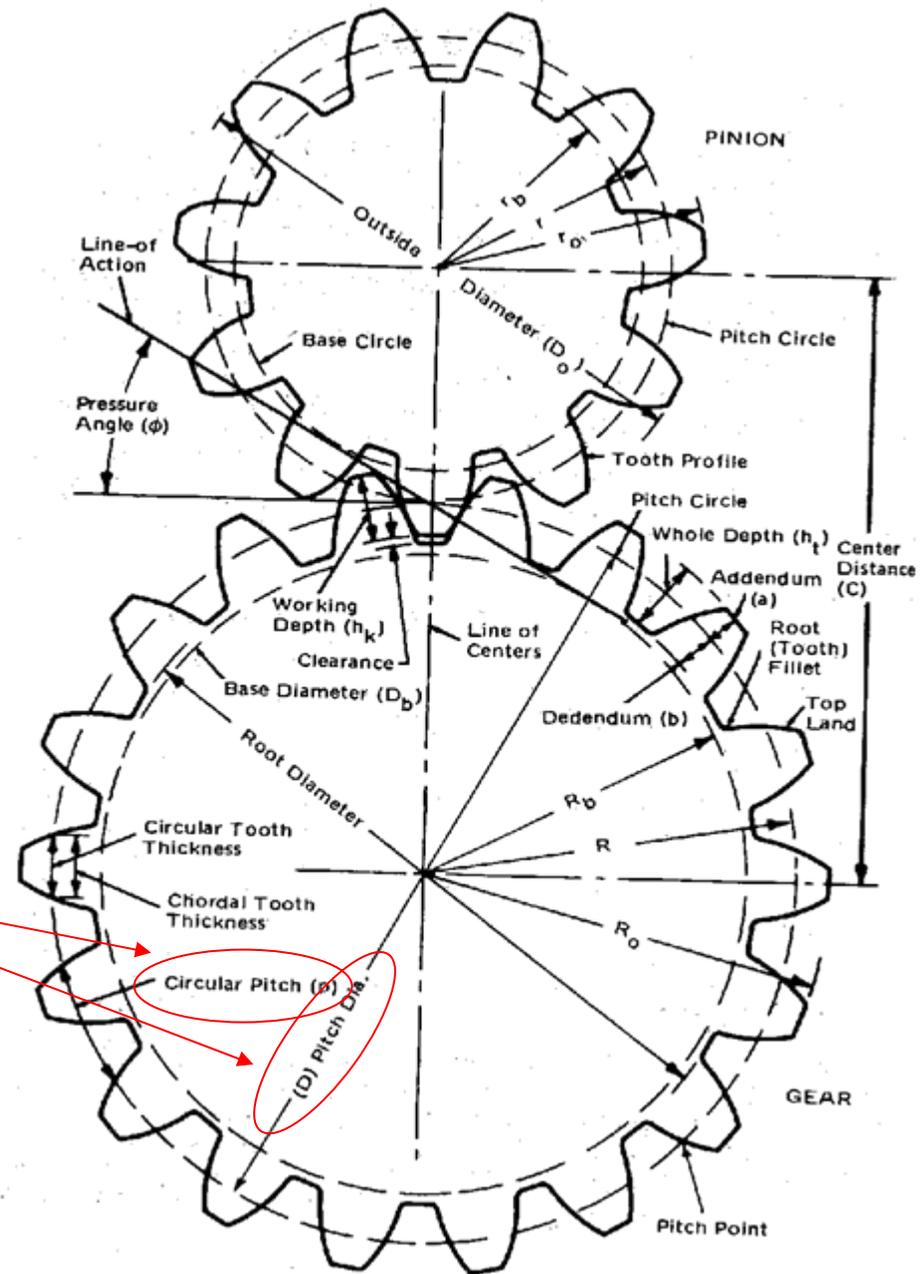
- Transmit motion between parallel shafts
- Teeth are parallel to the axis of rotation
- This is the simplest kind of gear we'll consider and most of today is dedicated to them



# Gear Terminology

Diametral pitch (teeth per inch)  
# of teeth on a gear with a  
1 inch pitch diameter

Easily confused



Source: Fig. 1.1 in "Gears." [Design and Application of Small Standardized components Data Book 757](#). Stock Drive Products, 1983. Accessed September 18, 2009. Courtesy of Stock Drive Products/Sterling Instrument.

# Other Types of Gears

Image courtesy of [perlmonger](#) on Flickr.

## Helical

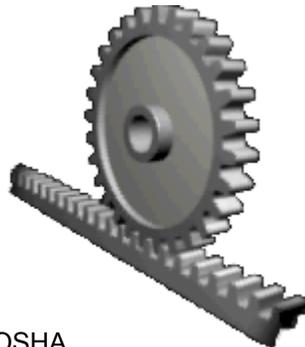


Images from Wikimedia Commons,  
<http://commons.wikimedia.org>

## Bevel



## Rack



Courtesy OSHA.

## Worm



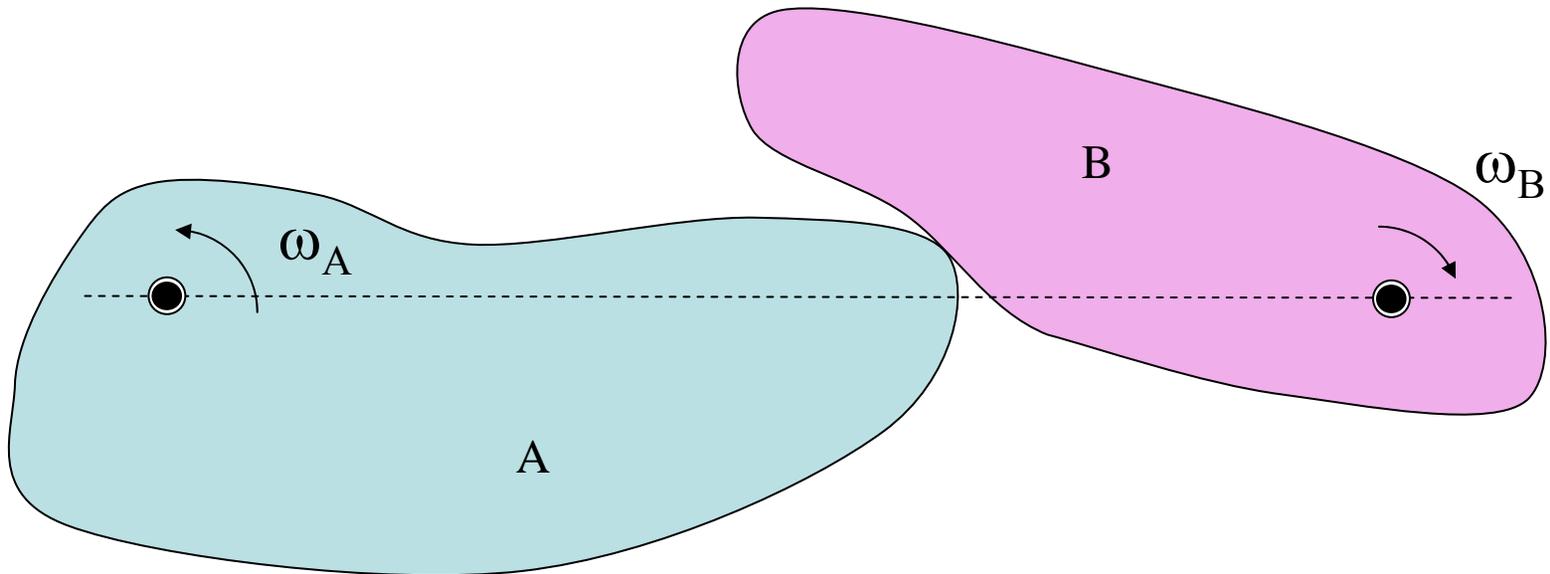
# Early Gears

Drawings of waterwheels and gears removed due to copyright restrictions.

Roman watermills at Barbegal  
300AD

Application for powering textile machinery  
18<sup>th</sup> century

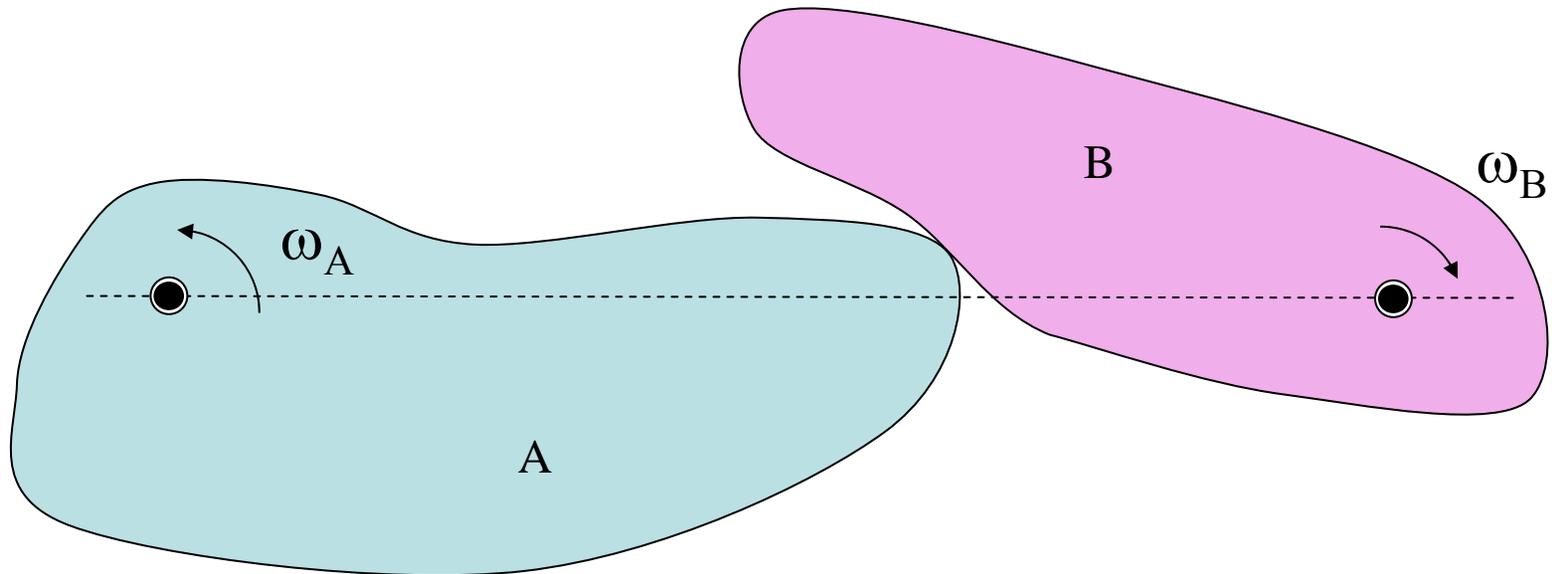
# Conjugate Action



Let's say  $\omega_A$  is a known. How can we determine  $\omega_B$ ?

