

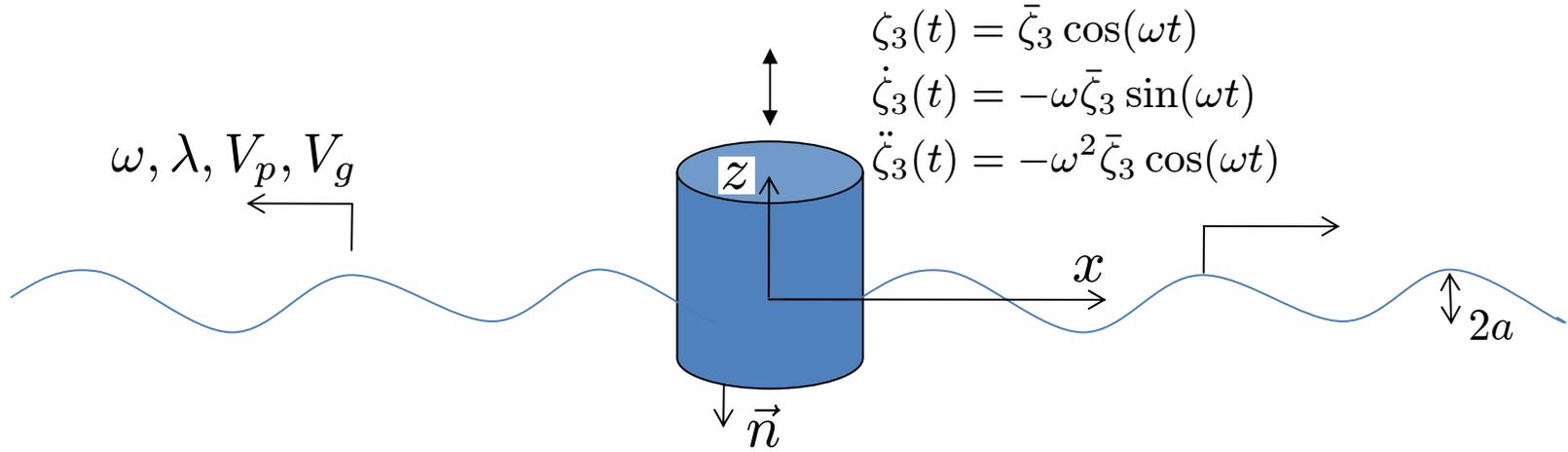
# **2.019 Design of Ocean Systems**

## **Lecture 6**

### **Seakeeping (II)**

**February 21, 2011**

# Wave Radiation Problem



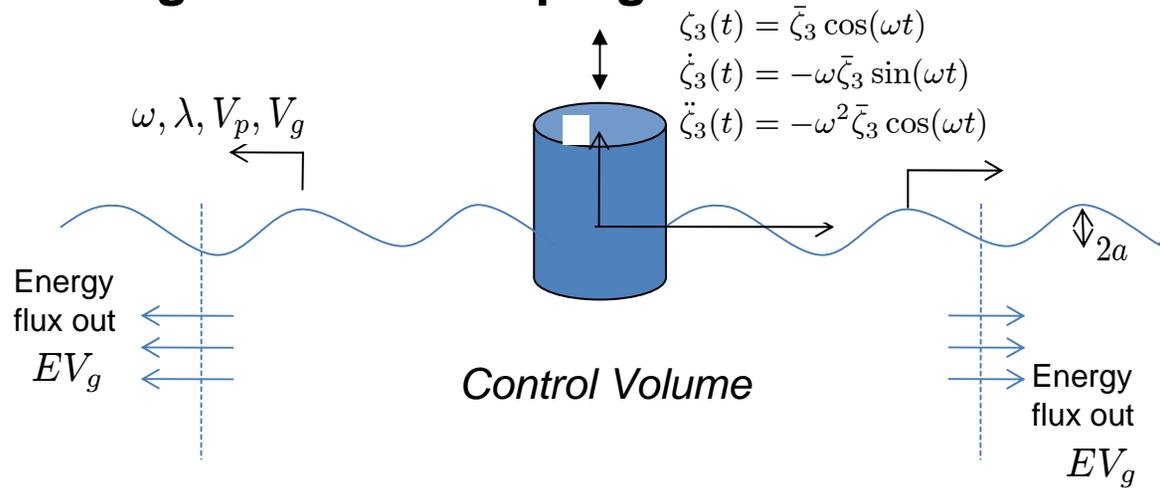
Total:  $P(t) = -\rho \frac{\partial \phi}{\partial t} - \rho g z$       Hydrodynamic:  $P_d(t) = -\rho \frac{\partial \phi}{\partial t} = \bar{P}_d \cos(\omega t - \psi)$

Hydrodynamic Force:

$$\begin{aligned}
 F_3(t) &= - \int \int_{S_B} P_d n_z dS = \bar{F}_3 \cos(\omega t - \psi) \\
 &= \bar{F}_3 \cos \psi \cos(\omega t) + \bar{F}_3 \sin \psi \sin(\omega t) \\
 &= - \frac{\bar{F}_3 \cos \psi}{\bar{\zeta}_3 \omega^2} \ddot{\zeta}_3(t) - \frac{\bar{F}_3 \sin \psi}{\bar{\zeta}_3 \omega} \dot{\zeta}_3(t) \\
 &= -A_{33} \ddot{\zeta}_3(t) - B_{33} \dot{\zeta}_3(t)
 \end{aligned}$$

