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*Solutions Manual for Electromechanical Dynamics*

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PROBLEM 11.1

Part a

We add up all the volume force densities on the elastic material, and with the help of equation 11.1.4, we write Newton's law as

$$\rho \frac{\partial^2 \delta_1}{\partial t^2} = \frac{\partial T_{11}}{\partial x_1} - \rho g \quad (a)$$

where we have taken  $\frac{\partial}{\partial x_2} = \frac{\partial}{\partial x_3} = 0$ . Since this is a static problem, we let  $\frac{\partial}{\partial t} = 0$ . Thus,

$$\frac{\partial T_{11}}{\partial x_1} = \rho g. \quad (b)$$

From 11.2.32, we obtain

$$T_{11} = (2G + \lambda) \frac{\partial \delta_1}{\partial x_1} \quad (c)$$

Therefore

$$(2G + \lambda) \frac{\partial^2 \delta_1}{\partial x_1^2} = \rho g \quad (d)$$

Solving for  $\delta_1$ , we obtain

$$\delta_1 = \frac{\rho g}{2(2G+\lambda)} x_1^2 + C_1 x + C_2 \quad (e)$$

where  $C_1$  and  $C_2$  are arbitrary constants of integration, which can be evaluated by the boundary conditions

$$\delta_1(0) = 0 \quad (f)$$

and

$$T_{11}(L) = (2G + \lambda) \frac{\partial \delta_1}{\partial x_1}(L) = 0 \quad (g)$$

since  $x_1 = L$  is a free surface. Therefore, the solution is

$$\delta_1 = \frac{\rho g x_1}{2(2G+\lambda)} [x_1 - 2L]. \quad (f)$$

Part b

Again applying 11.2.32

INTRODUCTION TO THE ELECTROMECHANICS OF ELASTIC MEDIA

PROBLEM 11.1 (Continued)

$$\begin{aligned}
 T_{11} &= (2G+\lambda) \frac{\partial \delta_1}{\partial x_1} = \rho g [x_1 - L] \\
 T_{12} &= T_{21} = 0 \\
 T_{13} &= T_{31} = 0 \\
 T_{22} &= \lambda \frac{\partial \delta_1}{\partial x_1} = \frac{\lambda \rho g}{(2G+\lambda)} [x_1 - L] \\
 T_{33} &= \lambda \frac{\partial \delta_1}{\partial x_1} = \frac{\lambda \rho g}{(2G+\lambda)} [x_1 - L] \\
 T_{32} &= T_{23} = 0
 \end{aligned} \tag{g}$$

$$\bar{\bar{T}} = \begin{bmatrix} T_{11} & 0 & 0 \\ 0 & T_{22} & 0 \\ 0 & 0 & T_{33} \end{bmatrix} \tag{h}$$

PROBLEM 11.2

Since the electric force only acts on the surface at  $x_1 = -L$ , the equation of motion for the elastic material ( $-L \leq x_1 \leq 0$ ) is from Eqs. (11.1.4) and (11.2.32),

$$\rho \frac{\partial^2 \delta_1}{\partial t^2} = (2G+\lambda) \frac{\partial^2 \delta_1}{\partial x_1^2} \tag{a}$$

The boundary conditions are

$$\delta_1(0, t) = 0$$

and

$$M \frac{\partial^2 \delta_1(-L, t)}{\partial t^2} = aD(2G+\lambda) \frac{\partial \delta_1}{\partial x_1}(-L, t) + f^e \tag{b}$$

$f^e$  is the electric force in the  $x_1$  direction at  $x_1 = -L$ , and may be found by using the Maxwell Stress Tensor  $T_{ij} = \epsilon E_i E_j - \frac{1}{2} \delta_{ij} \epsilon E_k E_k$  to be (see Appendix G for discussion of stress tensor),

$$f^e = -\frac{\epsilon}{2} E^2 aD$$

with

$$E = \frac{V_0 + V_1 \cos \omega t}{d + \delta_1(-L, t)} \tag{c}$$

PROBLEM 11.2 (continued)

Expanding  $f^e$  to linear terms only, we obtain

$$f^e = -\frac{\epsilon a D}{2} \left[ \frac{v_o^2}{d^2} + \frac{2v_o v_1 \cos \omega t}{d^2} - \frac{2v_o^2}{d^3} \delta_1(-L, t) \right] \quad (d)$$

We have neglected all second order products of small quantities.

Because of the constant bias  $v_o$ , and the sinusoidal nature of the perturbations, we assume solutions of the form

$$\delta_1(x_1, t) = \delta_1(x_1) + \text{Re } \hat{\delta} e^{j(\omega t - kx_1)} \quad (e)$$

where

$$\hat{\delta} \ll \delta_1(x_1) \ll L$$

The relationship between  $\omega$  and  $k$  is readily found by substituting (e) into (a), from which we obtain

$$k = \pm \frac{\omega}{v_p} \text{ with } v_p = \sqrt{\frac{2G+\lambda}{\rho}} \quad (f)$$

We first solve for the equilibrium configuration which is time independent.