Let's say  $\omega_A$  is a constant with time. Can we synthesize a shape of body B so that  $\omega_B$  is also constant with time?

# Pitch Point

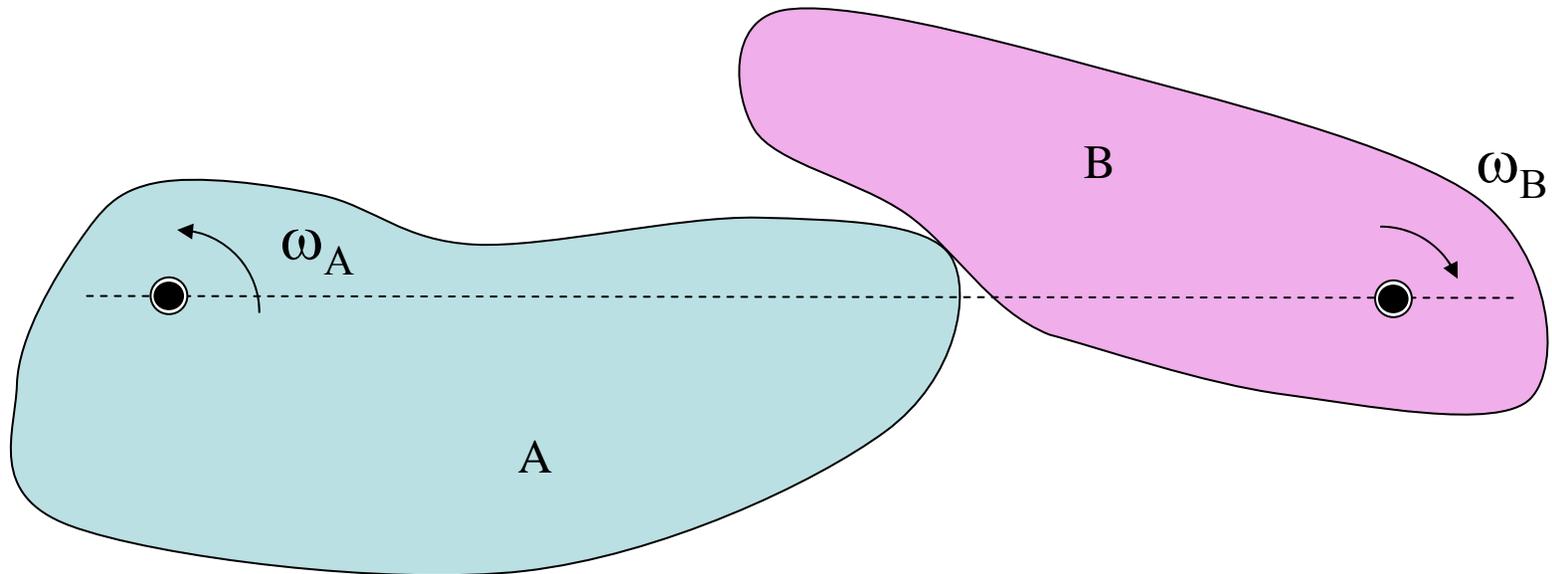


What is the pitch point?

What is the line of action?

What are the relationships among these?

# Sliding and Rolling



When one body is driving another, do the surfaces slide, roll, or both?

What is the relationship to the pitch circles?

How could you determine this?

# Rack Cutting

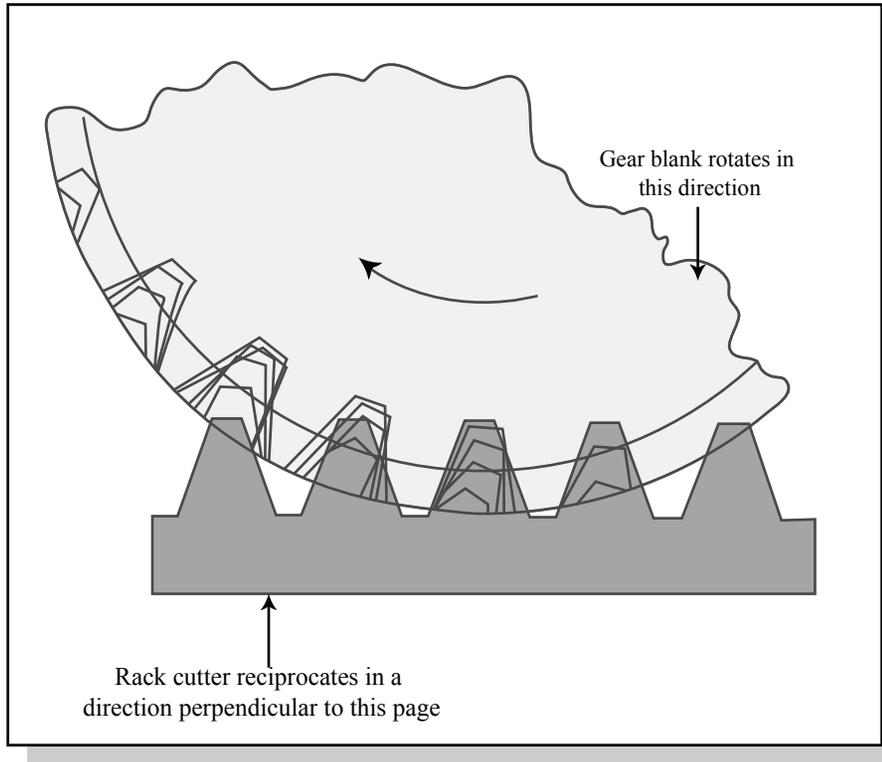
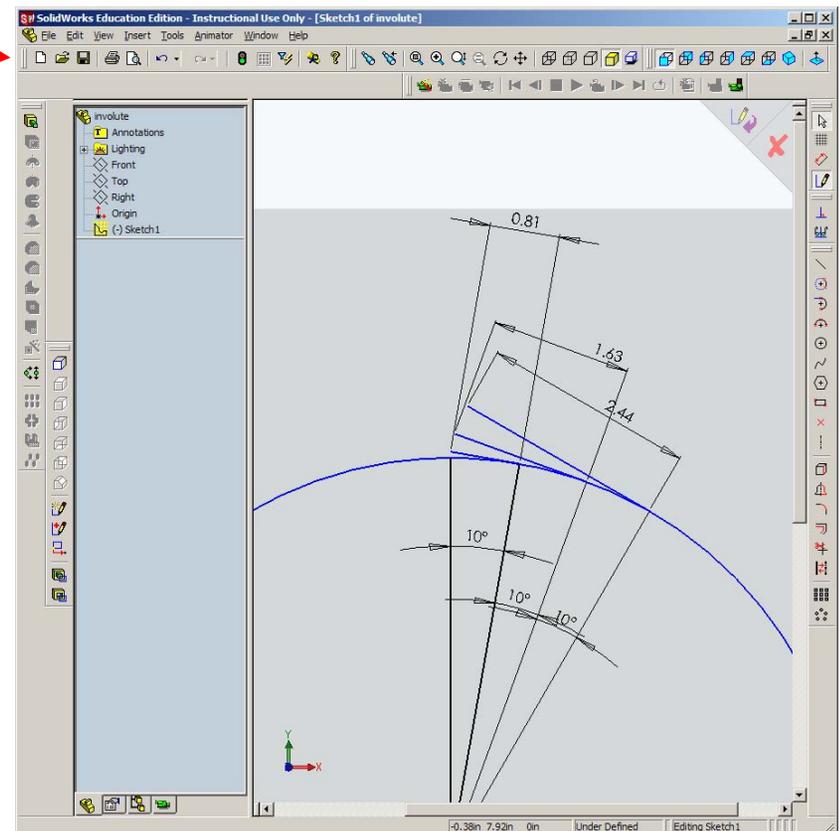


Figure by MIT OpenCourseWare.

- A way to get the relative motion you want
- Pick one shape as you wish
- Enforce the motion you want
- Cut away everything that interferes

# Involute Profile

- How it is constructed
  - Demo
- Properties
  - Conjugate action
  - Allows design of whole sets of compatible gears
  - Conjugate action not sensitive to center distance variations



# More Gear Terminology

From  
Shigley and Mischke

Image removed due to copyright restrictions. Please see [http://commons.wikimedia.org/wiki/File:Gear\\_words.png](http://commons.wikimedia.org/wiki/File:Gear_words.png)



This geometry is not an involute.

# Pressure Line

- Where the teeth contact, the surface normal defines a pressure line
- The force transmitted acts along this line
- The pressure line always includes the point of tangency between the pitch circles
- With the involute gear profile, the pressure line is constant