$A_{33}$ : Added mass;  $B_{33}$ : Wave damping

# Physical Meaning of Wave Damping



Averaged power into the fluid by the body:

$$\begin{aligned}
 \bar{E}_{in} &= \frac{1}{T} \int_0^T \{-F_3(t)\} \dot{\zeta}_3(t) dt \\
 &= \frac{1}{T} \int_0^T \left\{ A_{33} \ddot{\zeta}_3(t) \dot{\zeta}_3(t) + B_{33} \dot{\zeta}_3(t) \dot{\zeta}_3(t) \right\} dt = B_{33} (\bar{\zeta}_3 \omega)^2 / 2
 \end{aligned}$$

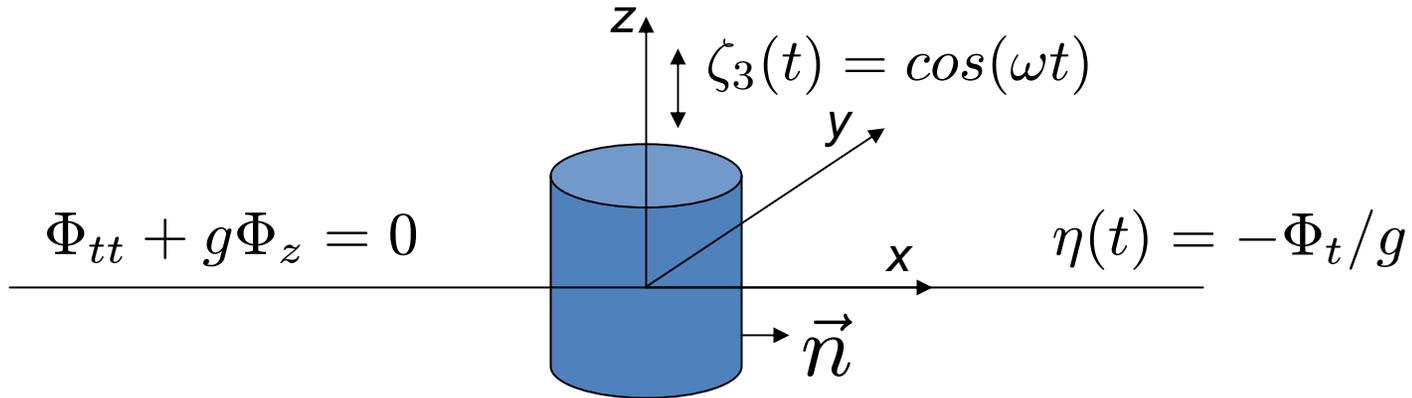
Averaged energy flux out of the control volume:  $\bar{E}_{flux} = 2V_g E \sim 2V_g a^2$

Conservation of energy:  $\frac{d\bar{E}}{dt} \equiv \bar{E}_{in} - \bar{E}_{flux} = 0$

$$\longrightarrow B_{33} \sim (a/\bar{\zeta}_3)^2 > 0$$

- $B_{33} = 0$  if  $a=0$  corresponding to  $\omega = \infty, 0$

# Mathematical Formulation of Heave Radiation Problem



Radiation condition:  
Generated waves  
must propagate away  
from the body

$$\nabla^2 \Phi(x, y, z, t) = 0$$

Deep water condition:  
 $\nabla \Phi \rightarrow 0$  as  $z \rightarrow -\infty$

Hydrodynamic Pressure:

$$P_d(x, y, z, t) = -\rho \Phi_t$$

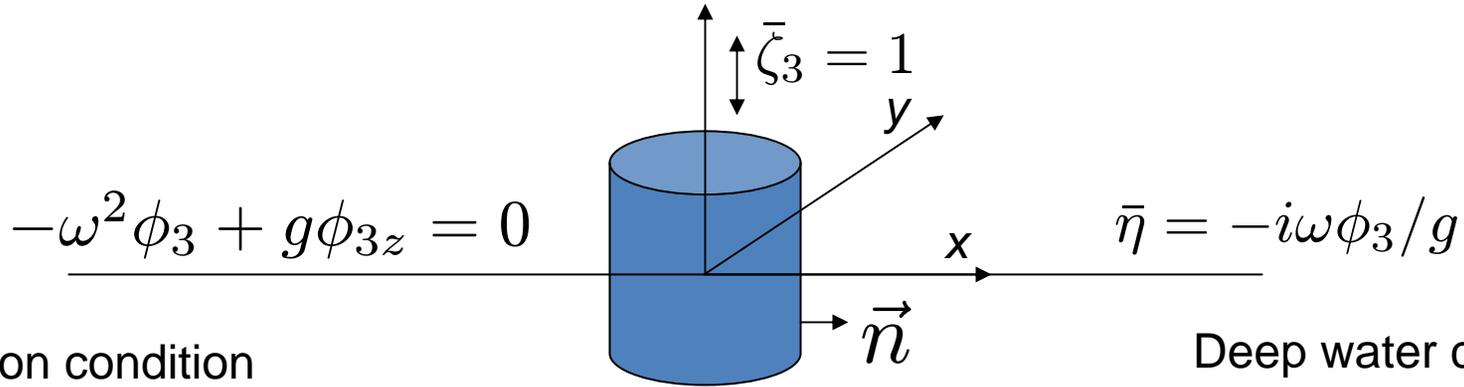
Radiation Force:

$$\vec{F}_R(t) = - \int_{S_B} P_d \vec{n} ds$$

Radiation Moment:

$$\vec{M}_R(t) = - \int_{S_B} P_d (\vec{x} \times \vec{n}) ds$$

# Frequency-Domain Formulation of Heave Radiation Problem



Radiation condition

Deep water condition:

$$\nabla^2 \phi_3(\vec{x}) = 0$$

$$\nabla \phi_3 \rightarrow 0 \quad \text{as} \quad z \rightarrow -\infty$$

Let:

$$\zeta_3(t) = \cos \omega t = \Re\{e^{i\omega t}\}$$

$$\Phi(\vec{x}, t) = \Re\{\phi_3(\vec{x})e^{i\omega t}\}$$

$$P_d(\vec{x}, t) = \Re\{p_d(\vec{x})e^{i\omega t}\}$$

$$\vec{F}_R(t) = \Re\{\vec{f}e^{i\omega t}\}$$

$$\vec{M}_R(t) = \Re\{\vec{m}e^{i\omega t}\}$$

$$p_d = -i\rho\omega\phi_3(\vec{x})$$

$$\vec{f} = -\int_{S_B} p_d \vec{n} dS$$

$$\vec{m} = -\int_{S_B} p_d(\vec{x} \times \vec{n}) dS$$

$$f_{3R} = i\rho\omega \int_{S_B} \phi_3 n_3 ds$$

$$F_{3R}(t) = -A_{33}\ddot{\zeta}_3(t) - B_{33}\dot{\zeta}_3(t) = \Re\{[\omega^2 A_{33} - i\omega B_{33}]e^{i\omega t}\}$$

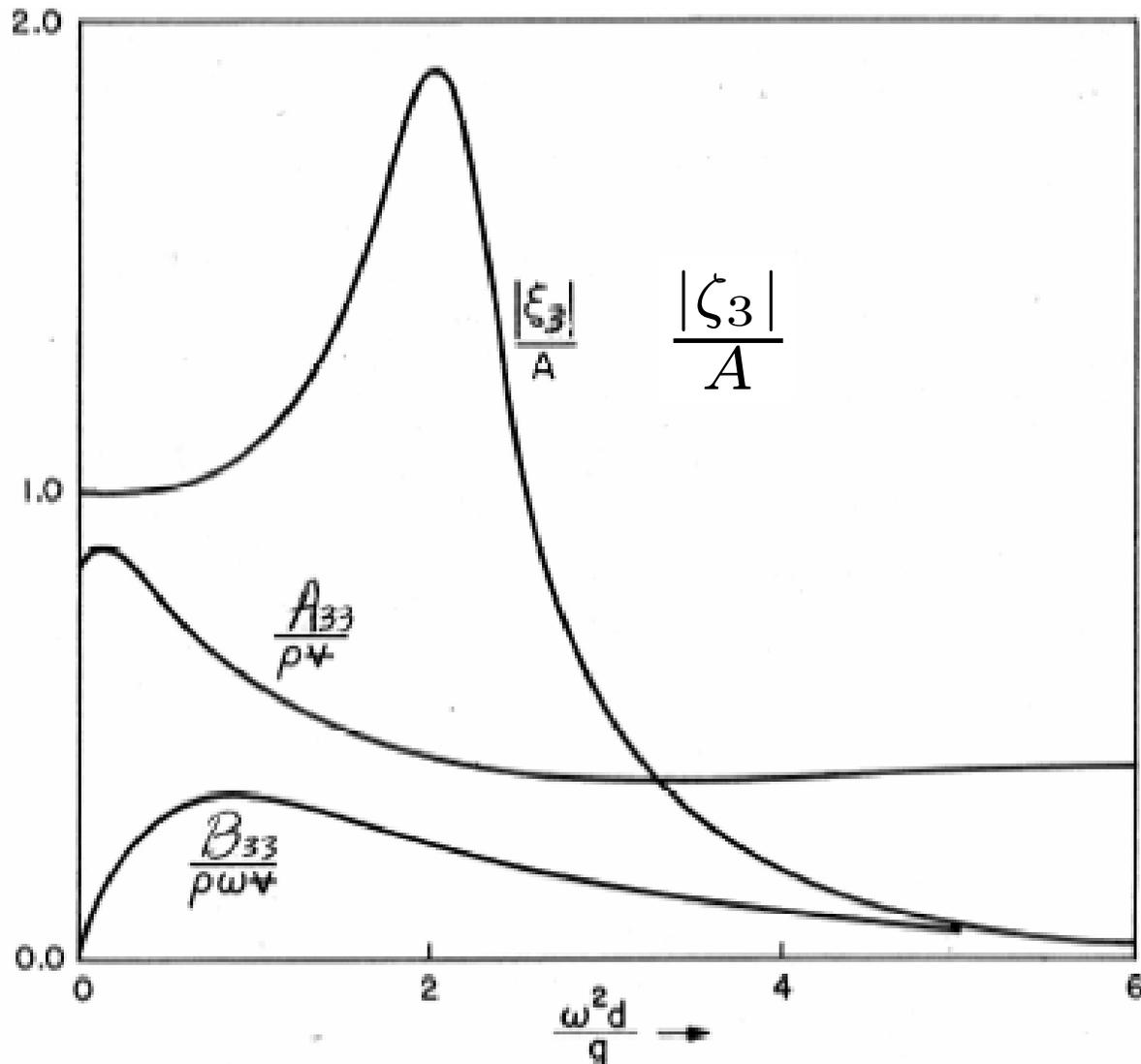
Thus,

$$A_{33} = \Re\left\{\frac{i\rho}{\omega} \int_{S_B} \phi_3 n_3 ds\right\}, \quad B_{33} = -\Im\left\{i\rho \int_{S_B} \phi_3 n_3 ds\right\}$$