Thus

$$\frac{\partial^2 \delta_1(x_1)}{\partial x_1^2} = 0 \quad (g)$$

This implies

$$\delta_1(x_1) = C_1 x_1 + C_2$$

Because  $\delta_1(0) = 0$ ,  $C_2 = 0$ .

From the boundary condition at  $x_1 = -L$  ((b) & (d))

$$aD(2G+\lambda)C_1 - \frac{\epsilon}{2} \frac{v_o^2}{d^2} aD = 0 \quad (h)$$

Therefore

$$\delta_1(x_1) = + \frac{\epsilon}{2} \frac{v_o^2}{d^2(2G+\lambda)} x_1 \quad (i)$$

Note that  $\delta_1(x_1 = -L)$  is negative, as it should be.

For the time varying part of the solution, using (f) and the boundary condition

$$\delta(0, t) = 0$$

PROBLEM 11.2 (continued)

we can let the perturbation  $\delta_1$  be of the form

$$\delta_1(x_1, t) = \text{Re } \hat{\delta} \sin kx_1 e^{j\omega t} \quad (j)$$

Substituting this assumed solution into (b) and using (d), we obtain

$$+ M\omega^2 \hat{\delta} \sin kL = aD(2G+\lambda)k \hat{\delta} \cos kL - \frac{\epsilon a D V_o V_1}{d^2} - \frac{\epsilon a D V_o^2}{d^3} \hat{\delta} \sin kL \quad (k)$$

Solving for  $\hat{\delta}$ , we have

$$\hat{\delta} = - \frac{\epsilon a D V_o V_1}{d^2 \left[ M\omega^2 \sin kL - aD(2G+\lambda)k \cos kL + \frac{\epsilon a D V_o^2}{d^3} \sin kL \right]}$$

Thus, because  $\hat{\delta}$  has been shown to be real,

$$\delta_1(-L, t) = - \frac{\epsilon}{2} \frac{V_o^2 L}{d^2(2G+\lambda)} - \hat{\delta} \sin kL \cos \omega t \quad (m)$$

Part b

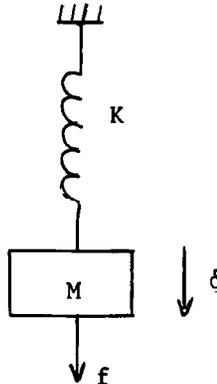
If  $kL \ll 1$ , we can approximate the sinusoidal part of (m) as

$$\delta_1(-L, t) = \frac{\epsilon a D V_o V_1 \cos \omega t}{d^2 \left[ M\omega^2 - \frac{aD(2G+\lambda)}{L} + \frac{\epsilon a D V_o^2}{d^3} \right]} \quad (n)$$

We recognize this as a force-displacement relation for a mass on the end of a spring.

Part c

We thus can model (n) as



PROBLEM 11.2 (Continued)

where

$$f = - \frac{\epsilon_0 a D V_0 V_1 \cos \omega t}{d^2}$$

and

$$K = \frac{aD(2G+\lambda)}{L} - \frac{\epsilon_0 a D V_0^2}{d^3}$$

We see that the electrical force acts like a negative spring constant.

PROBLEM 11.3

Part a

From (11.1.4), we have the equation of motion in the  $x_2$  direction as

$$\rho \frac{\partial^2 \delta_2}{\partial t^2} = \frac{\partial T_{21}}{\partial x_1} \quad (a)$$

From (11.2.32),

$$T_{21} = G \left[ \frac{\partial \delta_2}{\partial x_1} \right] \quad (b)$$

Therefore, substituting (b) into (a), we obtain an equation for  $\delta_2$

$$\rho \frac{\partial^2 \delta_2}{\partial t^2} = G \frac{\partial^2 \delta_2}{\partial x_1^2} \quad (c)$$

We assume solutions of the form

$$\delta_2 = \text{Re } \hat{\delta}_2 e^{j(\omega t - kx_1)} \quad (d)$$

where from (c) we obtain

$$k = \pm \frac{\omega}{v_p} \quad v_p^2 = \frac{G}{\rho}$$

Thus we let

$$\delta_2 = \text{Re} \left[ \delta_a e^{j(\omega t - kx_1)} + \delta_b e^{j(\omega t + kx_1)} \right] \quad (e)$$

$$\text{with } k = \frac{\omega}{v_p}$$

The boundary conditions are

$$\delta_2(l, t) = \delta_0 e^{j\omega t} \quad (f)$$

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PROBLEM 11.3 (continued)

and

$$\left. \frac{\partial \delta_2}{\partial x_1} \right|_{x_1=0} = 0 \quad (g)$$

since the surface at  $x_1 = 0$  is free.