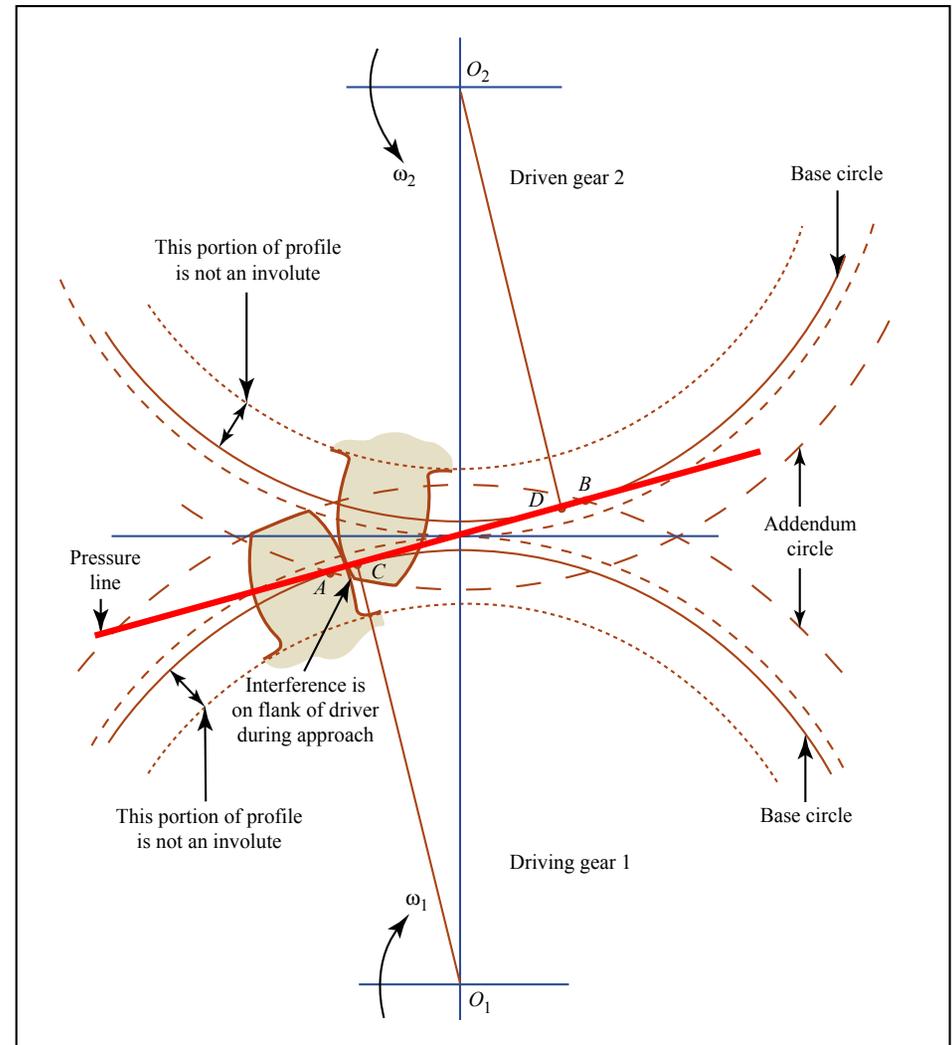
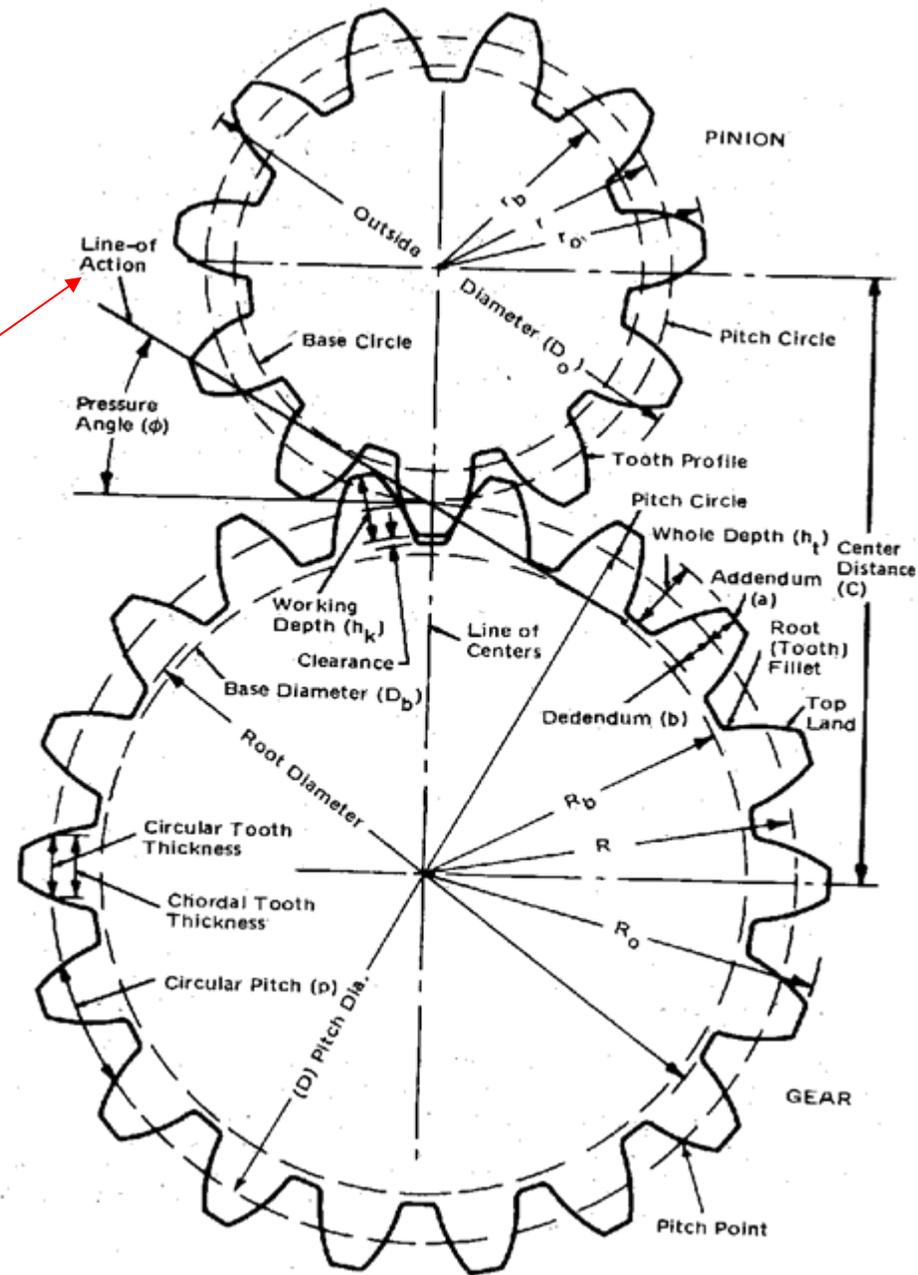


Figure by MIT OpenCourseWare.

# Gear Terminology

“Line of action” &  
“pressure line” &  
“generating line”  
are all synonymous



Source: Fig. 1.1 in “Gears.” [Design and Application of Small Standardized components Data Book 757](#). Stock Drive Products, 1983. Accessed September 18, 2009. Courtesy of Stock Drive Products/Sterling Instrument.

# Pressure Angle

- The pressure line acts at some angle to the tangent of the pitch circles
- This angle can be chosen by the designer
- It affects
  - Separation forces
  - Tooth shape

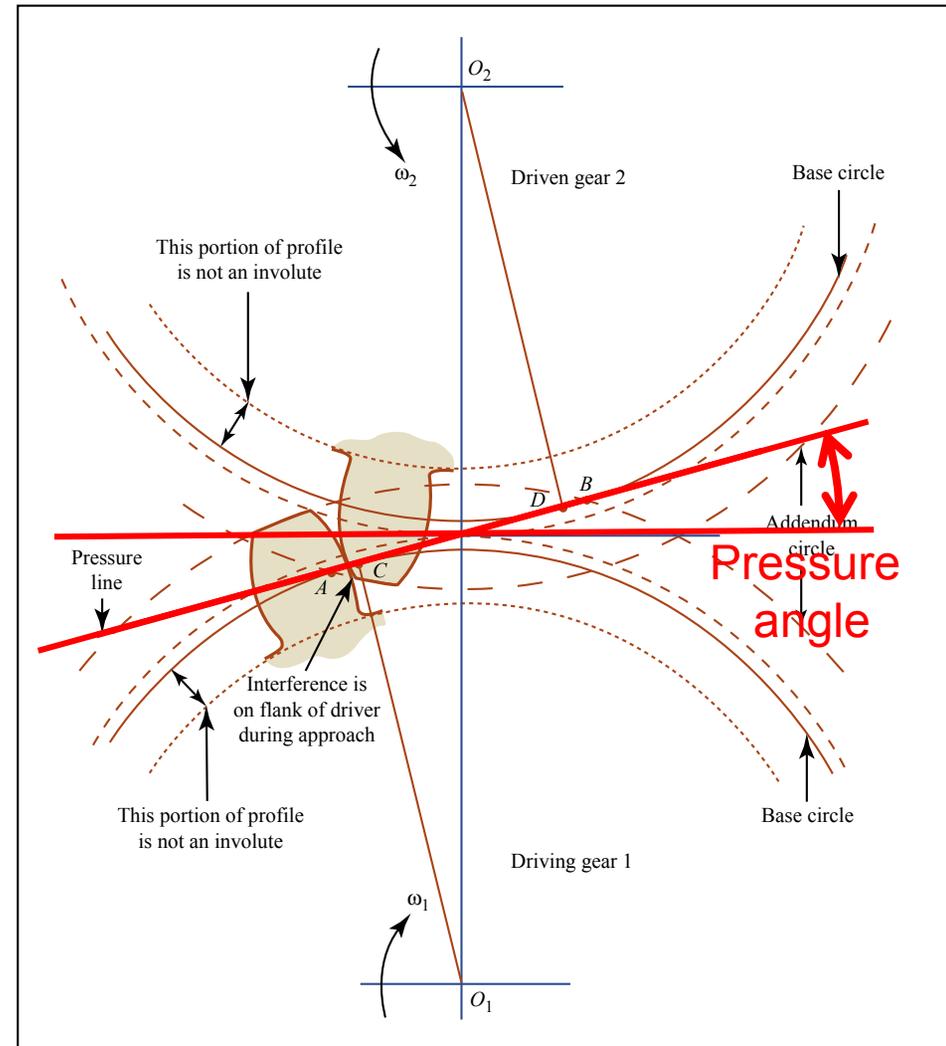
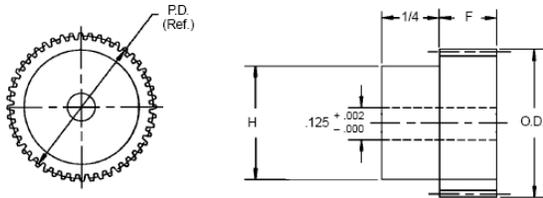


Figure by MIT OpenCourseWare.

# Concept Question

## Spur Gears

24, 32, 48, and 64 Pitch 1/8" Bore AGMA Quality 4  
Cold Rolled Steel and Brass 20° Pressure Angle



**B**  
9

A pair of gears are mated. One is driven at a set torque, the other is regulated at a set speed. The gears are the ones circled. What is the ratio of the separation forces and the total force on the bearing?

