

# **2.019 Design of Ocean Systems**

## **Lecture 7**

### **Seakeeping (III)**

**February 25, 2011**

# Motions and Wave Loads on a Barge

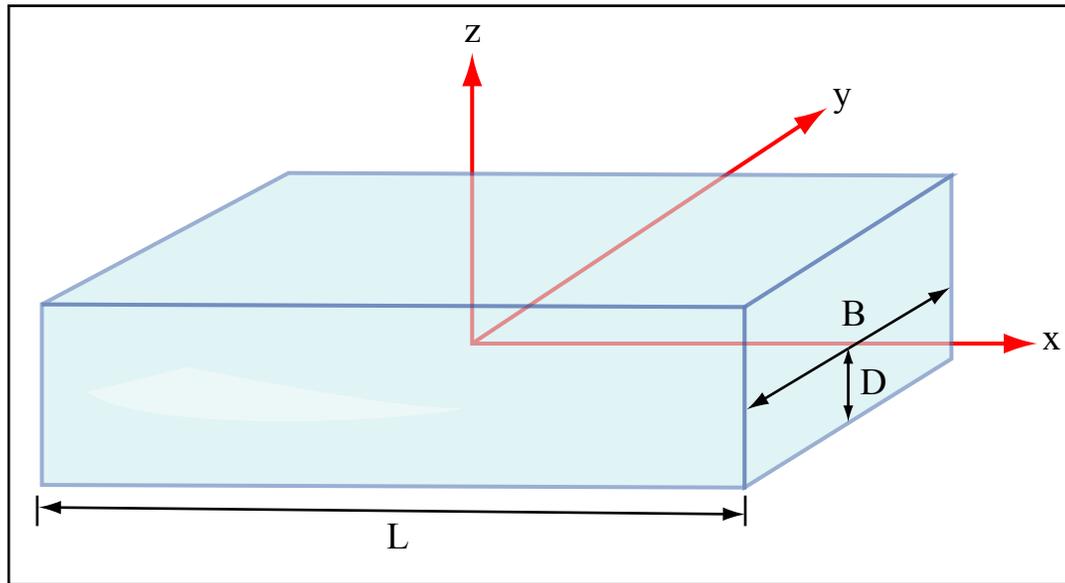


Image by MIT OpenCourseWare.

A regular plane progressive incident wave in deep water travels along the x-direction:

$$\eta_I(x, t) = a \cos(\omega t - kx)$$

$$\Phi_I(x, y, z, t) = -\frac{ga}{\omega} e^{kz} \sin(\omega t - kx)$$

To find the wave force and motion of the barge in the vertical direction using long-wave and strip theory assumptions.

# Heave Wave Excitation on a Barge (I)

$$F_{E3} = F_{I3} + F_{D3}$$

Using the strip theory (which is valid for  $B/L \ll 1$ ), we have:

$$F_{E3} = \int_{-L/2}^{L/2} f_{E3}(x)dx, \quad F_{I3} = \int_{-L/2}^{L/2} f_{I3}(x)dx, \quad F_{D3} = \int_{-L/2}^{L/2} f_{D3}(x)dx$$

Froude Krylov force component:

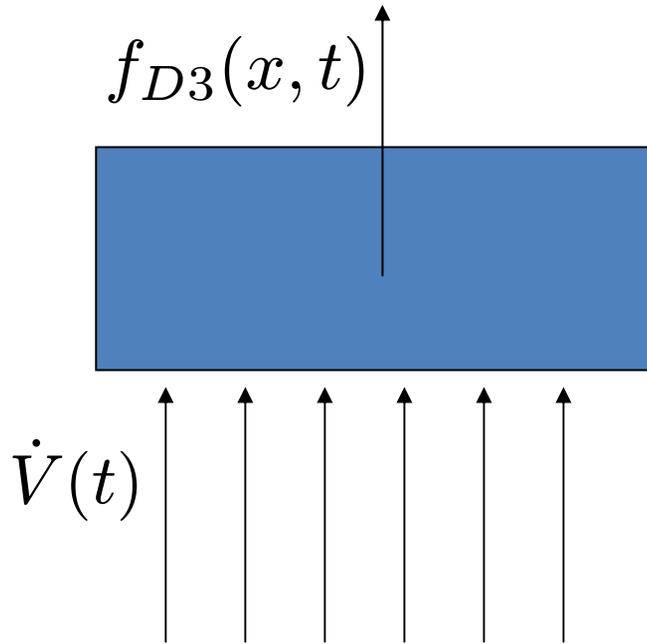
$$\begin{aligned} f_{I3}(x) &= - \int_{-B/2}^{B/2} P_I(x) n_z dy \\ &= \int_{-B/2}^{B/2} (-\rho \Phi_t(x, y, z = -D, t)) dy \\ &= B \rho g a e^{-kD} \cos(\omega t - kx) \end{aligned}$$