Therefore

$$\delta_a e^{-jk\ell} + \delta_b e^{jk\ell} = \delta_o \quad (h)$$

and

$$-jk \delta_a + jk \delta_b = 0 \quad (i)$$

Solving, we obtain

$$\delta_a = \delta_b = \frac{\delta_o}{2 \cos k\ell} \quad (j)$$

Therefore

$$\delta_2(x_1, t) = \text{Re} \left[ \frac{\delta_o}{\cos k\ell} \cos kx_1 e^{j\omega t} \right] = \frac{\delta_o}{\cos k\ell} \cos kx_1 \cos \omega t \quad (k)$$

and

$$\begin{aligned} T_{21}(x_1, t) &= -\text{Re} \left[ \frac{G\delta_o k}{\cos k\ell} \sin kx_1 e^{j\omega t} \right] \\ &= -\frac{G\delta_o k}{\cos k\ell} \sin kx_1 \cos \omega t \end{aligned} \quad (l)$$

Part b

In the limit as  $\omega$  gets small

$$\delta_2(x_1, t) \rightarrow \text{Re}[\delta_o e^{j\omega t}] \quad (m)$$

In this limit,  $\delta_2$  varies everywhere in phase with the source. The slab of elastic material moves as a rigid body. Note from (l) that the force per unit area at  $x_1 = \ell$  required to set the slab into motion is  $T_{21}(\ell, t) = \rho \ell \frac{d^2}{dt^2}(\delta_o \cos \omega t)$  or the mass /  $(x_2 - x_3)$  area times the rigid body acceleration.

Part c

The slab can resonate if we can have a finite displacement, even as  $\delta_o \rightarrow 0$ . This can happen if the denominator of (k) vanishes

$$\cos k\ell = 0 \quad (n)$$

or

$$\omega = \frac{(2n+1)\pi v}{2\ell} \quad n = 0, 1, 2, \dots \quad (o)$$

PROBLEM 11.3 (continued)

The lowest frequency is for  $n = 0$

$$\text{or } \omega_{\text{low}} = \frac{\pi v}{2\ell} p \quad (p)$$

PROBLEM 11.4

Part a

We have that

$$\tau_i = T_{ij}n_j = \alpha\delta_{ij}n_j$$

It is given that the  $T_{ij}$  are known, thus the above equation may be written as three scalar equations  $(T_{ij} - \alpha\delta_{ij})n_j = 0$ , or:

$$\begin{aligned} (T_{11} - \alpha)n_1 + T_{12}n_2 + T_{13}n_3 &= 0 \\ T_{21}n_1 + (T_{22} - \alpha)n_2 + T_{23}n_3 &= 0 \\ T_{31}n_1 + T_{32}n_2 + (T_{33} - \alpha)n_3 &= 0 \end{aligned} \quad (a)$$

Part b

The solution for these homogeneous equations requires that the determinant of the coefficients of the  $n_i$ 's equal zero.

Thus

$$\begin{aligned} (T_{11} - \alpha)[(T_{22} - \alpha)(T_{33} - \alpha) - (T_{23})^2] \\ - T_{12}[T_{12}(T_{33} - \alpha) - T_{13}T_{23}] \\ + T_{13}[T_{12}T_{23} - T_{13}(T_{22} - \alpha)] = 0 \end{aligned} \quad (b)$$

where we have used the fact that

$$T_{ij} = T_{ji}. \quad (c)$$

Since the  $T_{ij}$  are known, this equation can be solved for  $\alpha$ .

Part c

Consider  $T_{12} = T_{21} = T_0$ , with all other components equal to zero. The determinant of coefficients then reduces to

$$-\alpha^3 + T_0^2\alpha = 0 \quad (d)$$

for which  $\alpha = 0 \quad (e)$

or  $\alpha = \pm T_0 \quad (f)$

The  $\alpha = 0$  solution indicates that with the normal in the  $x_3$  direction, there is no normal stress. The  $\alpha = \pm T_0$  solution implies that there are two surfaces where the net traction is purely normal with stresses  $\pm T_0$ , respectively, as

PROBLEM 11.4 (continued)

found in example 11.2.1. Note that the normal to the surface for which the shear stress is zero can be found from (a), since  $\alpha$  is known, and it is known that  $|\bar{n}| = 1$ .

PROBLEM 11.5

From Eqs. 11.2.25 - 11.2.28, we have

$$e_{11} = \frac{1}{E} [T_{11} - \nu(T_{22} + T_{33})] \quad (a)$$

$$e_{22} = \frac{1}{E} [T_{22} - \nu(T_{33} + T_{11})] \quad (b)$$

$$e_{33} = \frac{1}{E} [T_{33} - \nu(T_{11} + T_{22})] \quad (c)$$

and

$$e_{ij} = \frac{T_{ij}}{2G} \quad i \neq j \quad (d)$$

These relations must still hold in a primed coordinate system, where we can use the transformations

$$T'_{ij} = a_{ik} a_{jl} T_{kl} \quad (e)$$

and

$$e'_{ij} = a_{ik} a_{jl} e_{kl} \quad (f)$$

For an example, we look at  $e'_{11}$

$$e'_{11} = a_{1k} a_{1l} e_{kl} = \frac{1}{E} [T'_{11} - \nu(T'_{22} + T'_{33})] \quad (g)$$

This may be rewritten as

$$a_{1k} a_{1l} e_{kl} = \frac{1}{E} [(1 + \nu) a_{1k} a_{1l} T_{kl} - \nu \delta_{kl} T_{kl}] \quad (h)$$

where we have used the relation from Eq.(8.2.23), page G10 or 439.

$$a_{pr} a_{ps} = \delta_{ps} \quad (i)$$

Consider some values of  $k$  and  $l$  where  $k \neq l$ .