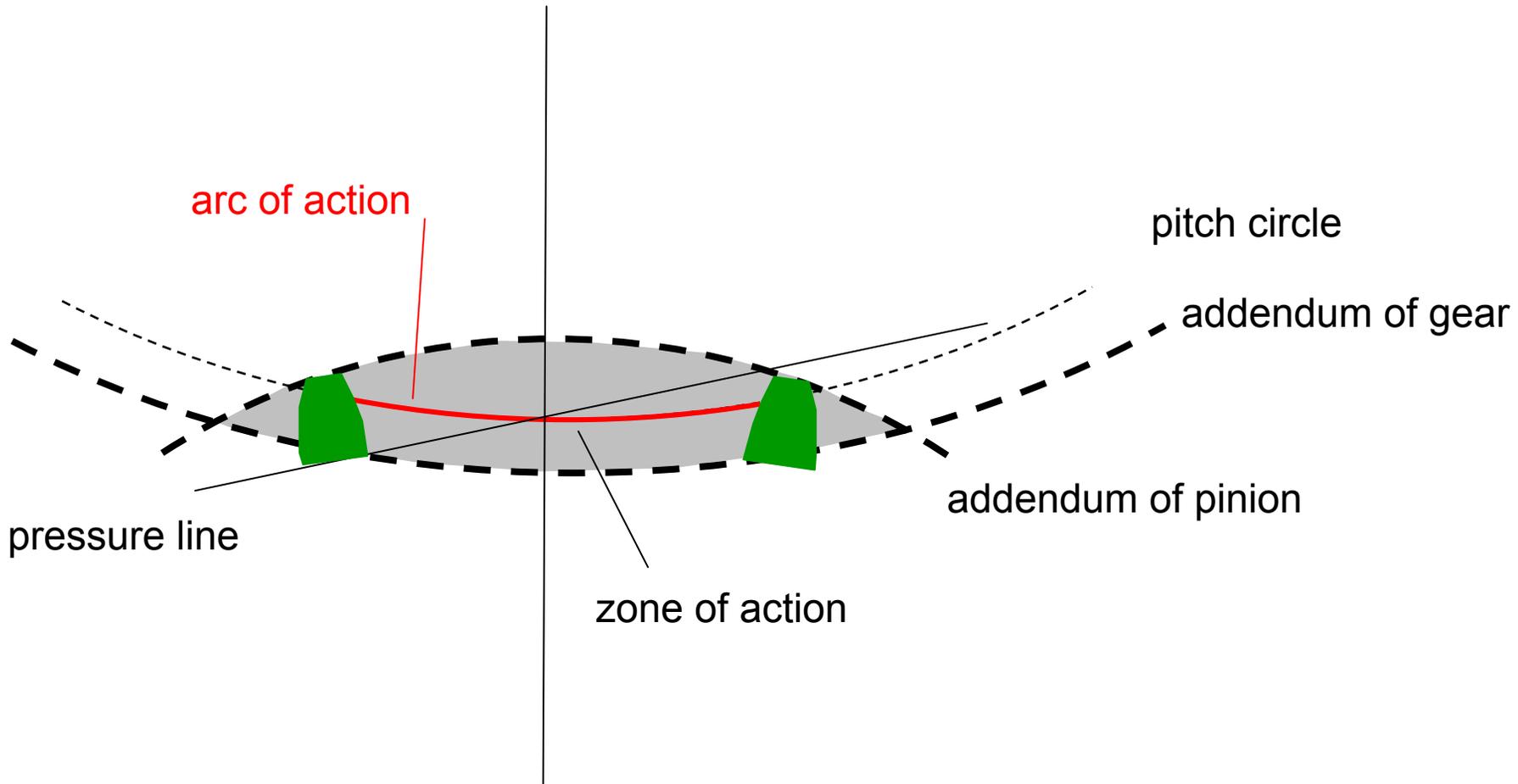
COLD ROLLED STEEL C12L14 OR C12L15 WITH SELENIUM		BRASS ALLOY 360		NO OF TEETH	PITCH DIA.	OUTSIDE DIA.	H	F
STOCK NUMBER	STOCK NUMBER							
24 PITCH (.1309)								
PX24S-8	PX24B-8	8	.333	.416	.208	1/4		
PX24S-9	PX24B-9	9	.375	.458	.250			
PX24S-10	PX24B-10	10	.417	.500	.291			
PX24S-12	PX24B-12	12	.500	.583	.375			
PX24S-16	PX24B-16	16	.666	.750	.542			
PX24S-18	PX24B-18	18	.750	.833	.625			
—	PX24B-22	22	.916	1.000	.792			
32 PITCH (.0981)								
PX32S-10	PX32B-10	10	.312	.375	.218	1/4		
PX32S-11	PX32B-11	11	.344	.406	.250			
PX32S-12	PX32B-12	12	.375	.437	.281			
PX32S-14	PX32B-14	14	.438	.500	.343			
PX32S-15	PX32B-15	15	.469	.531	.375			
PX32S-16	PX32B-16	16	.500	.562	.406			
PX32S-18	PX32B-18	18	.563	.625	.468			
PX32S-20	PX32B-20	20	.625	.688	.532			
—	PX32B-24	24	.750	.813	.656			
48 PITCH (.0654)								
PX48S-14	PX48B-14	14	.292	.333	.229	1/8		
PX48S-15	PX48B-15	15	.312	.353	.250			
PX48S-16	PX48B-16	16	.333	.375	.271			
PX48S-18	PX48B-18	18	.375	.417	.312			
PX48S-24	PX48B-24	24	.500	.542	.437			
PX48S-32	PX48B-32	32	.666	.708	.604			
—	PX48B-36	36	.750	.792	.687			
—	PX48B-40	40	.833	.875	.770			
64 PITCH (.0490)								
PX64S-15	PX64B-15	15	.234	.265	.187	1/8		
PX64S-16	PX64B-16	16	.250	.281	.203			
PX64S-18	PX64B-18	18	.281	.312	.234			
—	PX64B-24	24	.375	.406	.328			
—	PX64B-40	40	.625	.656	.578			
—	PX64B-48	48	.750	.781	.703			

Berg Manufacturing "The Mark of Quality"

1-800-232-BERG

1.  $\ll 0.3$
2. About 0.3
3. About 0.5
4.  $\gg 0.5$

# Contact Ratio



contact ratio = length of arc of action / pitch = average number of teeth engaged

# Interference

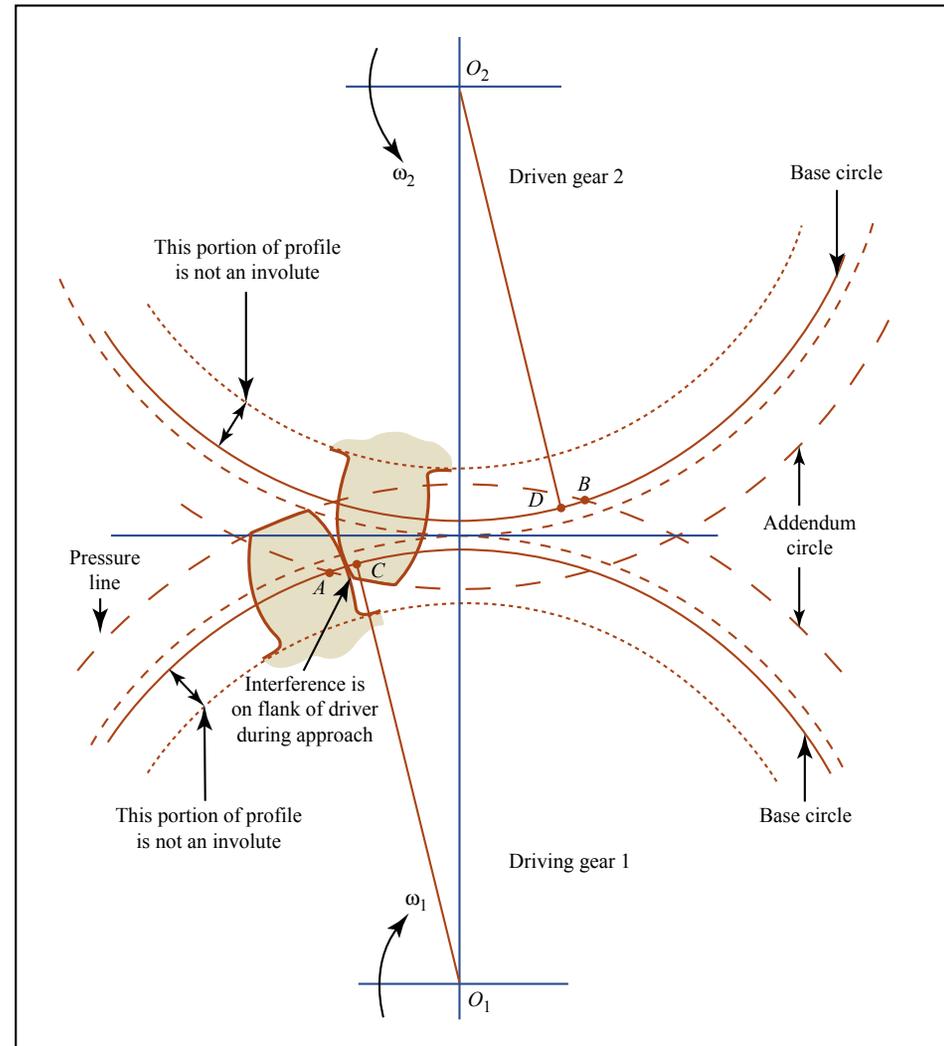
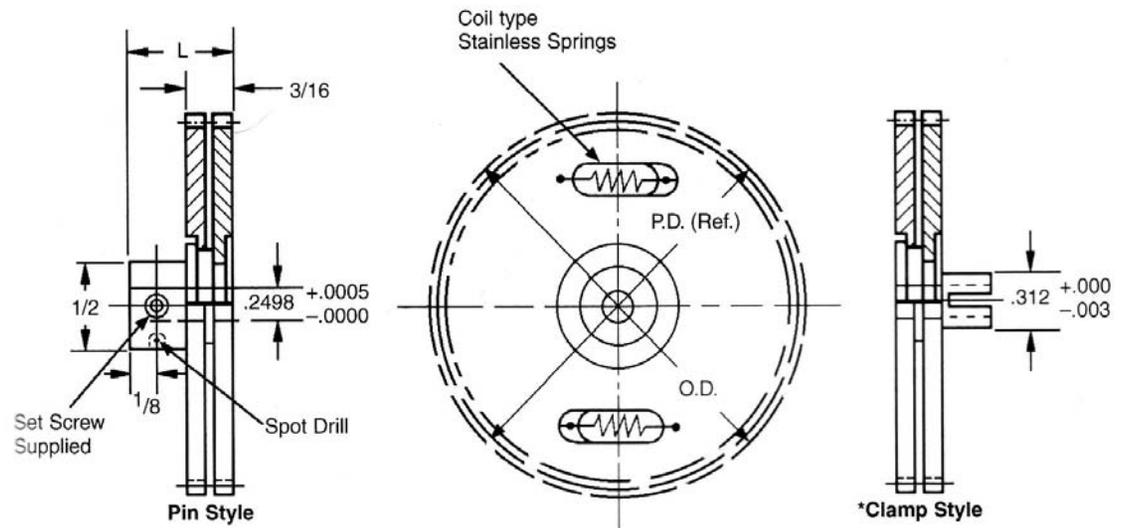


Figure by MIT OpenCourseWare.