$$\begin{aligned} F_{E3} &= \int_{-L/2}^{L/2} f_{E3}(x) dx \\ &= \int_{-L/2}^{L/2} B \rho g a e^{-kD} \cos(\omega t - kx) dx \\ &= \rho g a B \left( \frac{2}{k} \right) e^{-kD} \sin \frac{kL}{2} \cos \omega t \end{aligned}$$

In the limit  $\omega \rightarrow 0$ :  $F_{E3} \rightarrow \rho g a B L \cos \omega t = \rho g \eta(t) (BL)$

## Heave Wave Excitation on a Barge (II)

Long-wave assumption: wave motion is a flow slowly varying in space and time. The wave diffraction effect is approximated by the added mass effect.



$$f_{D3}(x, t) = A_{33}^{2D}(x) \dot{V}(x, t)$$

$$\begin{aligned} V(x, t) &= \Phi_{Iz}(x, z = -D/2, t) \\ &= -\frac{g a k}{\omega} e^{-kD/2} \sin(\omega t - kx) \end{aligned}$$

$$\dot{V}(x, t) = -g a k e^{-kD/2} \cos(\omega t - kx)$$

$$f_{D3}(x, t) = -g a k e^{-kD/2} A_{33}^{2D} \cos(\omega t - kx)$$

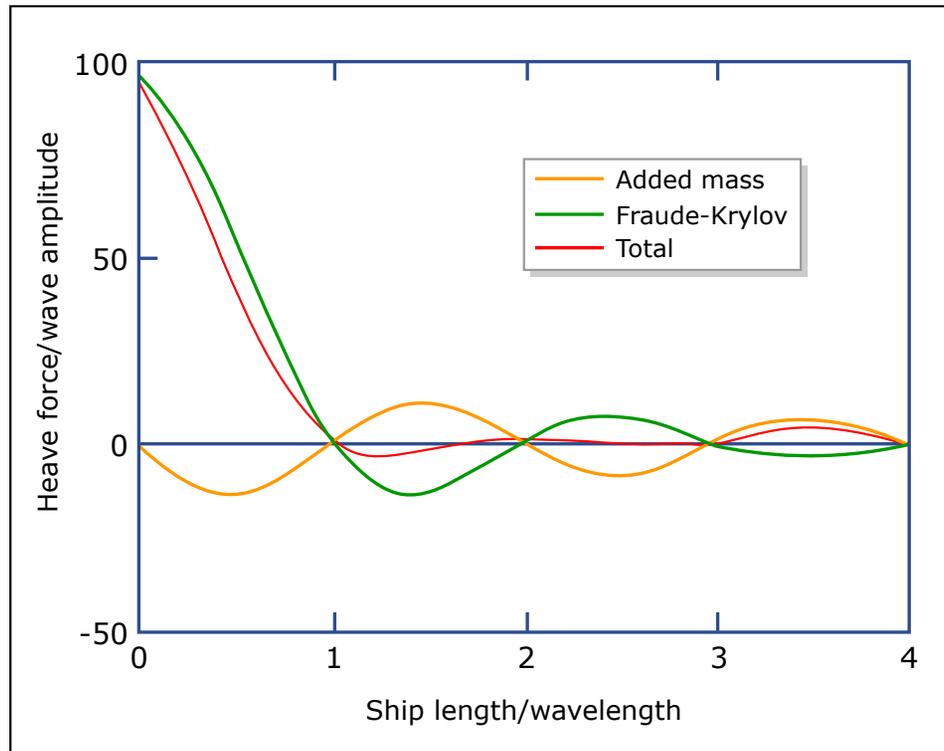
$$F_{D3}(t) = \int_{-L/2}^{L/2} f_{D3}(x, t) dx = -2g a e^{-kD/2} A_{33}^{2D} \sin \frac{kL}{2} \cos \omega t$$

