

Eq. of small oscillation

$$\underline{M}\ddot{\underline{x}} + \underline{K}\underline{x} = 0$$

$$\underline{x} = \underline{q} - \underline{q}_0$$

$$\ddot{\underline{x}}(t) = \underline{Q} e^{\lambda t}$$

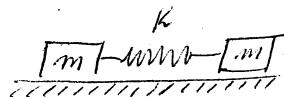
$$= \underline{Q} e^{\pm i\omega t}$$

↓

K is positive def.

$$\ddot{\underline{x}}(t) = \sum_j c_j \underline{q}_j e^{\pm i\omega t}$$

Example 2



Guess mode shapes

$$\underline{a}_1 = \begin{pmatrix} 1 \\ -1 \end{pmatrix} \quad \omega_1^2 = \frac{2K}{m}$$

$$\underline{a}_1 = \begin{pmatrix} 1 \\ 1 \end{pmatrix}, \quad \omega_2^2 = 0 \quad \text{rigid body mode}$$

For $\omega_2^2 \neq 0 \Rightarrow$ Normal Mode: $\ddot{\underline{x}}(t) = c_1 \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{\omega_1 t} + c_2 \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{\omega_2 t}$

Initial displacement

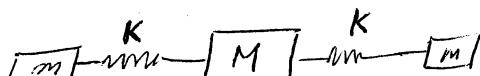
$$\ddot{\underline{x}}(0) = \begin{pmatrix} x_0 \\ x_0 \end{pmatrix} = \begin{pmatrix} c_1 \\ c_1 \end{pmatrix} \Rightarrow c_1 = x_0$$

$$\ddot{\underline{x}}(0) = \begin{pmatrix} v_0 \\ v_0 \end{pmatrix} = \begin{pmatrix} \omega_2 c_2 \\ \omega_2 c_2 \end{pmatrix} \Rightarrow c_2 = \frac{v_0}{\omega_2}$$

$$\Rightarrow \ddot{\underline{x}}(t) = x_0 \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{\omega_1 t} + \frac{v_0}{\omega_2} \begin{pmatrix} 1 \\ 1 \end{pmatrix} e^{\omega_2 t}$$

$$\neq \begin{pmatrix} x_0 \\ x_0 \end{pmatrix} + \begin{pmatrix} v_0 t \\ v_0 t \end{pmatrix}$$

Example 3.



guess mode shapes & natural frequencies

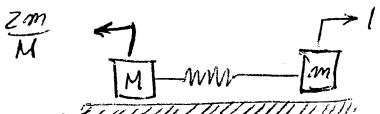
$$(1) \quad \underline{a}_1 = \begin{pmatrix} 1 \\ 0 \\ -1 \end{pmatrix}; \quad \omega_1^2 = \frac{K}{m}$$

$$(2) \quad \text{Rigid Body mode } \underline{a}_2 = \begin{pmatrix} 1 \\ 1 \\ 1 \end{pmatrix} \quad \omega_2^2 = 0$$

$$(3) \quad \underline{a}_3 = \begin{pmatrix} 1 \\ -A \\ 1 \end{pmatrix}$$

By Conservation of linear momentum if we subtract the rigid body motion from the full motion, the CM should not move

$$m_1 l + m_2 l - MA = 0 \Rightarrow A = \frac{m_1 l + m_2 l}{M} \rightarrow \underline{a}_3 = \begin{pmatrix} 1 \\ -A \\ 1 \end{pmatrix}$$



the equivalent stiffness for 3rd mass

$$\tilde{K} = K \cdot 1 + k \cdot \frac{2m}{M} = K(1 + \frac{2m}{M})$$

$$\Rightarrow \text{For the 1DOF oscillator } m\ddot{x}_3 + \tilde{K}x_3 = 0 \Rightarrow \omega_3^2 = \frac{\tilde{K}}{m} = \frac{K(1 + \frac{2m}{M})}{m}$$

$$M\ddot{x} + K\bar{x} = 0 \quad (*)$$

K pos. definite



- $\alpha_1, \dots, \alpha_n$ mode shapes
- $\omega_1^2, \dots, \omega_n^2$ natural frequencies (squared)

$$\text{General Solution (1)} \quad \underline{x}(t) = \sum_{j=1}^n (P_j \alpha_j e^{i\omega_j t} + Q_j \alpha_j e^{-i\omega_j t})$$