# Backlash



**Hub Material:**  
303 Stainless Steel

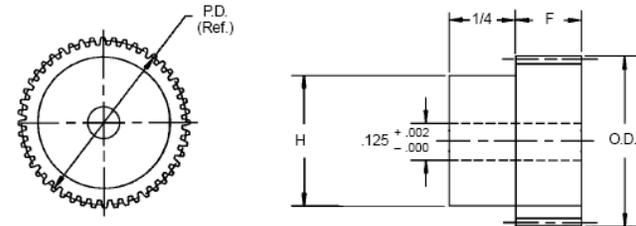
Courtesy of W. M. Berg, Inc. Used with permission.

# Gear Selection

- Pitch
- Face width
- Material
- Pressure angle
- # of teeth
- Hub style, bore, etc.

## Spur Gears

24, 32, 48, and 64 Pitch 1/8" Bore AGMA Quality 4  
Cold Rolled Steel and Brass 20° Pressure Angle

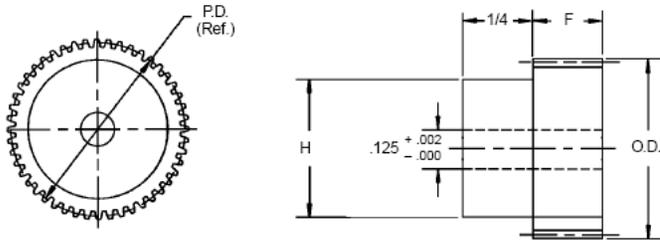


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COLD ROLLED STEEL C12L14 OR C12L15 WITH SELENIUM		BRASS ALLOY 360		NO OF TEETH	PITCH DIA.	OUTSIDE DIA.	H	F
STOCK NUMBER	STOCK NUMBER							
24 PITCH (.1309)								
PX24S-8	PX24B-8	8	.333	8	.416	.208	1/4	
PX24S-9	PX24B-9	9	.375	9	.458	.250		
PX24S-10	PX24B-10	10	.417	10	.500	.291		
PX24S-12	PX24B-12	12	.500	12	.583	.375		
PX24S-16	PX24B-16	16	.666	16	.750	.542		
PX24S-18	PX24B-18	18	.750	18	.833	.625		
—	PX24B-22	22	.916	22	1.000	.792		
32 PITCH (.0981)								
PX32S-10	PX32B-10	10	.312	10	.375	.218	1/4	
PX32S-11	PX32B-11	11	.344	11	.406	.250		
PX32S-12	PX32B-12	12	.375	12	.437	.281		
PX32S-14	PX32B-14	14	.438	14	.500	.343		
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48 PITCH (.0654)								
PX48S-14	PX48B-14	14	.292	14	.333	.229	1/8	
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PX64S-16	PX64B-16	16	.250	16	.281	.203		
PX64S-18	PX64B-18	18	.281	18	.312	.234		
—	PX64B-24	24	.375	24	.406	.328		
—	PX64B-40	40	.625	40	.656	.578		
—	PX64B-48	48	.750	48	.781	.703		

# Spur Gears

24, 32, 48, and 64 Pitch 1/8" Bore AGMA Quality 4  
Cold Rolled Steel and Brass 20° Pressure Angle



**B**  
9

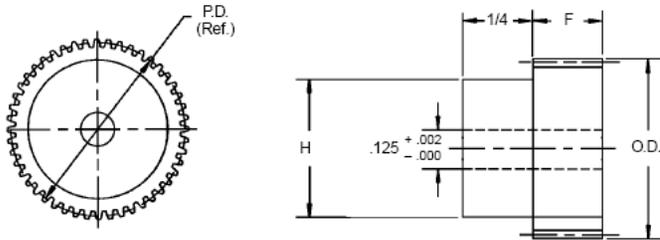
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—	PX64B-48	48	.750	.781	.703		

You call up the number 1-800-232-BERG and ask that, for a special application, you want a 48 pitch spur gear, but with a pitch dia of 0.32 inches. They will probably say:

1. OK, no problem
2. OK, but it will cost a lot
3. No, this is not technically possible

# Spur Gears

24, 32, 48, and 64 Pitch 1/8" Bore AGMA Quality 4  
Cold Rolled Steel and Brass 20° Pressure Angle



**B**  
9

COLD ROLLED STEEL C12L14 OR C12L15 WITH SELENIUM		BRASS ALLOY 360					
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You call up the number 1-800-232-BERG and ask that, for a special application, you want a 48 pitch spur gear, but with a pitch dia of half the smallest one in the catalog. They will probably say:

1. OK, no problem
2. OK, but it will cost a lot
3. OK, but it will be weak
4. No, this is not technically possible

# Ways Gears Fail

Exceed endurance limit in bending

Exceed static yield stress in bending



← “stripping”

Image courtesy of [deltaMike](#) at Flickr.

Exceed endurance limit in contact stress

Images removed due to copyright restrictions. Please see

[http://materials.open.ac.uk/mem/mem\\_mf6.htm](http://materials.open.ac.uk/mem/mem_mf6.htm)

[http://www.hghouston.com/x/39\\_gearpit.html](http://www.hghouston.com/x/39_gearpit.html)

← “pitting”

# Stress in Gears

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# A Beam in Bending

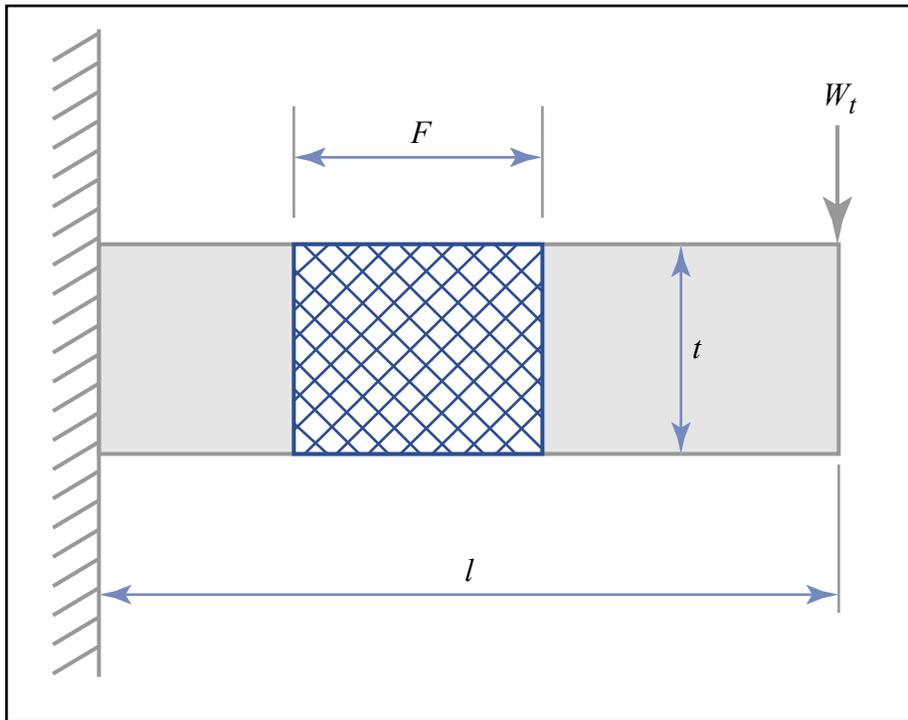


Figure by MIT OpenCourseWare.

$$\sigma = \frac{M}{I / c} = \frac{6W_t l}{Ft^2}$$

# Concept Question

- In selecting a gear of one inch pitch diameter, we are choosing between 48 and 24 pitch gear teeth. The effect on torque that can be transmitted before bending failure of the teeth is

1. Around a factor of 10
2. Around a factor of 4
3. Around a factor of 2
4. Less than a factor of 2

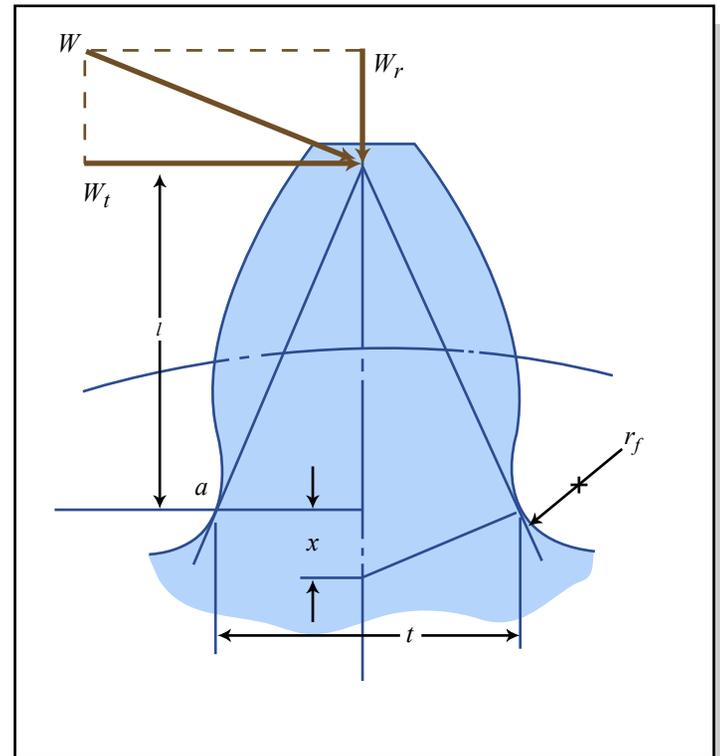


Figure by MIT OpenCourseWare.

# Strength of Gears

- Any good catalog will have a formula and tables
- What factors must enter the equation?
  - 
  - 
  -
- Where do the teeth wear the most?

**K**  
4

## Gear Reference Guide

### GEAR TOOTH STRENGTH

Many factors must be considered when designing a gear train. The information listed on this page should be used as a general guideline for your application. If more critical strength calculation is required W.M. Berg suggests that you consult our engineering department or any one of the many gear handbooks that are readily available.