Then, from the stress-strain relation in the unprimed frame,

$$a_{1k} a_{1l} e_{kl} = a_{1k} a_{1l} \frac{T_{kl}}{2G} = \frac{a_{1k} a_{1l}}{E} (1 + \nu) T_{kl} \quad (j)$$

Thus

$$\frac{1}{2G} = \frac{1 + \nu}{E} \quad (k)$$

or  $E = 2G(1 + \nu)$  which agrees with Eq. (g) of example 11.2.1.

**PROBLEM 11.6**

**Part a**

Following the analysis in Eqs. 11.4.16 - 11.4.26, the equation of motion for the bar is

$$\frac{\partial^2 \xi}{\partial t^2} + \frac{Eb^2}{3\rho} \frac{\partial^4 \xi}{\partial x_1^4} = 0 \quad (a)$$

where  $\xi$  measures the bar displacement in the  $x_2$  direction,  $T_2$  in Eq. 11.4.26 = 0 as the surfaces at  $x_2 = \pm b$  are free. The boundary conditions for this problem are that at  $x_1 = 0$  and at  $x_1 = L$

$$T_{21} = 0 \quad \text{and} \quad T_{11} = 0 \quad (b)$$

as the ends are free.

We assume solutions of the form

$$\xi = \text{Re } \hat{\xi}(x) e^{j\omega t} \quad (c)$$

As in example 11.4.4, the solutions for  $\hat{\xi}(x)$  are

$$\hat{\xi}(x) = A \sin \alpha x_1 + B \cos \alpha x_1 + C \sinh \alpha x_1 + D \cosh \alpha x_1 \quad (d)$$

with

$$\alpha = \left[ \omega^2 \left( \frac{3\rho}{Eb^2} \right) \right]^{1/4}$$

Now, from Eqs. 11.4.18 and 11.4.21,

$$T_{21} = \frac{(x_2^2 - b^2)E}{2} \frac{\partial^3 \xi}{\partial x_1^3} \quad (e)$$

which implies

$$\frac{\partial^3 \xi}{\partial x_1^3} = 0 \quad (f)$$

$$\text{at } x_1 = 0, x_1 = L$$

and

$$T_{11} = -x_2 E \frac{\partial^2 \xi}{\partial x_1^2} \quad (g)$$

which implies

$$\frac{\partial^2 \xi}{\partial x_1^2} = 0 \quad (h)$$

$$\text{at } x_1 = 0 \text{ and } x_1 = L$$

PROBLEM 11.6 (continued)

With these relations, the boundary conditions require that

$$\begin{aligned}
 - A & & + C & & & = 0 \\
 - A \cos \alpha L + B \sin \alpha L + C \cosh \alpha L + D \sinh \alpha L & = 0 \\
 - B & & + D & & & = 0 \\
 - A \sin \alpha L - B \cos \alpha L + C \sinh \alpha L + D \cosh \alpha L & = 0
 \end{aligned} \tag{i}$$

The solution to this set of homogeneous equations requires that the determinant of the coefficients of A, B, C, and D equal zero. Performing this operation, we obtain

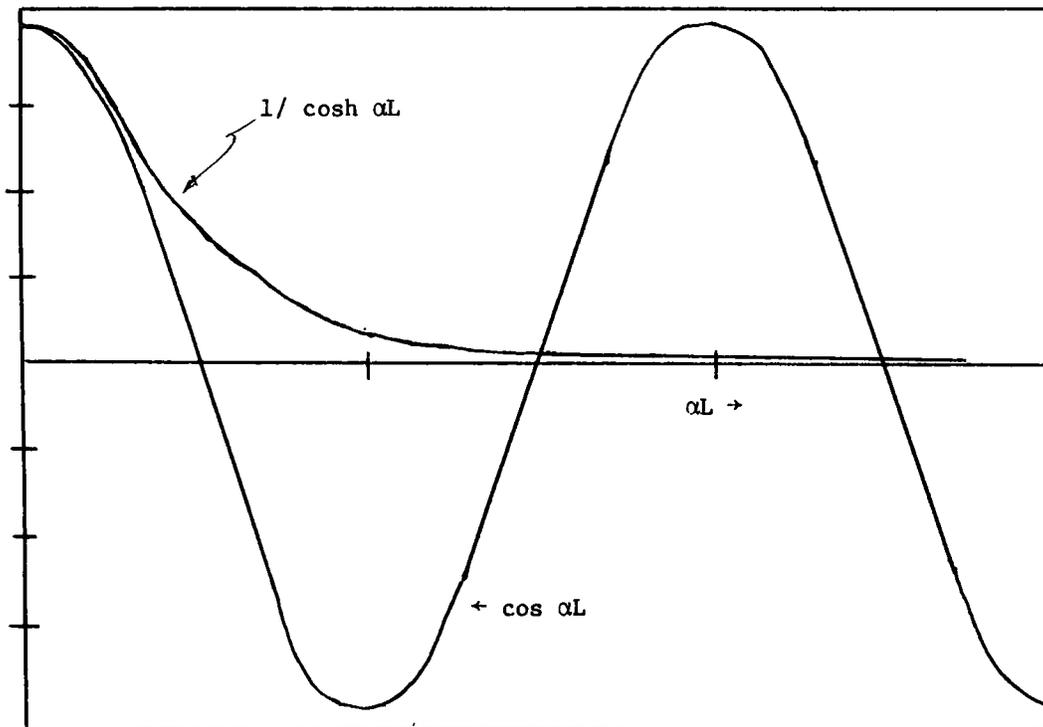
$$\cos \alpha L \cosh \alpha L = 1 \tag{j}$$