Complex Constants determined by I.C.

$$(2) \Rightarrow P_j = \bar{Q}_j = \gamma_j + i\delta_j$$

($\underline{x}(t)$ must be real)

Substitution of (2) into (1) gives

$$\begin{aligned} \underline{x}(t) &= \sum_j (2\gamma_j \cos \omega_j t - 2\delta_j \sin \omega_j t) \alpha_j \\ &= \sum_{j=1}^n \underbrace{2\sqrt{\gamma_j^2 + \delta_j^2}}_{C_j} \left(\underbrace{\frac{\gamma_j}{\sqrt{\gamma_j^2 + \delta_j^2}} \cos \omega_j t}_{\sin(\beta_j)} - \underbrace{\frac{\delta_j}{\sqrt{\gamma_j^2 + \delta_j^2}} \sin \omega_j t}_{\cos(\beta_j)} \right) \alpha_j \end{aligned}$$

$$\Rightarrow \underline{x}(t) = \sum_{j=1}^n C_j \alpha_j \sin(\omega_j t + \beta_j)$$

C_j, β_j real constants determined by I.C.

Orthogonality of mode shapes

$$(3) \quad -\omega_j^2 \underline{M} \alpha_j + \underline{K} \alpha_j = 0 \quad \left. \right\} \omega_j \neq \omega_K$$

$$(4) \quad -\omega_K^2 \underline{M} \alpha_K + \underline{K} \alpha_K = 0$$

$$\underline{a}_k^T (3) - \underline{a}_j^T (4) :$$

$$(\omega_2^2 - \omega_1^2) \underline{a}_k^T M \underline{a}_j = 0$$

$$\text{Used: } \underline{a}_k^T \underline{a}_j = \underline{a}_j^T \underline{a}_k$$

$$\underline{a}_k^T M \underline{a}_j = \underline{a}_j^T M \underline{a}_k$$

$$\Rightarrow \boxed{\underline{a}_k^T M \underline{a}_j = 0} \text{ for any } j \neq k$$

$$\text{Now: } \underline{a}_k^T (3) + \underline{a}_j^T (4) :$$

$$\boxed{\underline{a}_k^T \underline{a}_j = 0} \text{ for } j \neq k$$

We can use orthogonality properties to decouple the lin. eq of motion into a system of uncoupled linear oscillations

$$\text{let } \underline{x} = \underline{\Phi} \underline{y}$$

\underline{y} modal or Principal Coordinates
projections of \underline{x} onto a_1, \dots, a_n

$$M \dot{\underline{x}} + K \underline{x} = 0$$

$$\Rightarrow M \underline{\Phi} \ddot{\underline{y}} + K \underline{\Phi} \underline{y} = 0$$

$$\text{left multiply by } \underline{\Phi}^T \quad \underline{\Phi}^T M \underline{\Phi} \ddot{\underline{y}} + \underline{\Phi}^T K \underline{\Phi} \underline{y} = 0$$

$$\text{Note: } \underline{\Phi}^T M \underline{\Phi} = \begin{bmatrix} -a_1^T \\ -a_2^T \\ \vdots \\ -a_n^T \end{bmatrix} \begin{bmatrix} M \end{bmatrix} \begin{bmatrix} 1 & 1 & \dots & 1 \end{bmatrix} = \begin{bmatrix} m_1 & \dots & m_n \\ m_2 & \dots & \dots \\ \vdots & \ddots & \dots \\ m_n & \dots & m_n \end{bmatrix}$$

Same for K

$\Rightarrow (5)$ takes the form

$$\begin{pmatrix} m_1 & 0 \\ 0 & m_1 \end{pmatrix} \ddot{\underline{y}} + \begin{pmatrix} k_1 & 0 \\ 0 & k_n \end{pmatrix} \underline{y} = 0$$

$$\boxed{\ddot{y}_j + \frac{k_j}{m_j} y_j = 0} \quad j=1, \dots, n$$

$$\omega_j^2$$