When a gear train is transmitting motion, it is safe to assume that all of the load is being carried by one tooth. This is because as the load approaches the end of the tooth, where the bending force would be the greatest, a second tooth comes into mesh to share the load. Simple results can be obtained from the Lewis bending strength equation.

$$W_t = \frac{SFY}{D.P.}$$

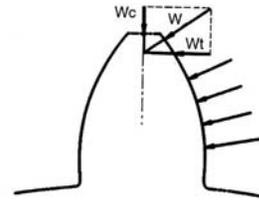
$W_t$  = Maximum transmitted load (lbs or N)

S = Maximum bending tooth stress (taken as 1/3 of the tensile strength) See Table C on Page 5

F = Face width of gear (in. or mm)

D.P. = Diametral Pitch = 1/module (for equation only)

Y = Lewis Factor (See Table)



NOTE: The maximum bending tooth stress (S) is valid for well lubricated, low shock applications. For high shock, poorly lubricated applications, the safe stress could be as low as .025S. If your design calls for an unfriendly environment for gears, you might want to lower S to assure a reasonable amount of gear life.

LEWIS FACTOR - Y	NO. OF TEETH	14 1/2° INVOLUTE	20° INVOLUTE
	10	0.176	0.201
11	0.192	0.226	
12	0.210	0.245	
13	0.223	0.264	
14	0.236	0.276	
15	0.245	0.289	
16	0.255	0.295	
17	0.264	0.302	
18	0.270	0.308	
19	0.277	0.314	
20	0.283	0.320	
22	0.292	0.330	
24	0.302	0.337	
26	0.308	0.344	
28	0.314	0.352	
30	0.318	0.358	
32	0.322	0.364	
34	0.325	0.370	
36	0.329	0.377	
38	0.332	0.383	
40	0.336	0.389	
45	0.340	0.399	
50	0.346	0.408	
55	0.352	0.415	
60	0.355	0.421	
65	0.358	0.425	
70	0.360	0.429	
75	0.361	0.433	
80	0.363	0.436	
90	0.366	0.442	
100	0.368	0.446	
150	0.375	0.458	
200	0.378	0.463	
300	0.382	0.471	
RACK	0.390	0.484	

# The Lewis Formula

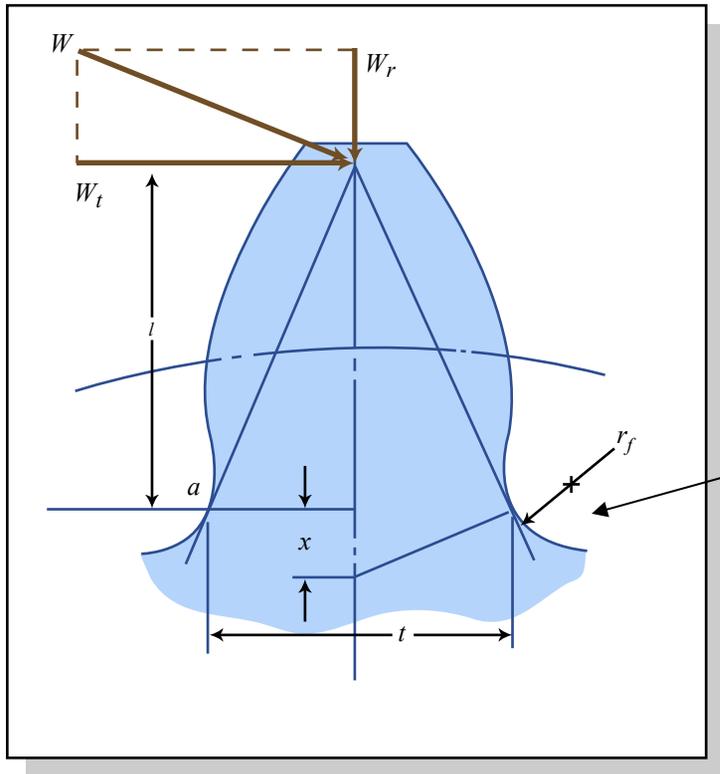


Figure by MIT OpenCourseWare.

$$\sigma = \frac{W_t P}{F y}$$

Diametral pitch  
(teeth/inch)

“Lewis form factor”

Face width

Low form factor → High stress

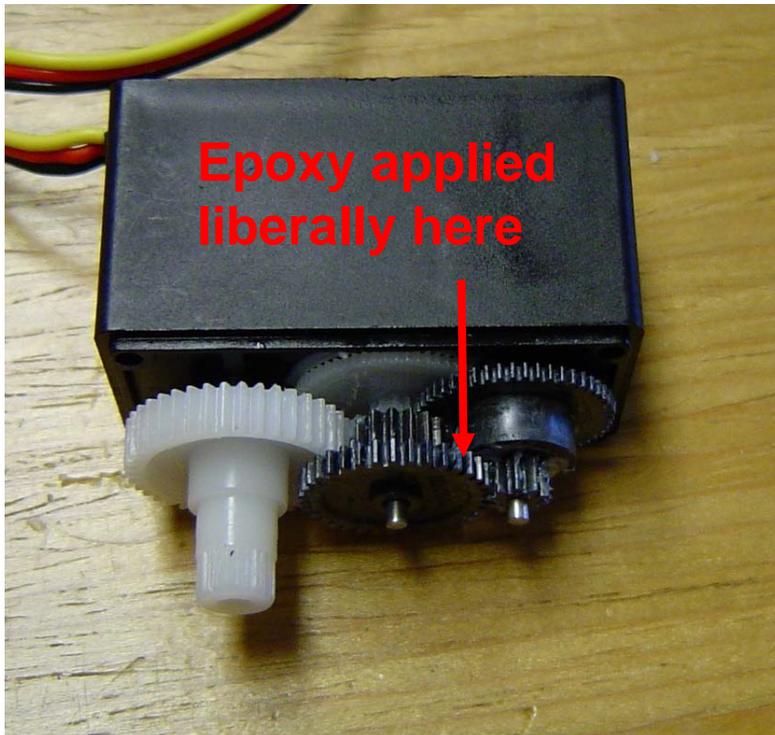
Point of max stress due to bending

# Or Use a Canned Tool

Please see “Spur Gear Tooth Strength” at <http://www.wmberg.com/tools/>

# Discussion Questions

- I glued the third stage teeth of this servo together
- Now I will apply a load to the output shaft (up to 10lbs)
- What's going to happen?



# Concept Question

- For a gear to provide the highest strength at a fixed diameter, we prefer
  1. High pressure angle
  2. Low pressure angle
  3. It doesn't matter much

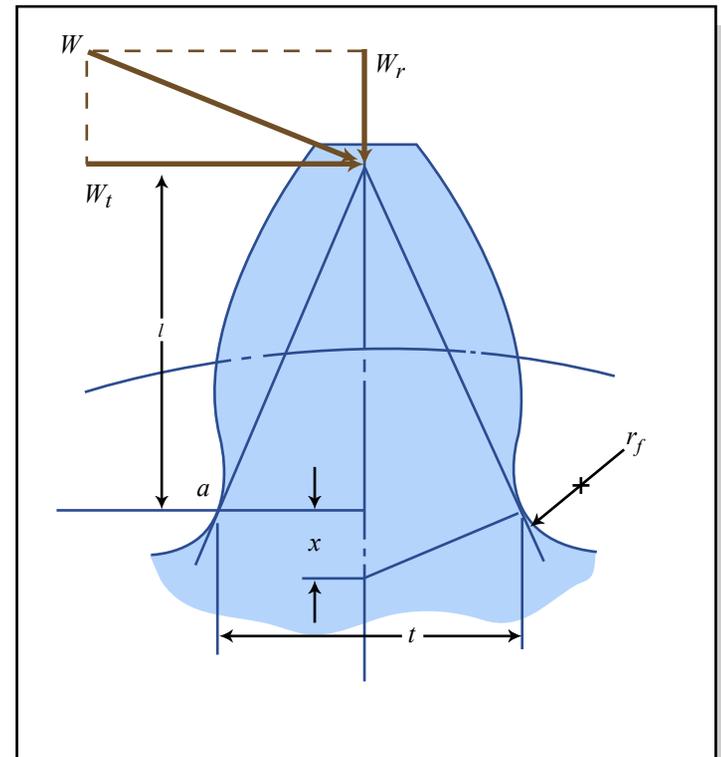
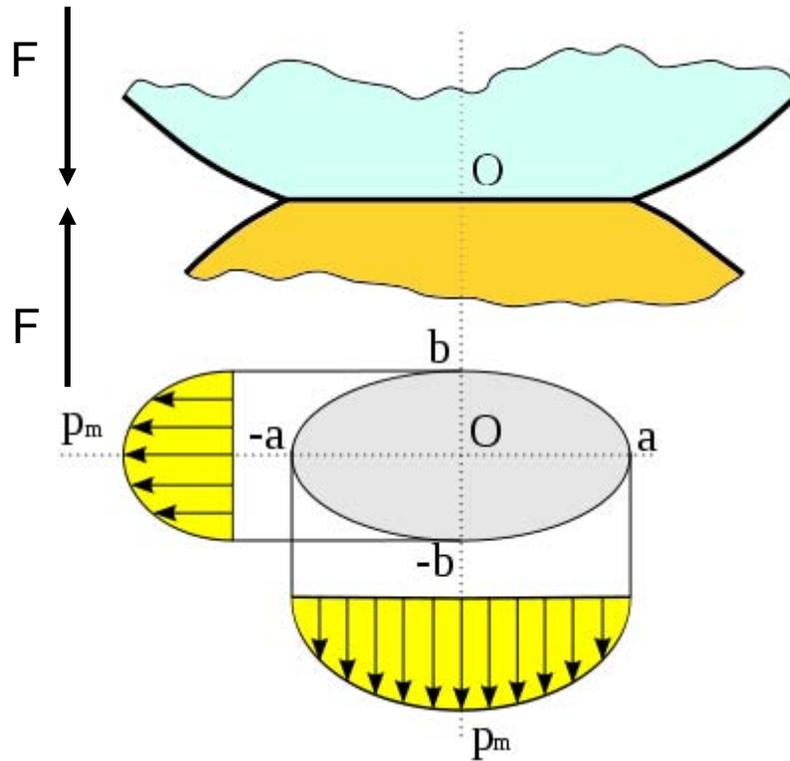