Thus,

$$\beta = \alpha L = \left[ \omega^2 \left( \frac{3\rho}{Eb^2} \right) \right]^{1/4} L \tag{k}$$

Part b

The roots of  $\cos \beta = \frac{1}{\cosh \beta}$  follow from the figure.



Note from the figure that the roots  $\alpha L$  are essentially the roots  $3\pi/2, 5\pi/2, \dots$  of  $\cos \alpha L = 0$ .

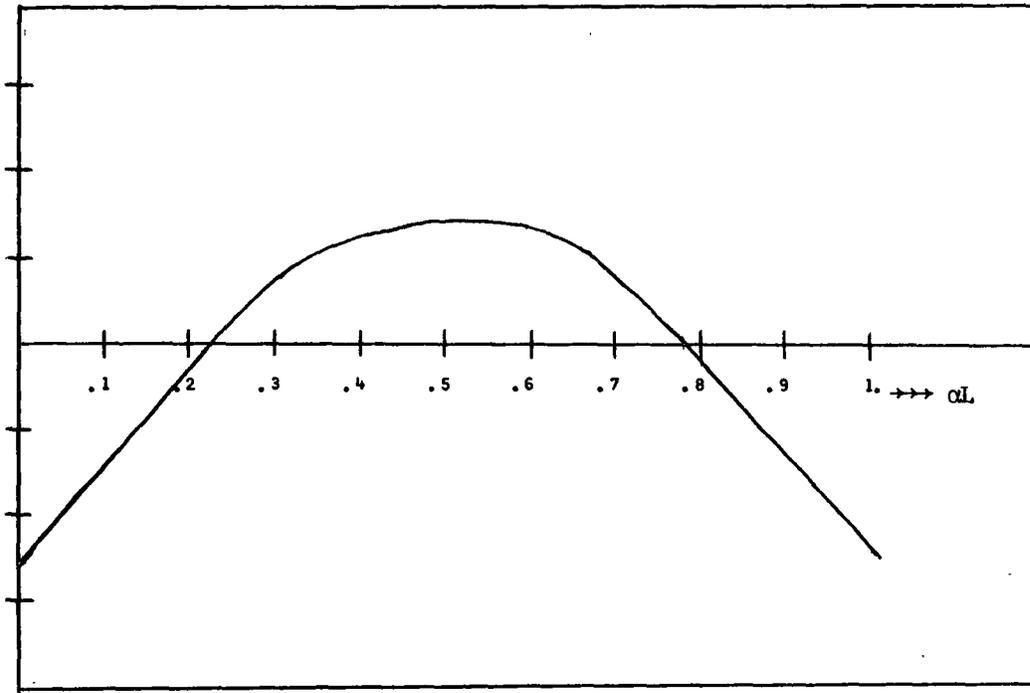
PROBLEM 11.6 (continued)

Part c

It follows from (i) that the eigenfunction is

$$\hat{\xi} = A'[(\sin \alpha x_1 + \sinh \alpha x_1)(\sin \alpha L + \sinh \alpha L) + (\cos \alpha L - \cosh \alpha L)(\cos \alpha x_1 + \cosh \alpha x_1)] \quad (2)$$

where  $A'$  is an arbitrary amplitude. This expression is found by taking one of the constants  $A \dots D$  as known, and solving for the others. Then, (d) gives the required dependence on  $x_1$  to within an arbitrary constant. A sketch of this function is shown in the figure.