$$A_{13} = \Re\left\{\frac{i\rho}{\omega} \int_{S_B} \phi_3 n_1 ds\right\}, \quad B_{13} = -\Im\left\{i\rho \int_{S_B} \phi_3 n_1 ds\right\}$$

$$A_{23}, B_{23}, \dots, A_{63}, B_{63}$$

- $A_{ij}$  and  $B_{ij}$  are symmetric, i.e.  $A_{ij} = A_{ji}$ ,  $B_{ij} = B_{ji}$ ,  $i=1, \dots, 6$ ;  $j=1, \dots, 6$
- $A_{ij}$  and  $B_{ij}$  are functions of frequency  $\omega$



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### 6.24

Added-mass and damping coefficients for a sphere of diameter  $d$ , half submerged in deep water.  $\nabla$  is the displaced volume  $\pi d^3/12$ . Also shown is the heave-response ratio (190).

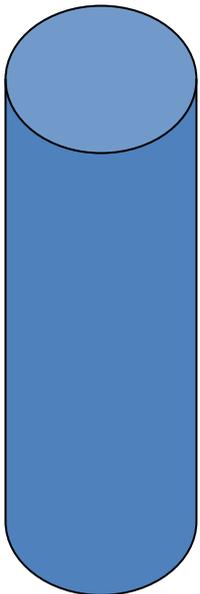
## Examples: Added Mass at Low Frequency

At low frequencies, i.e.  $\omega \rightarrow 0$ :

$$\frac{d^2\Phi}{dt^2} \sim \omega^2 \rightarrow 0 \quad \text{as } \omega \rightarrow 0$$

Thus, the free surface boundary condition becomes:  $\Phi_z = 0$

(1) slender vertical circular cylinder



Surge added mass

$$m_{11} = \rho\pi R^2 h$$

Wave damping = 0

(2) slender ship with a semi-circle cross section

Sway added mass

$$m_{11} = \rho \frac{\pi R^2}{2} L$$

Wave damping = 0

## Examples: Added Mass at High Frequency

At high frequencies, i.e.  $\omega \rightarrow \infty$ :

$$\frac{d^2\Phi}{dt^2} \sim \omega^2 \rightarrow \infty \quad \text{as } \omega \rightarrow \infty$$

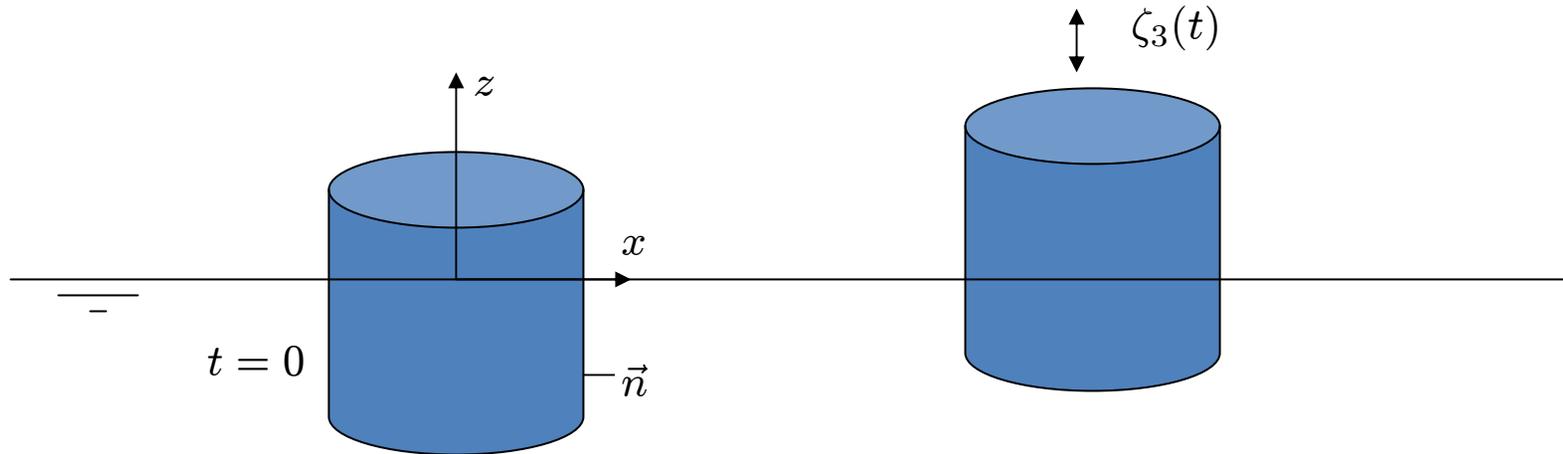
Thus, the free surface boundary condition becomes:  $\Phi=0$

Slender ship with a semi-circle cross section:

$$\text{Heave added mass: } m_{33} = \rho \frac{\pi R^2}{2} L$$

$$\text{Wave damping} = 0$$

# Hydrostatic Restoring Effect in Body Motion



Wetted body surface:  $S_0$

$$S_B(t) = S_0 + \Delta S(t)$$

Hydrostatic pressure:  $P_s = -\rho g z$

$$\text{Hydrostatic force: } \vec{F}_s = - \int_{S_B(t)} P_s \vec{n} ds = - \int_{S_0} P_s \vec{n} ds - \int_{\Delta S(t)} P_s \vec{n} ds$$

