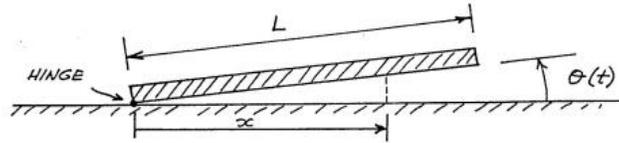


MIT Department of Mechanical Engineering
2.25 Advanced Fluid Mechanics

Problem 3.05

This problem is from "Advanced Fluid Mechanics Problems" by A.H. Shapiro and A.A. Sonin

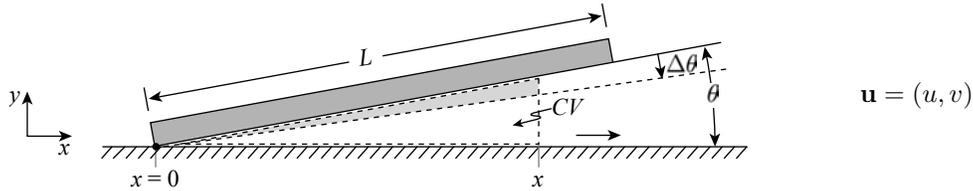


A long, flat plate of breadth L (L being small compared with the length perpendicular to the sketch) is hinged at the left side to a flat wall, and the gap between the plate and wall is filled with an incompressible liquid of density ρ . If the plate is at a *small* angle $\theta(t)$ and is depressed at an angular rate

$$\omega(t) = -\frac{d\theta}{dt},$$

obtain an expression for the average liquid speed $u(x, t)$ in the x -direction at station x and time t .

Solution:



Conservation of mass:

$$\left(\begin{array}{c} \text{time rate of change} \\ \text{of mass in the volume} \end{array} \right) = \left(\begin{array}{c} \text{rate of mass} \\ \text{flow in} \end{array} \right) - \left(\begin{array}{c} \text{rate of mass} \\ \text{flow out} \end{array} \right) \tag{3.05a}$$

If the plate moves \$\Delta\theta\$ in time, \$\Delta t\$, fluid in the shaded area, *must flow out* through the control surface (unit depth into the page).

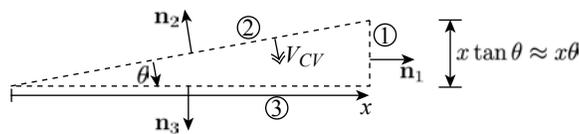
$$\begin{aligned} \left(\begin{array}{c} \text{time rate of change} \\ \text{of mass in the volume} \end{array} \right) &= \frac{\rho}{2} x(x \underbrace{\tan \Delta\theta}_{\approx \Delta\theta}) \frac{1}{\Delta t} = -\frac{1}{2} x^2 \omega \rho ; \\ \text{(rate of mass flow in)} &= 0 ; \\ \text{(rate of mass flow out)} &= u_{av}(x \underbrace{\tan \theta}_{\approx \theta}) \rho \end{aligned}$$

From (3.05a)

$$-\frac{1}{2} x^2 \omega = 0 - u_{av} \theta x \Rightarrow \boxed{u_{av} = \frac{x\omega}{2\theta}} .$$

More formally by conservation of mass:

$$\frac{\partial}{\partial t} \int_V \rho dV = - \int_s \rho(\mathbf{u} - \mathbf{V}_{CV}) \cdot \mathbf{n} ds . \tag{3.05b}$$



$$\begin{aligned} \int_V \rho dV &= \rho \int_0^x dx' \int_0^{x'\theta} dy' = \frac{1}{2} x(x\theta) \Rightarrow \frac{d}{dt} \int_V \rho dV = -\frac{1}{2} \rho x^2 \omega \\ \int_{\textcircled{3}} &= 0, \quad \int_{\textcircled{2}} = 0, \quad \text{b.c. } \mathbf{u} = \mathbf{V}_{CV} \text{ (no flux through a solid boundary)} \\ \int_{\textcircled{1}} &= - \int_{\theta}^{x\theta} \rho \underbrace{u}_{=\mathbf{u} \cdot \mathbf{n}} dy = \rho x \theta \underbrace{\frac{1}{x\theta} \int_0^{x\theta} u dy}_{u_{av}} \end{aligned}$$

Putting this together in (3.05b)

$$+\frac{1}{2} \rho x^2 \omega = +\rho x \theta u_{av} \Rightarrow \boxed{u_{av} = \frac{x\omega}{2\theta}} .$$



Problem Solution by Sungyon Lee, Fall 2005

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