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PROBLEM 11.7

As in problem 11.6, the equation of motion for the elastic beam is

$$\frac{\partial^2 \xi}{\partial t^2} + \frac{Eb^2}{3\rho} \frac{\partial^4 \xi}{\partial x_1^4} = 0 \quad (a)$$

The four boundary conditions for this problem are:

$$\xi(x_1 = 0) = 0 \quad \xi(x_1 = L) = 0$$

$$\delta_1(0) = -x_2 \left. \frac{\partial \xi}{\partial x_1} \right|_{x_1=0} = 0 \quad \delta_1(L) = -x_2 \left. \frac{\partial \xi}{\partial x_1} \right|_{x_1=L} = 0 \quad (b)$$

We assume solutions of the form

$\xi(x_1, t) = \text{Re } \hat{\xi}(x_1) e^{j\omega t}$ , and as in problem 11.6, the solutions for  $\hat{\xi}(x_1)$  are

$$\hat{\xi}(x_1) = A \sin \alpha x_1 + B \cos \alpha x_1 + C \sinh \alpha x_1 + D \cosh \alpha x_1$$

$$\text{with } \alpha = \left[ \omega^2 \left( \frac{3\rho}{Eb^2} \right) \right]^{1/4} \quad (d)$$

Applying the boundary conditions, we obtain

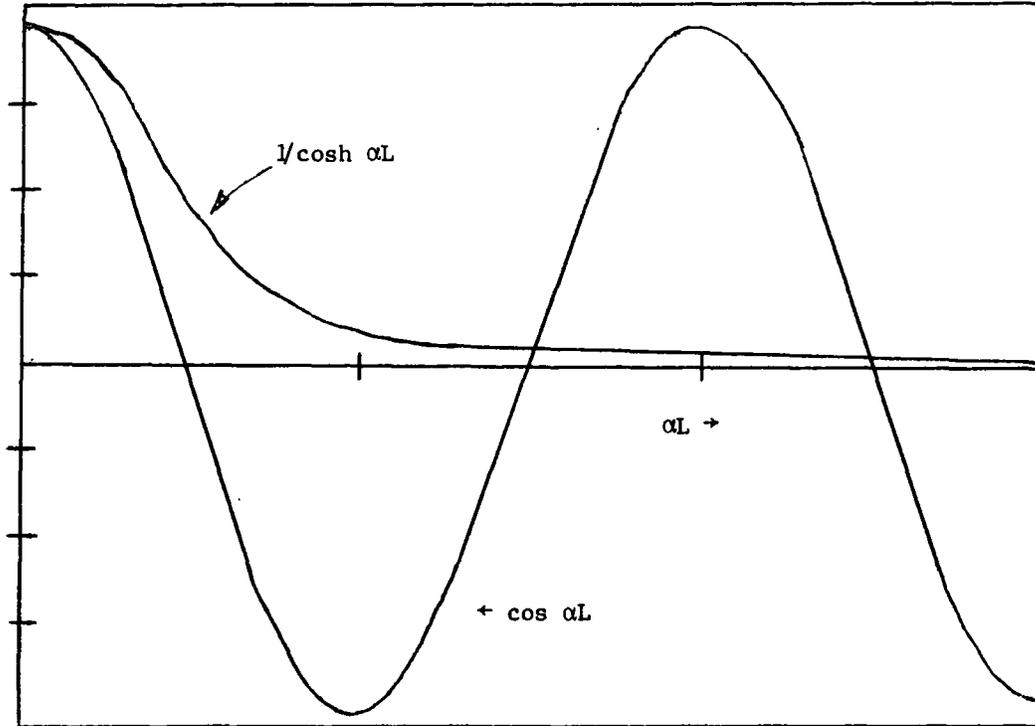
$$\begin{aligned} B + D &= 0 \\ A \sin \alpha L + B \cos \alpha L + C \sinh \alpha L + D \cosh \alpha L &= 0 \\ A + C &= 0 \\ A \cos \alpha L - B \sin \alpha L + C \cosh \alpha L + D \sinh \alpha L &= 0 \end{aligned} \quad (e)$$

The solution to this set of homogeneous equations requires that the determinant of the coefficients of A, B, C, D, equal zero. Performing this operation, we obtain

$$\cos \alpha L \cosh \alpha L = +1 \quad (f)$$

To solve for the natural frequencies, we must use a graphical procedure.

PROBLEM 11.7 (continued)



The first natural frequency is at about

$$\alpha L = \frac{3\pi}{2}$$

Thus

$$\omega^2 \left( \frac{3\rho}{Eb^2} \right) L^4 = \left( \frac{3\pi}{2} \right)^4$$

or

$$\omega = \frac{\left( \frac{3\pi}{2} \right)^2}{L^2} \left( \frac{Eb^2}{3\rho} \right)^{1/2} \quad (g)$$

Part b

We are given that  $L = .5 \text{ m}$  and  $b = 5 \times 10^{-4} \text{ m}$

From Table 9.1, Appendix G, the parameters for steel are:

$$E \approx 2 \times 10^{11} \text{ N/m}^2$$

$$\rho \approx 7.75 \times 10^3 \text{ kg/m}^3$$

PROBLEM 11.7 (continued)

$$\omega \approx 120 \text{ rad/sec.}$$

$$\text{Then, } f_1 = \frac{\omega}{2\pi} \approx 19 \text{ Hz.}$$

Part c

$$\text{For the next higher resonance, } \alpha L \approx \frac{5}{2} \pi$$

$$\text{Therefore, } f_2 = \left(\frac{5}{2}\right)^2 f_1 \approx 53 \text{ Hz.}$$