┌───────────▶ Hydrostatic restoring effect

└───────────▶ Balanced by other forces at equilibrium

$$F_{s3}(t) = \rho g \int_{V_{ol}} dV_{ol} = \bar{F}_{s3} - \rho g S_{wl} \zeta_3(t)$$

$S_{wl}$  : Water plane surface area of the body

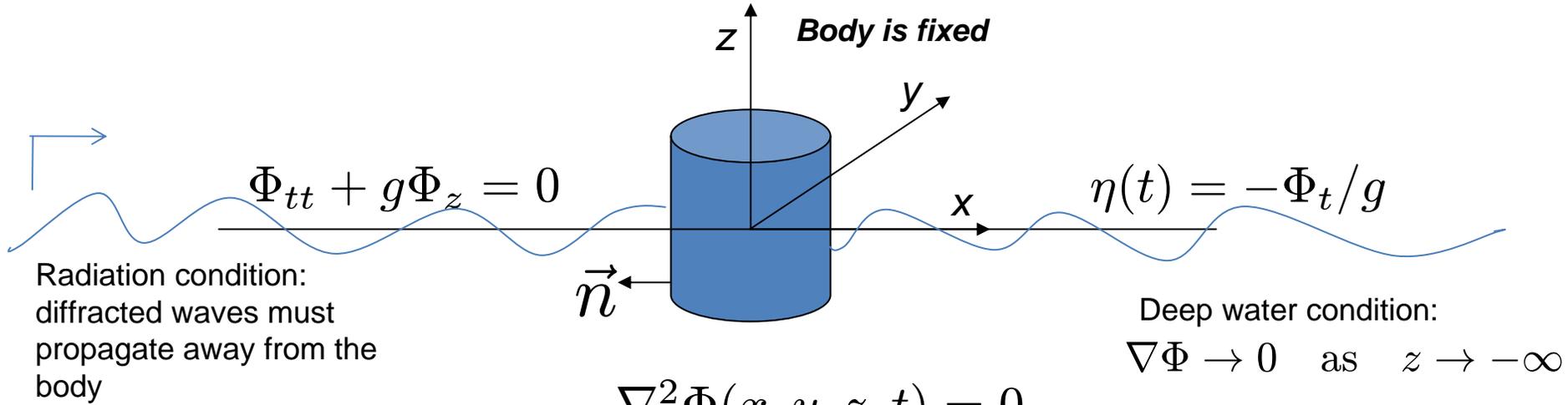
$C_{33} = \rho g S_{wl}$  : Hydrostatic restoring coefficient (i.e. spring constant)

$$\text{Hydrostatic moment } \vec{M}_s = - \int_{S_B(t)} P_s (\vec{x} \times \vec{n}) ds = - \int_{S_0} P_s (\vec{x} \times \vec{n}) ds - \int_{\Delta S(t)} P_s (\vec{x} \times \vec{n}) ds$$

Hydrostatic restoring force/moment:  $F_{si3}(t) = -C_{i3} \zeta_3(t), \quad i = 1, \dots, 6$

In general  $F_{sij}(t) = - \sum_{j=1}^6 C_{ij} \zeta_j, \quad i = 1, \dots, 6$  where  $C_{ij}$  is 6x6 restoring coef. matrix

# Wave Diffraction Problem



$$\nabla^2 \Phi(x, y, z, t) = 0$$

$$\vec{F}_E, \vec{M}_E = ???$$

Total potential:  $\Phi(\vec{x}, t) = \Phi_I(\vec{x}, t) + \Phi_D(\vec{x}, t)$

$\Phi_I$  : Incident wave potential (of a plane progressive wave)

$\Phi_D$  : Diffracted (or scattered) wave potential

Total dynamic pressure:  $P_d = -\rho\Phi_{It} - \rho\Phi_{Dt}$

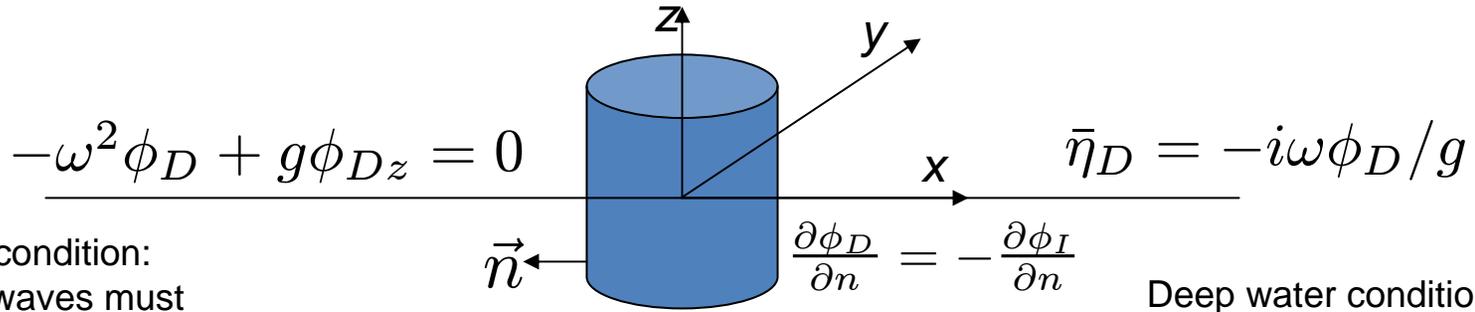
└─ Diffraction effect

Total wave excitations (force/moment):  $\vec{F}_E(t) = \int_{S_B} \rho\Phi_{It}\vec{n}ds + \int_{S_B} \rho\Phi_{Dt}\vec{n}ds = \vec{F}_I + \vec{F}_D$