In the limit  $\omega \rightarrow 0$ :  $F_{D3} \rightarrow 0$

## Heave Wave Excitation on a Barge (III)

$$F_{E3}(t) = F_{I3} + F_{D3} = \underbrace{[\rho g a B e^{-kD}]}_{\text{Froude Krylov}} - \underbrace{g A_{33}^{2D} k a e^{-kD/2}}_{\text{Added mass effect}} \left(\frac{2}{k}\right) \sin \frac{kL}{2} \cos \omega t$$

$$A_{33}^{2D} = C_a \left[ \rho \frac{\pi}{2} (B/2)^2 \right], \quad C_a \sim 1.0$$



## Radiation Force

Added mass coefficient:  $A_{33} = \int_{-L/2}^{L/2} A_{33}^{2D}(x) dx = LA_{33}^{2D}$

Wave damping coefficient:  $B_{33} \rightarrow 0$  with long-wave assumption

Radiation force:  $F_{R3} = -A_{33}\ddot{\zeta}_3(t) = -LA_{33}^{2D}\ddot{\zeta}_3(t)$

## Restoring Force

$$F_{S3} = -C_{33}\zeta_3(t) = -\rho gBL\zeta_3(t)$$

## Equation of Motion

$$(M + A_{33})\ddot{\zeta}_3 + B_v\dot{\zeta}_3 + \rho gBL\zeta_3(t) = F_{E3}(t)$$

If  $B_v = 0$ ,  $\zeta_3(t) = \bar{\zeta}_3 \cos(\omega t)$

$$\bar{\zeta}_3(\omega) = \frac{[\rho g a B e^{-kD} - g A_{33}^{2D} k a e^{-kD/2}] \left(\frac{2}{k}\right) \sin \frac{kL}{2}}{-\omega^2 (M + A_{33}) + \rho g B L}$$

In the limit  $\omega \rightarrow 0$ :

$$\bar{\zeta}_3 = \frac{\rho g B L a}{-\omega^2 (M + A_{33}) + \rho g B L} = a$$

$$\zeta_3(t) = a \cos \omega t = \eta(x = 0, t)$$

Barge responds to move like a fluid particle in the limit of very long wave.

### Natural Frequency

$$-\omega_n^2 (M + A_{33}) + \rho g B L = 0$$

$$\omega_n = \left( \frac{\rho g B L}{M + A_{33}} \right)^{1/2} = \left( \frac{g B}{B D + A_{33}^2 D} \right)^{1/2} = \left( \frac{g}{D + C_a(\pi/8) B} \right)^{1/2}$$

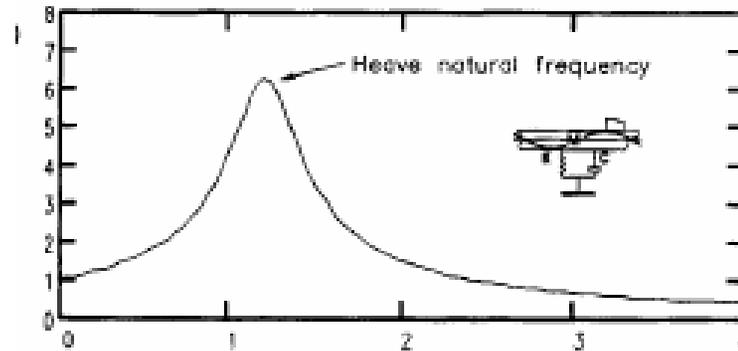
Natural period:  $T_n = \frac{2\pi}{\omega_n}$   $T_n$  increases with D and B.