PROBLEM 11.8

Part a

As in Prob. 11.7, the equation of motion for the beam is

$$\frac{\partial^2 \xi}{\partial t^2} + \frac{Eb^2}{3\rho} \frac{\partial^4 \xi}{\partial x_1^4} = 0 \quad (a)$$

At  $x_1 = L$ , there is a free end, so the boundary conditions are:

$$T_{11}(x_1=L) = 0$$

$$\text{and } T_{21}(x_1=L) = 0 \quad (b)$$

The boundary conditions at  $x_1 = 0$  are

$$M \frac{\partial^2 \xi(0,t)}{\partial t^2} = + \int_{x_1=0}^L (T_{21})_D dx_2 + \bar{f}_e + \bar{F}_o \quad (c)$$

and

$$\delta_1(x_1 = 0) = 0 \quad (d)$$

The  $\bar{H}$  field in the air gap and in the plunger is

$$\bar{H} = \frac{Ni}{D} \bar{i}_1 \quad (e)$$

Using the Maxwell stress tensor

$$\bar{f}_e = - \frac{(\mu - \mu_o)}{2} \left( \frac{N^2 i^2}{D^2} \right) D^2 \bar{i}_2 = - \frac{N^2 i^2}{2} (\mu - \mu_o) \bar{i}_2 \quad (f)$$

$$\text{with } i = I_o + i_1 \cos \omega t = I_o + \text{Re } i_1 e^{j\omega t}$$

PROBLEM 11.8 (continued)

We linearize  $\bar{f}^e$  to obtain

$$\bar{f}^e = -\frac{N^2}{2} (\mu - \mu_0) [I_0^2 + 2I_0 i_1 \cos \omega t] \bar{i}_2 \quad (g)$$

For equilibrium

$$\bar{F}_0 - \frac{N^2}{2} (\mu - \mu_0) I_0^2 \bar{i}_2 = 0$$

Thus 
$$\bar{F}_0 = \frac{N^2}{2} (\mu - \mu_0) I_0^2 \bar{i}_2 \quad (h)$$

Part b

We write the solution to Eq. (a) in the form

$$\xi(x_1, t) = \text{Re } \hat{\xi}(x_1) e^{j\omega t}$$

where, from example 11.4.4

$$\hat{\xi}(x_1) = A_1 \sin \alpha x_1 + A_2 \cos \alpha x_1 + A_3 \sinh \alpha x_1 + A_4 \cosh \alpha x_1 \quad (i)$$

with

$$\alpha = \left[ \omega^2 \left( \frac{3\rho}{Eb^2} \right) \right]^{1/4}$$

Now, from Eqs. 11.4.6 and 11.4.16

$$T_{11}(x=L) = E \frac{\partial \delta}{\partial x_1} = -E x_2 \frac{\partial^2 \xi}{\partial x_1^2} = 0 \quad (j)$$

Thus 
$$\frac{\partial^2 \xi}{\partial x_1^2} (x_1 = L) = 0$$

From Eq. 11.4.21

$$T_{21} = \frac{(x_2^2 - b^2)}{2} E \frac{\partial^3 \xi}{\partial x_1^3} \quad (k)$$

and from Eq. 11.4.16

$$\delta_1(x_1=0) = -x_2 \left( \frac{\partial \xi}{\partial x_1} \right)_{x_1=0} = 0 \quad (l)$$

Thus 
$$\left( \frac{\partial \xi}{\partial x_1} \right)_{x_1=0} = 0$$

PROBLEM 11.8(continued)

Applying the boundary conditions from Eqs. (b), (c), (d) to our solution of Eq. (i), we obtain the four equations

$$\begin{aligned}
 A_1 &+ A_3 &= 0 \\
 -A_1 \sin \alpha L &- A_2 \cos \alpha L + A_3 \sinh \alpha L + A_4 \cosh \alpha L &= 0 \\
 -A_1 \cos \alpha L &+ A_2 \sin \alpha L + A_3 \cosh \alpha L + A_4 \sinh \alpha L &= 0 \quad (m) \\
 -\frac{2}{3} \alpha^3 b^3 EDA_1 &+ M \omega^2 A_2 + \frac{2}{3} \alpha^3 b^3 EDA_3 + M \omega^2 A_4 &= + N^2 I_o i_1 (\mu - \mu_o)
 \end{aligned}$$

Now

$$v = \frac{d\lambda}{dt} = \frac{d}{dt} \left\{ \frac{N^2 i}{D} D \left[ \mu_o \xi(0) + \mu(D - \xi(0)) \right] \right\} \quad (n)$$

$$\text{or } \hat{v} = -N^2 I_o (\mu - \mu_o) j\omega (A_2 + A_4) + N^2 i_1 \mu D j\omega \quad (o)$$