Froude-Krylov force

$$\vec{M}_E(t) = \int_{S_B} \rho\Phi_{It}(\vec{x} \times \vec{n})ds + \int_{S_B} \rho\Phi_{Dt}(\vec{x} \times \vec{n})ds = \vec{M}_I + \vec{M}_D$$

# Frequency-Domain Formulation of Wave Diffraction Problem



Radiation condition:  
diffracted waves must  
propagate away from the  
body

$$\vec{n} \leftarrow \quad \frac{\partial \phi_D}{\partial n} = -\frac{\partial \phi_I}{\partial n}$$

$$\nabla^2 \phi_D(x, y, z) = 0$$

Deep water condition:  
 $\nabla \phi_D \rightarrow 0$  as  $z \rightarrow -\infty$

Incident  $\eta_I(x, y, t) = a \cos(\omega t - kx) = \Re\{ae^{-ikx}e^{i\omega t}\} = \Re\{\bar{\eta}_I e^{i\omega t}\}$

Diffraction potential:  $\Phi_D(\vec{x}, t) = \Re\{\phi_D(\vec{x})e^{i\omega t}\}$

Total dynamic pressure:  $P_d(\vec{x}, t) = \Re\{p_d(\vec{x})e^{i\omega t}\}, \quad p_d(\vec{x}) = p_I + p_D$   
 $p_I = -i\rho\omega\phi_I, \quad p_D = -i\rho\omega\phi_D$

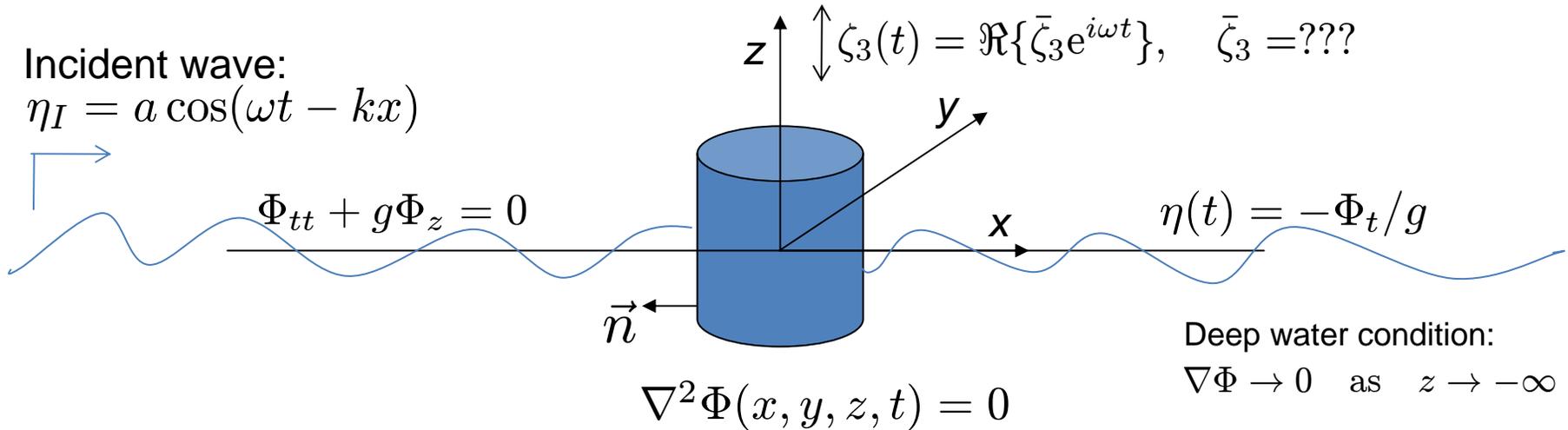
Total wave excitations:  $\vec{F}_E(t) = \Re\{\vec{f}_E e^{i\omega t}\}, \quad \vec{f}_E = \vec{f}_{EI} + \vec{f}_{ED}$

Froude-Krylov force:  $\vec{f}_{EI} = -\int_{S_B} p_I \vec{n} ds = i\rho\omega \int_{S_B} \phi_I \vec{n} ds$

Diffraction force:  $\vec{f}_{ED} = -\int_{S_B} p_D \vec{n} ds = i\rho\omega \int_{S_B} \phi_D \vec{n} ds$

$\vec{M}_E(t) = \Re\{\vec{m}_E e^{i\omega t}\}, \quad \vec{m}_E = \vec{m}_{EI} + \vec{m}_{ED}$

# Heave Response of A Floating Body to Ambient Waves



- Decompose the total problem into a sum of diffraction problem and radiation problem:

$$\Phi(\vec{x}, t) = \underbrace{\Phi_I(\vec{x}, t) + \Phi_D(\vec{x}, t)}_{\text{Diffraction problem}} + \underbrace{\Phi_R(\vec{x}, t)}_{\text{Radiation problem}}$$

- From the diffraction problem:

$$\text{Wave excitation force: } F_{E3}(t) = \Re\{f_{E3} e^{i\omega t}\}, \quad f_{E3} = f_{3I} + f_{3D}$$

