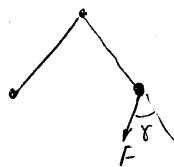


Session 11

W

F: follower force

$$\mathbf{F} = -\nabla V$$

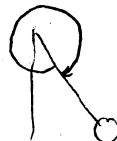
$$\left. \begin{aligned} F_x &= -\frac{\partial V}{\partial x} \\ F_y &= -\frac{\partial V}{\partial y} \end{aligned} \right\} \quad \frac{\partial}{\partial y} F_x = \frac{\partial}{\partial x} F_y$$

→ \mathbf{F} is not potentialworks done by \mathbf{F}

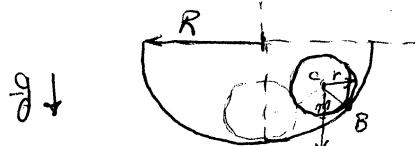
$$W_{12} = |\mathbf{F}| \sin \theta \cdot L (\varphi_2 - \varphi_1)$$

Note: W_{12} is not path-independent

This system is locally Conservative



Example



$$1 \text{ DOF} = 3 - 2 = 1 \rightarrow \text{choose } q$$

as a generalized coordinate

Question: Eq of motion
Reaction forcesTo eliminate the role of constraint forces, take any momentum principle w.r.t. \mathbf{P}

$$\underline{H}_B + \underline{V}_B \times \underline{P} = \underline{M}_B$$

$$\underline{M}_B = m g r \sin q \underline{k}$$

$$\underline{V}_B \parallel \underline{v}_C \Rightarrow \underline{V}_B \parallel \underline{P}$$

$$\underline{H}_B \neq \underline{E}_B \Leftrightarrow !$$

$$\Rightarrow \underline{H}_B = \underline{H}_C + \underline{P} \times \underline{r}_{CB}$$

$$= \underline{L}_C \underline{w} \underline{k} + \underline{P} \times \underline{r}_{CB}$$

$$= \frac{1}{2} m r^2 (-\dot{\theta}) \underline{k} + m(R-r) \dot{q} (R \dot{\theta} \underline{i} + \sin q \dot{j}) \times \underline{r} (\dot{\theta} \underline{i} - \cos q \dot{j})$$

$$\text{Note: } R\dot{\theta} = r(\dot{\theta} + \dot{q}) \Rightarrow \dot{\theta} = \frac{R-r}{r} \dot{q}$$

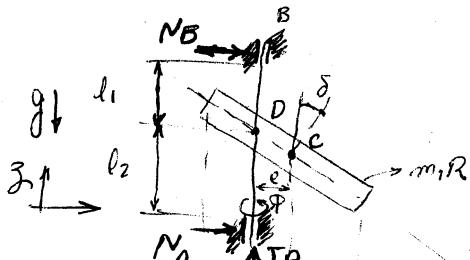
$$\Rightarrow \underline{H}_B = -\frac{3}{2} m r (R-r) \dot{q} \underline{k}$$

$$\Rightarrow \text{Eq. of motion: } -\frac{3}{2} m r (R-r) \ddot{q} = m g r \sin q \Rightarrow \ddot{q} + \frac{2g}{3(R-r)} \sin q = 0$$

$$\text{For frequency of small oscillations linearized} \Rightarrow \ddot{q} + \frac{2g}{3(R-r)} q = 0 \quad \omega = \sqrt{\frac{2g}{3(R-r)}}$$

Example

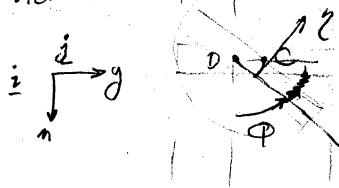
eccentric skewed disk on a rotating shaft



Reaction forces at A & B?

$$\# \text{DOF} = 6 - 3 - 2 = 1 \rightarrow \text{use } \phi$$

top view



Linear momentum principle:

$$\dot{P} = m\dot{\alpha} = (N_A + N_B)C_i\dot{\alpha}^i + (N_A + N_B)C_j\dot{\alpha}^j + (T_A - Mg)k$$

$$\dot{r}_c = e(C_i\dot{\alpha}^i + C_j\dot{\alpha}^j) + z_c \dot{k}$$

$$\Rightarrow \dot{a}_c = -e(\ddot{\alpha}\sum \dot{\alpha}^2 + \dot{\alpha}^2 \dot{\alpha})^i + e(\ddot{\alpha}\cos\phi - \dot{\alpha}^2 \dot{\alpha})^j$$

→ Linear momentum principle:

$$-em(\ddot{\alpha}\sin\phi + \dot{\alpha}^2 \cos\phi) = (N_A + N_B)C_i\dot{\alpha}^i$$

$$em(\ddot{\alpha}\cos\phi - \dot{\alpha}^2 \sin\phi) = (N_A + N_B)C_j\dot{\alpha}^j$$

$$0 = T_A - Mg$$

$$\boxed{T_A = Mg}$$

$$\Rightarrow -em\dot{\alpha}^2 = N_A + N_B$$

Angular momentum principle: $\dot{H}_c + \vec{V}_c \times \vec{P} = \vec{M}_c$

$$\vec{M}_c = N_B(l_1 + e\tan\delta) - N_A(l_2 - e\tan\delta) + T_Ae$$

$$\dot{H}_c = \underline{H}_c + \omega \times \underline{H}_c$$

relative to frame

$$\underline{H}_c = \underline{I}_c \omega = \begin{pmatrix} I_1 & 0 & 0 \\ 0 & I_2 & 0 \\ 0 & 0 & I_3 \end{pmatrix} \begin{pmatrix} -\dot{\alpha}\sin\delta \\ \dot{\alpha} \\ \dot{\alpha}\cos\delta \end{pmatrix}$$

$$= \begin{Bmatrix} -I_1\dot{\alpha}\sin\delta \\ 0 \\ I_3\dot{\alpha}\cos\delta \end{Bmatrix}$$

$$\Rightarrow \dot{H}_c = \underline{H}_c + \omega \times \underline{H}_c = \begin{Bmatrix} -I_1\dot{\alpha}\sin\delta \\ 0 \\ I_3\dot{\alpha}\cos\delta \end{Bmatrix} + \begin{Bmatrix} 0 & 0 & 0 \\ -4\sin\delta & 0 & 4\cos\delta \\ -4\delta & 0 & I_3\dot{\alpha}\cos\delta \end{Bmatrix}$$

$$= \begin{pmatrix} -I_1\dot{\alpha}\sin\delta \\ -\dot{\alpha}^2 \sin\delta \cos\delta (\Sigma_1 - \Sigma_3) \\ I_3 \dot{\alpha} \cos^2\delta \end{pmatrix}$$

$$I_1 = I_2 = \frac{1}{4}m[R^2 + \frac{h^2}{3}]$$

$$\Sigma_3 = \frac{1}{2}mR^2$$