For example, for D=20m, B=60m, we have  $T_n = 13s$ .

# Sample Results for Heave Motion

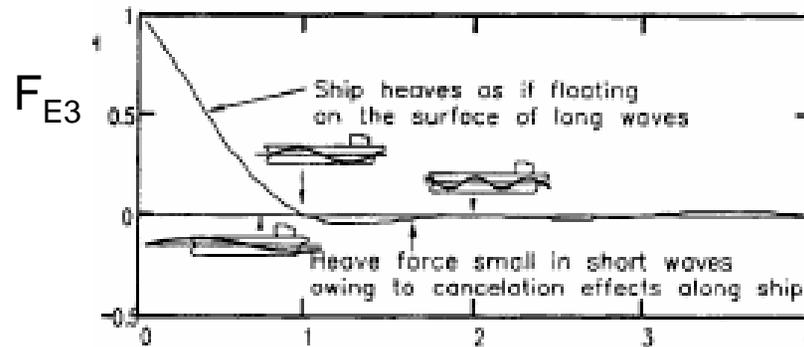
Draft:  $D=12$  m  
 Width:  $B=40$  m

Natural period:  $T_n = 11.4$  s  
 $B_v = 8\%$  critical damping

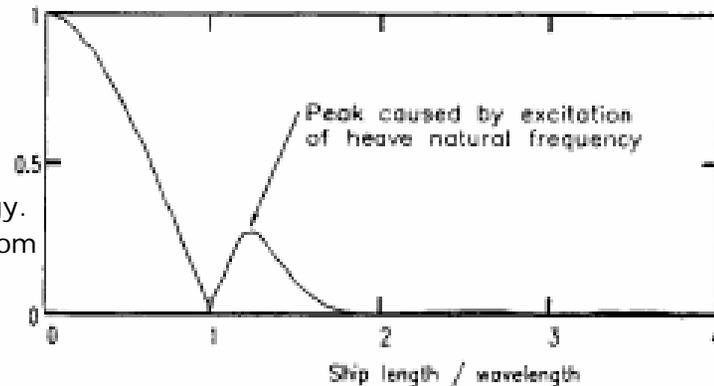


Response of the ship as a mass, spring, damper system

(Damping ratio taken as 0.08)



RAO



## Pitch Motion and Wave Loads on a Barge

$$\begin{aligned} \text{Wave Excitation: } F_{E5}(t) &= \int_{-L/2}^{L/2} -x f_{E3} dx \\ &= \left[ \rho g B a e^{-kD} - g k a e^{-kD/2} A_{33}^{2D} \right] \int_{-L/2}^{L/2} [-x \cos(\omega t - kx)] dx \end{aligned}$$

$$\text{Added mass and wave damping: } A_{55} = \int_{-L/2}^{L/2} x^2 A_{33}^{2D} dx, \quad B_{55} = 0 \quad \text{as } \omega \rightarrow 0$$

$$\text{Radiation moment: } F_{R5} = -A_{55} \ddot{\zeta}_5(t) - B_{55} \dot{\zeta}_5(t)$$

$$\text{Hydrostatic restoring moment: } F_{S5} = -C_{55} \zeta_5(t) = -[\rho g \nabla(Z_B - B_G) + \rho g \int_{A_{wp}} x^2 ds] \zeta_5(t)$$