- From the radiation problem:

$$\text{Wave radiation force: } F_{R3}(t) = -A_{33}\ddot{\zeta}_3(t) - B_{33}\dot{\zeta}_3(t) = \Re\{(-\omega^2 A_{33} - i\omega B_{33})\bar{\zeta}_3 e^{i\omega t}\}$$

$$\text{Hydrostatic restoring force: } F_{s3} = -C_{33}\zeta_3(t) = \Re\{(-C_{33}\bar{\zeta}_3) e^{i\omega t}\}$$

- Total hydrodynamic and hydrostatic forces:

$$F_{E3} + F_{R3} + F_{s3} = \Re\{[f_{E3} - (\omega^2 A_{33} + i\omega B_{33} + C_{33})\bar{\zeta}_3]e^{i\omega t}\}$$

- Applying Newton's second law:

$$F_{E3} + F_{R3} + F_{s3} = m\ddot{\zeta}_3(t)$$

$$\Re\{(-\omega^2 m)e^{i\omega t}\} = \Re\{[f_{E3} - (-\omega^2 A_{33} + i\omega B_{33} + C_{33})\bar{\zeta}_3]e^{i\omega t}\}$$



**Equation of Motion:**  $[-\omega^2(m + A_{33}) + i\omega B_{33} + C_{33}]\bar{\zeta}_3 = f_{3I} + f_{3D}$

- Heave motion amplitude:  $\bar{\zeta}_3 = \frac{f_{3I} + f_{3D}}{-\omega^2(m + A_{33}) + i\omega B_{33} + C_{33}}$

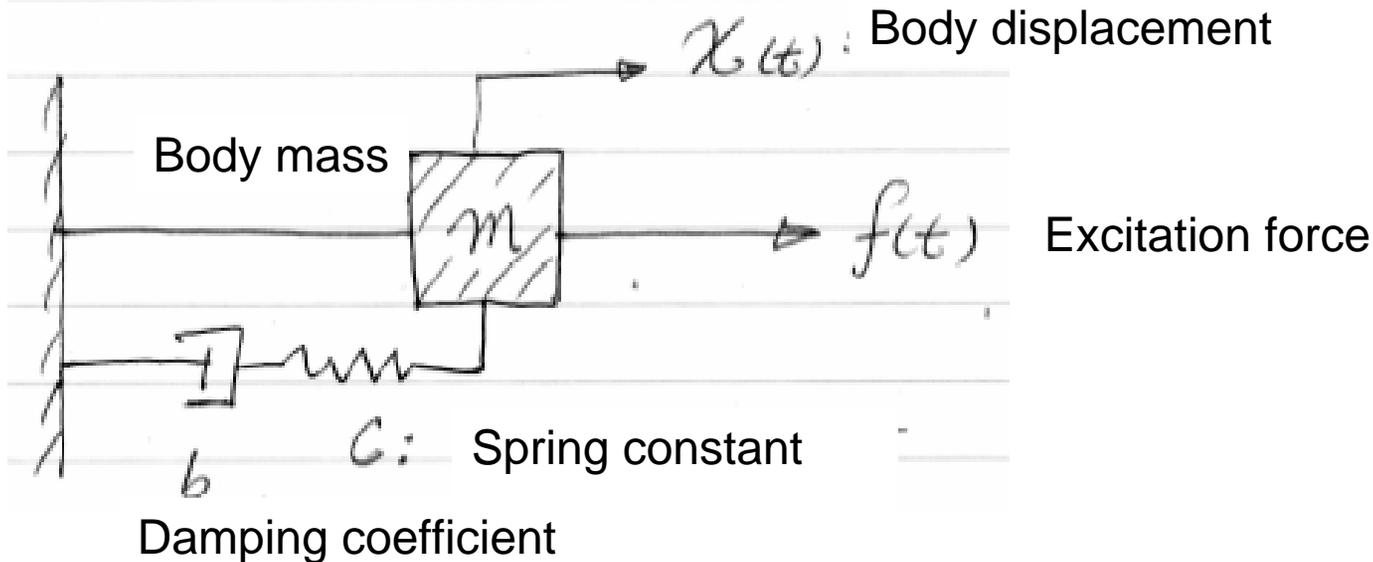


**Response Amplitude Operator (RAO):**  $\frac{\bar{\zeta}_3(\omega)}{a} = \frac{(f_{3I} + f_{3D})/a}{-\omega^2(m + A_{33}) + i\omega B_{33} + C_{33}}$

$$= \left| \frac{(f_{3I} + f_{3D})/a}{-\omega^2(m + A_{33}) + i\omega B_{33} + C_{33}} \right| e^{i\alpha}$$

Heave natural frequency:  $-\omega_{n3}(m + A_{33}) + C_{33} = 0 \quad \rightarrow \quad \omega_{3n} = \left(\frac{C_{33}}{m + A_{33}}\right)^{\frac{1}{2}}$

# Analogy to a Simple Mass-Spring-Dashpot System



Equation of motion:  $m\ddot{x} + b\dot{x} + cx = f(t)$

For harmonic excitation,  $f(t) = f_0 \cos \omega t$ , we have harmonic response:  $x(t) = x_0 \cos(\omega t + \alpha)$ ,  $x_0 = ??$