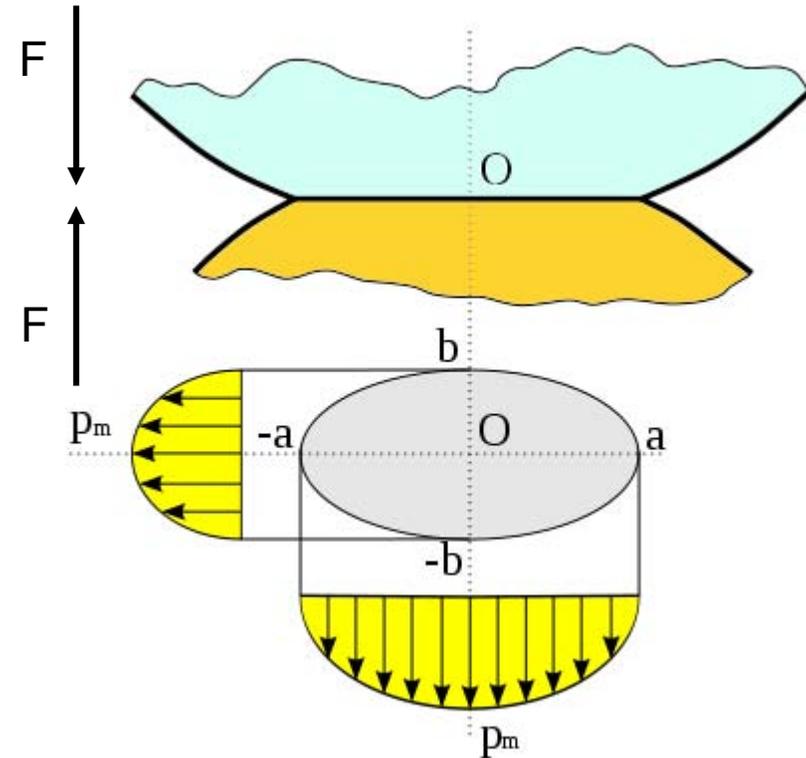
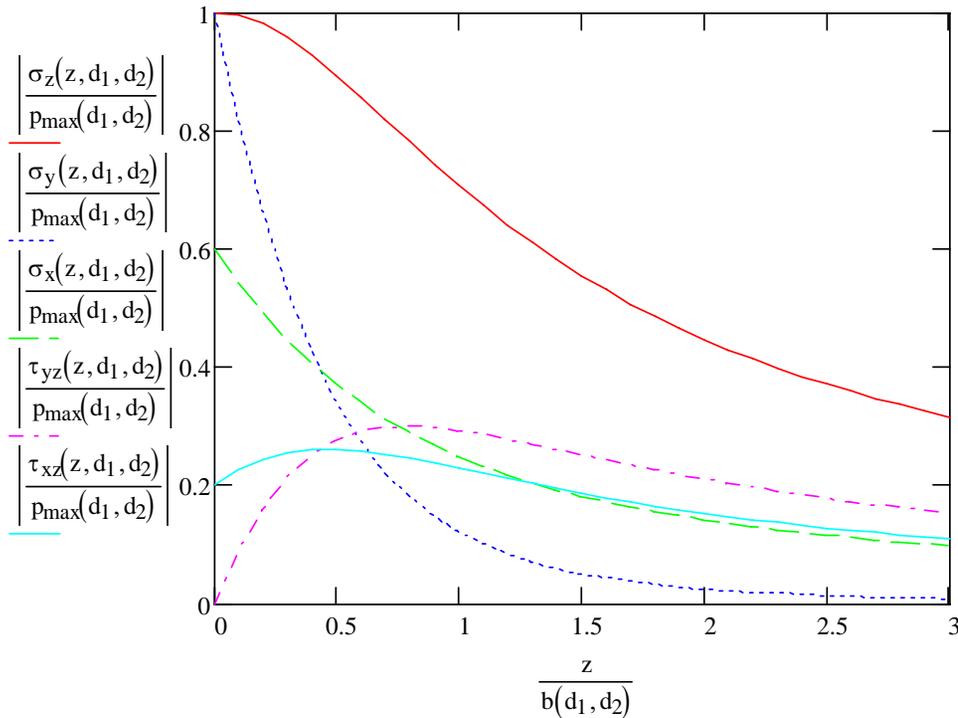
Figure by MIT OpenCourseWare.

# Contact Stress (Hertzian Stress)



# Contact Stress

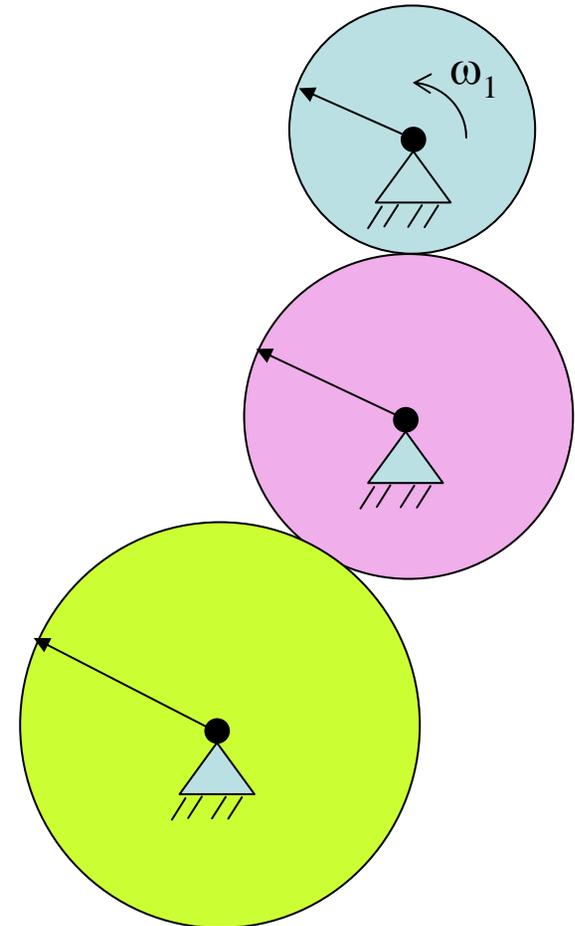
## Quantitative Characterization



$$b(d_1, d_2) := \sqrt{\frac{2 \cdot F}{\pi \cdot 1} \cdot \frac{\frac{(1 - \nu_1^2)}{E_1} + \frac{1 - \nu_2^2}{E_2}}{\frac{1}{d_1} + \frac{1}{d_2}}} \quad p_{\max}(d_1, d_2) := \frac{2 \cdot F}{\pi \cdot b(d_1, d_2) \cdot 1}$$

# Simple Gear Trains

- A “simple” gear train has only one gear on each shaft
- How does this arrangement behave?



# Compound Gear Trains

- A “compound” gear train has at least one shaft with multiple gears
- How does this arrangement behave?



Image from Wikimedia Commons,  
<http://commons.wikimedia.org>



# Discussion Questions

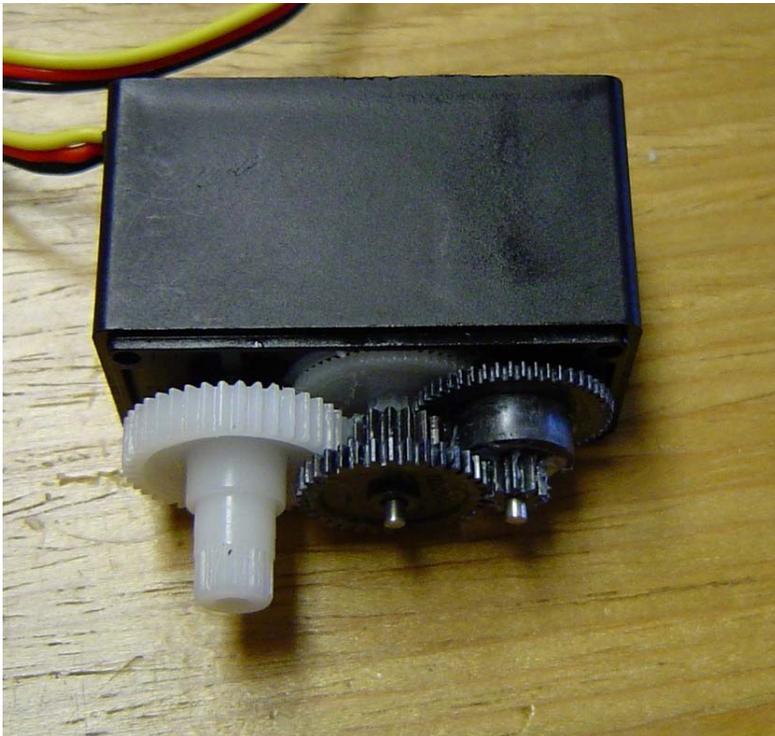


- Are there any disadvantages to a helical gear as compared to a spur gear?
- How can the disadvantages be remedied?
- Is a helical gear set stronger than a spur gear of the same diameter, pitch, face width, & material?