$$\text{Moment of inertia: } I_{55} = \rho D B \int_{-L/2}^{L/2} x^2 dx$$

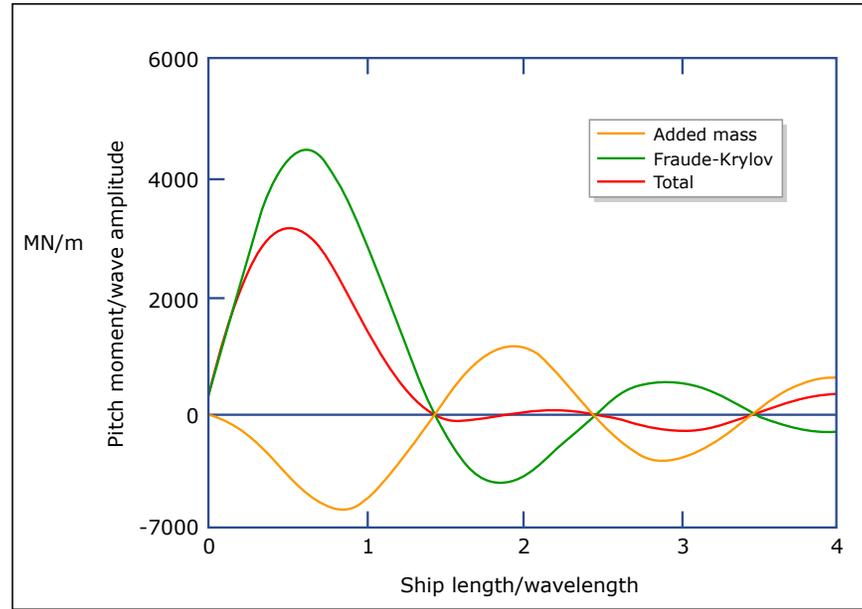
From the equation of motion for pitch, we can get pitch motion:  $\zeta_5(t) = \bar{\zeta}_5 \sin \omega t$

$$\frac{\bar{\zeta}_5}{a} = \dots$$

# Sample Results for Pitch Motion

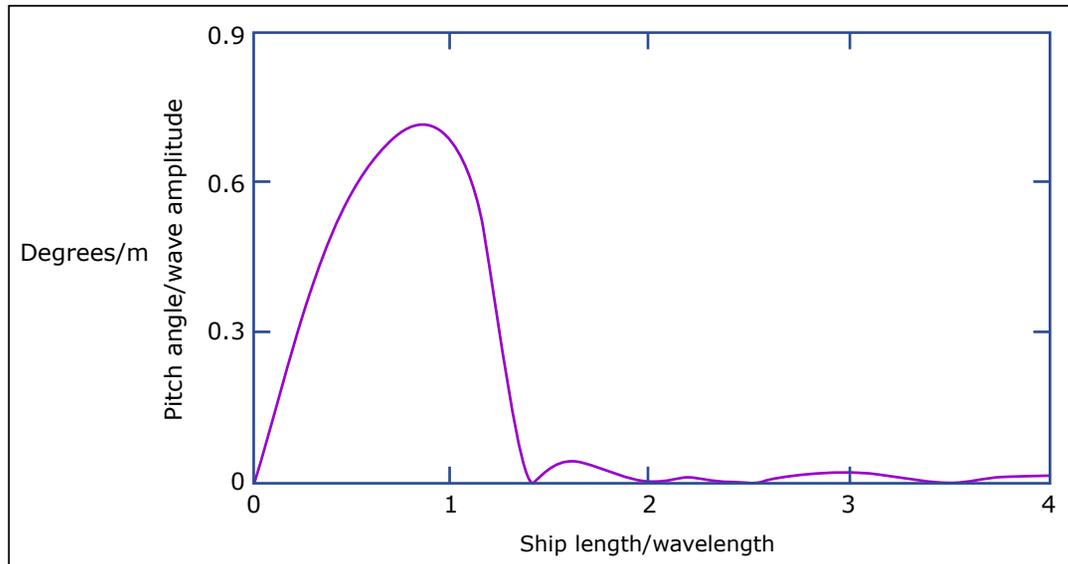
Draft:  $D=12$  m  
Width:  $B=40$  m

$B_v = 8\%$  critical damping



$F_{E5}$

RAO



Deck elevation at bow:

$$Z_D = \zeta_3(t) - (L/2)\zeta_5(t) + H \quad \text{where } H \text{ is deck height}$$

Bottom elevation at bow:

$$Z_B = \zeta_3(t) - (L/2)\zeta_5(t) - D \quad \text{where } D \text{ is draft}$$

Wave elevation at bow:

$$Z_w = \eta(x = L/2, t) = a \cos(\omega t - kL/2)$$

If  $Z_D < Z_w$ , wave overtopping occurs;  
If  $Z_B > Z_w$ , ship bottom is out of water

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