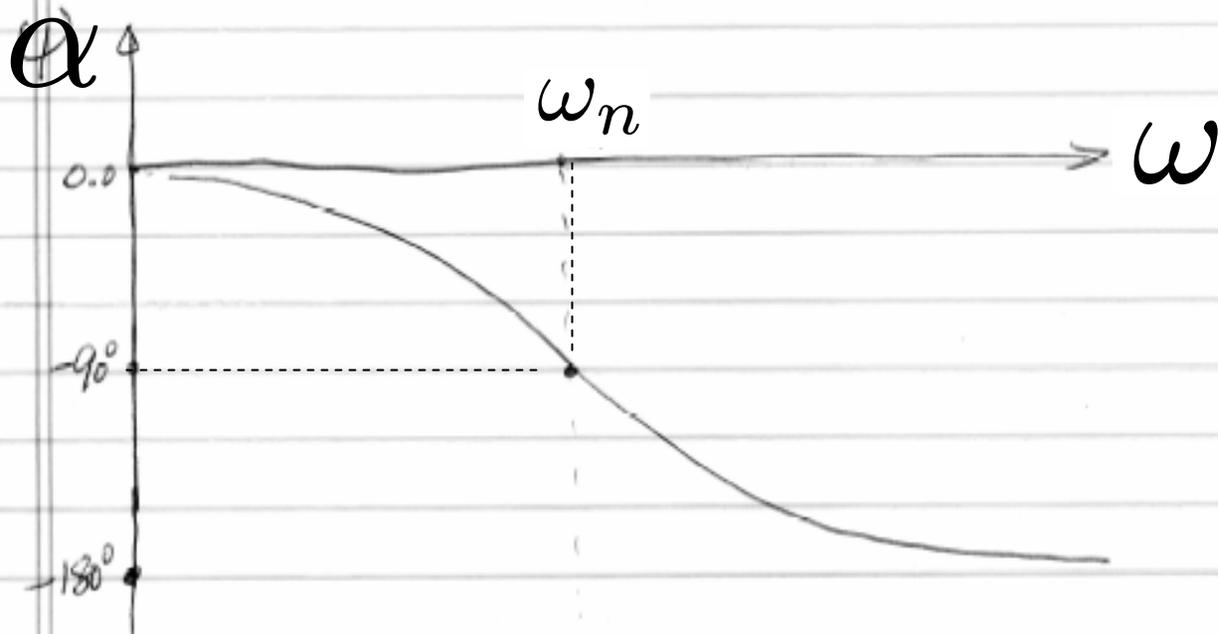
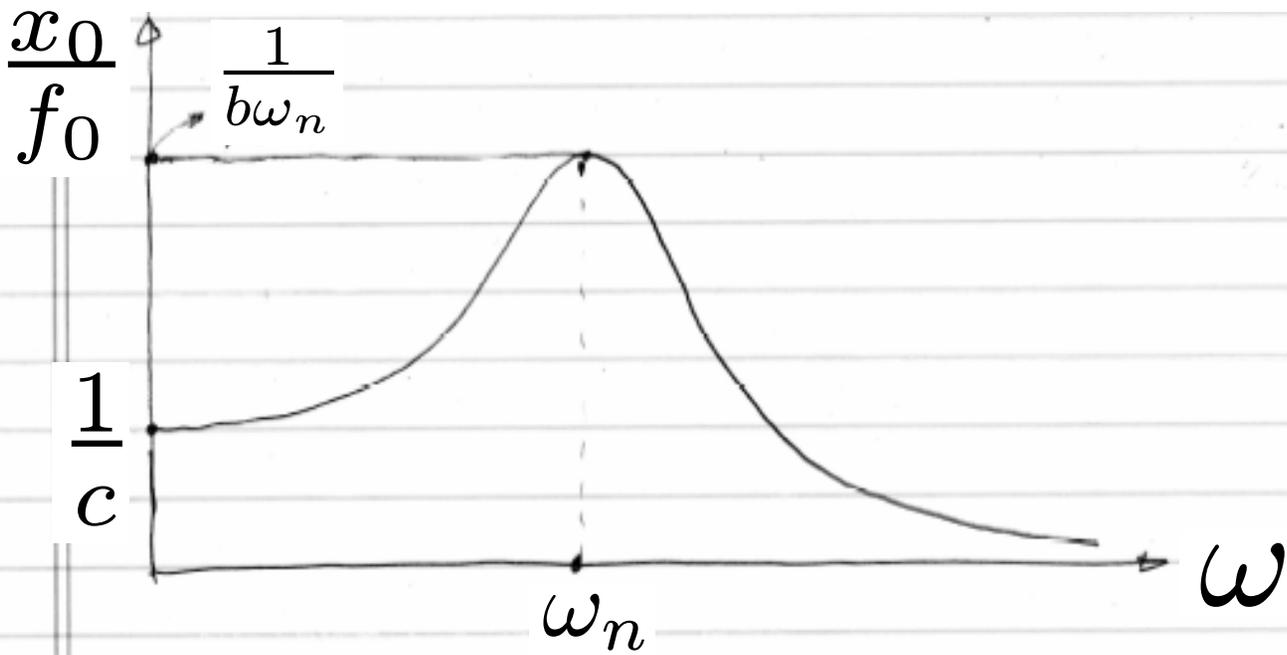
From equation of motion, we obtain:  $x_0 = \frac{f_0}{[(c - m\omega^2)^2 + b^2\omega^2]^{1/2}}$  and  $\alpha = \tan^{-1} \left( \frac{-b\omega}{c - m\omega^2} \right)$

Natural frequency:  $\omega_n = (c/m)^{1/2}$

$$\frac{x_0}{f_0} = \frac{1}{c}, \quad \text{at } \omega=0$$

$$\frac{x_0}{f_0} = \frac{1}{b\omega_n}, \quad \text{at } \omega=\omega_n$$

$$\frac{x_0}{f_0} \rightarrow 0, \quad \text{as } \omega \rightarrow \infty$$



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