Image from Wikimedia Commons,  
<http://commons.wikimedia.org>

# Concept Question

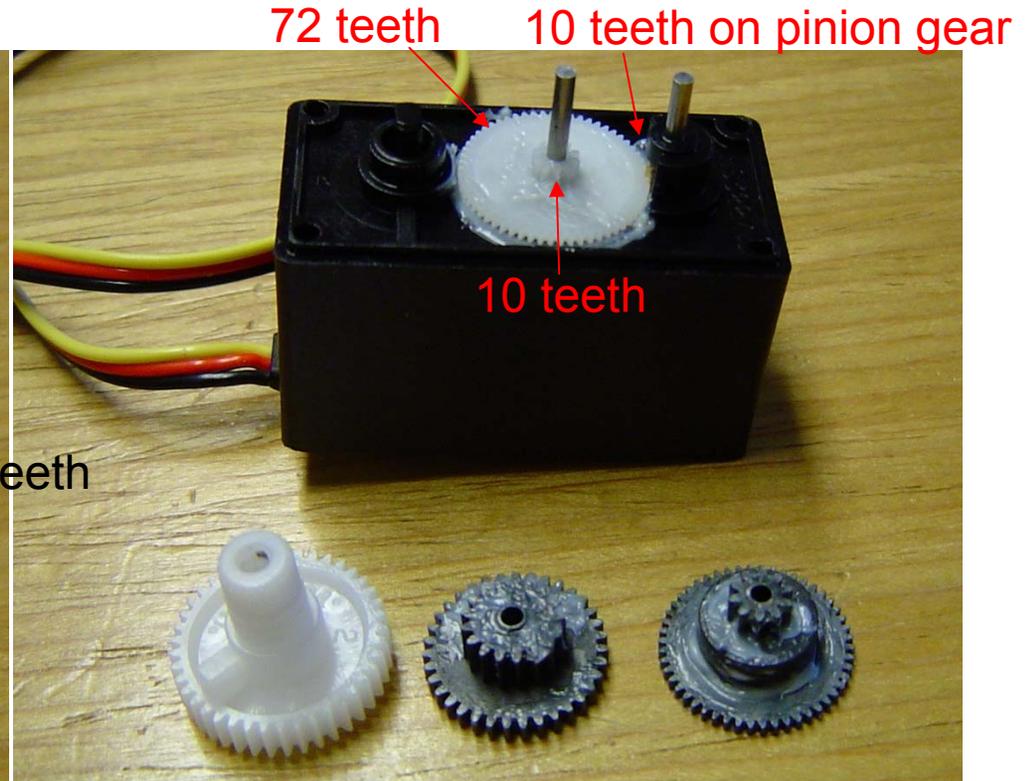
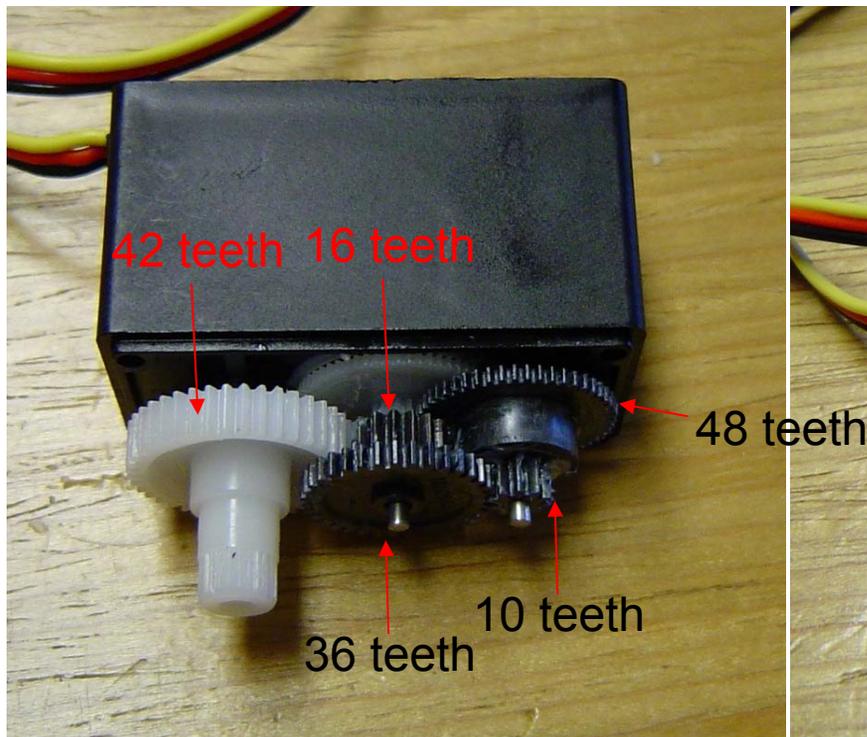
A compound gear train is formed of eight gears. As we proceed from the pinion on the electric motor to the gear on the output shaft, how do the pitch and face width vary?



1. Pitch rises,  
face width rises
2. Pitch rises,  
face width falls
3. Pitch falls,  
face width rises
4. Pitch falls,  
face width falls

# Discussion Questions

- How many stages in this device?
- How do you suppose this number is chosen?
- Are the reduction ratios typically all nearly the same in all successive stages?



# Differentials

- Allows shafts to move at different speeds
- Applies same torque to both
- Slippage problem

Image removed due to copyright restrictions. Please see <http://mossmotors.com/Graphics/Products/Schematics/SPM-027.gif>

<http://static.howstuffworks.com/flash/differential.swf>

# Next Steps

- Begin Homework #3
- Next lecture Thursday 19 March
  - CAD case study
- Spring break
- Lecture Tuesday 31 March
  - More gears, and also springs
- HW#3 due 7 April
- Quiz #2 on 16 April
- Impounding week 29 April to 1 May

# Planetary Gear Trains

- One or more of the gear axes are allowed to rotate
- aka “epicyclic”
- Used in
  - Power tools
  - Automatic transmissions
  - Gear boxes



Courtesy NASA.

Please also see

[http://commons.wikimedia.org/wiki/File:Epicyclic\\_carrier\\_locked.png](http://commons.wikimedia.org/wiki/File:Epicyclic_carrier_locked.png)

[http://i.i.com.com/cnwk.1d/i/ss/2007/0828\\_Driving\\_it/DSG\\_440.jpg](http://i.i.com.com/cnwk.1d/i/ss/2007/0828_Driving_it/DSG_440.jpg)

# Analysis of Planetary Gear Trains

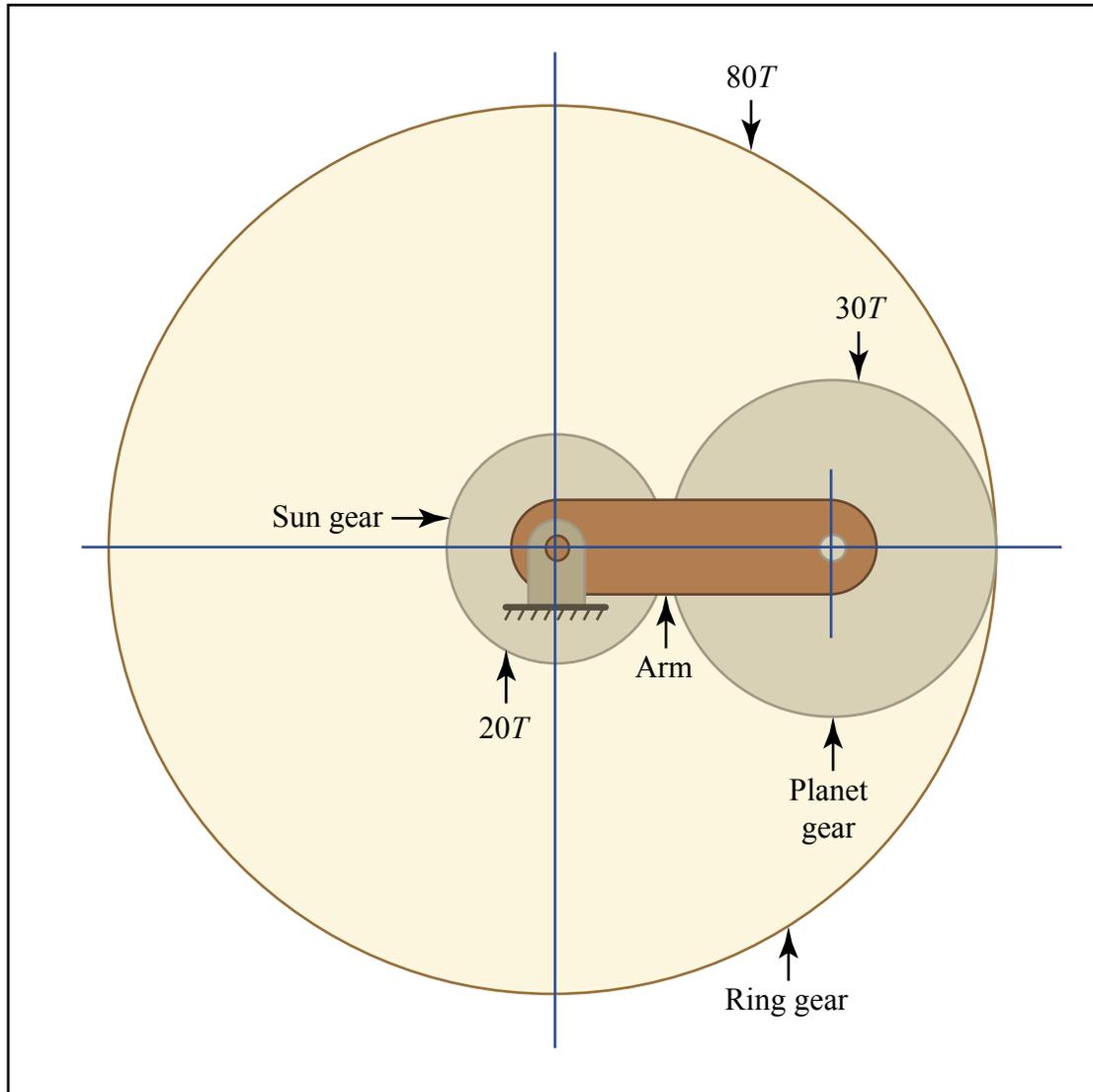


Figure by MIT OpenCourseWare.

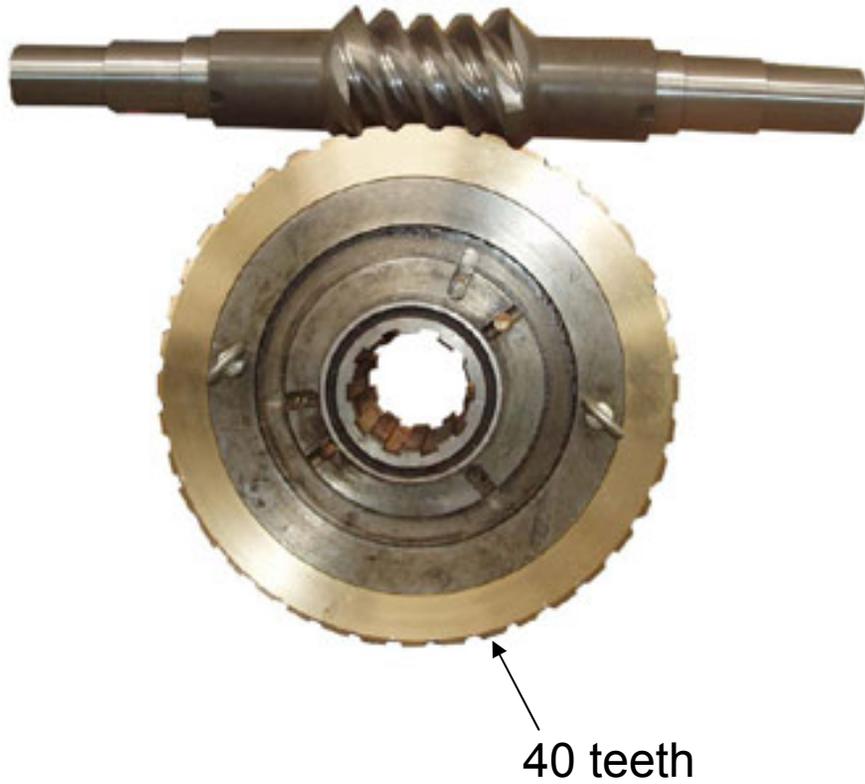
# Name That Gear



What type of worm gear set is this?

- 1) Single-enveloping, single threaded
- 2) Single-enveloping, multi-threaded
- 3) Double enveloping single threaded worm gear
- 4) Double enveloping multi- threaded

# Follow up



What is the reduction ratio of this gear set?

- 1) 10:1
- 2) 20:1
- 3) 40:1
- 4) 80:1