We solve Eqs. (m) for  $A_2$  and  $A_4$  using Cramer's rule to obtain

$$A_2 = \frac{N^2 I_o i_1 (\mu - \mu_o) (-1 + \sin \alpha L \sinh \alpha L - \cos \alpha L \cosh \alpha L)}{-2M\omega^2 (1 + \cos \alpha L \cosh \alpha L) + \frac{4}{3} (\alpha b)^3 ED (\cos \alpha L \sinh \alpha L + \sin \alpha L \cosh \alpha L)} \quad (p)$$

and

$$A_4 = \frac{N^2 I_o i_1 (\mu - \mu_o) (-1 - \cos \alpha L \cosh \alpha L - \sin \alpha L \sinh \alpha L)}{-2M\omega^2 (1 + \cos \alpha L \cosh \alpha L) + \frac{4}{3} (\alpha b)^3 ED (\cos \alpha L \sinh \alpha L + \sin \alpha L \cosh \alpha L)} \quad (q)$$

Thus

$$\begin{aligned}
 Z(j\omega) = \frac{\hat{v}(j\omega)}{i_1} &= \frac{+ [N^2 I_o (\mu - \mu_o)]^2 j\omega (+2 + 2 \cos \alpha L \cosh \alpha L)}{-2M\omega^2 (1 + \cos \alpha L \cosh \alpha L) + \frac{4}{3} (\alpha b)^3 ED (\cos \alpha L \sinh \alpha L + \sin \alpha L \cosh \alpha L)} \\
 &+ N^2 \mu D j\omega \quad (r)
 \end{aligned}$$

Part c

$Z(j\omega)$  has poles when

$$+2M\omega^2 (1 + \cos \alpha L \cosh \alpha L) = \frac{4}{3} (\alpha b)^3 ED (\cos \alpha L \sinh \alpha L + \sin \alpha L \cosh \alpha L)$$

PROBLEM 11.9

Part a

The flux above and below the beam must remain constant. Therefore, the  $\bar{H}$  field above is

$$\bar{H}_a = \frac{H_o (a-b)}{(a-b-\xi)} \bar{I}_1 \quad (a)$$

and the  $\bar{H}$  field below is

$$\bar{H}_b = \frac{H_o (a-b)}{(a-b+\xi)} \bar{I}_1 \quad (b)$$

Using the Maxwell stress tensor, the magnetic force on the beam is

$$\begin{aligned} T_2 &= -\frac{\mu_o}{2} (H_a^2 - H_b^2) = -\frac{\mu_o}{2} H_o^2 (a-b)^2 \left( + \frac{4\xi}{(a-b)^3} \right) \\ &= -\frac{2\mu_o H_o^2 \xi}{(a-b)} \end{aligned} \quad (c)$$

Thus, from Eq. 11.4.26, the equation of motion on the beam is

$$\frac{\partial^2 \xi}{\partial t^2} + \frac{Eb^2}{3\rho} \frac{\partial^4 \xi}{\partial x_1^4} = -\frac{\mu_o H_o^2 \xi}{(a-b)b\rho} \quad (d)$$

Again, we let

$$\xi(x_1, t) = \text{Re } \hat{\xi}(x_1) e^{j\omega t} \quad (e)$$

with the boundary conditions

$$\begin{aligned} \xi(x_1=0) &= 0 & \xi(x_1=L) &= 0 \\ \delta_1(x_1=0) & & \delta_1(x_1=L) &= 0 \end{aligned} \quad (f)$$

Since  $\delta_1 = -x_2 \partial \xi / \partial x_1$  from Eq. 11.4.16, this implies that:

$$\frac{\partial \xi}{\partial x_1} (x_1=0) = 0 \text{ and } \frac{\partial \xi}{\partial x_1} (x_1=L) = 0 \quad (g)$$

Substituting our assumed solution into the equation of motion, we have

$$-\omega^2 \hat{\xi} + \frac{Eb^2}{3\rho} \frac{\partial^4 \hat{\xi}}{\partial x_1^4} + \frac{\mu_o H_o^2 \hat{\xi}}{(a-b)b\rho} = 0 \quad (h)$$

Thus we see that our solutions are again of the form

$$\hat{\xi}(x) = A \sin \alpha x + B \cos \alpha x + C \sinh \alpha x + D \cosh \alpha x \quad (i)$$

PROBLEM 11.9 (continued)

where now

$$\alpha = \left[ \left( \omega^2 - \frac{\mu_o H_o^2}{(a-b)b\rho} \right) \left( \frac{3\rho}{Eb^2} \right) \right]^{1/4} \quad (j)$$

Since the boundary conditions for this problem are identical to that of problem 11.7, we can take the solutions from that problem, substituting the new value of  $\alpha$ . From problem 11.7, the solution must satisfy

$$\cos \alpha L \cosh \alpha L = 1 \quad (k)$$

The first resonance occurs when

$$\begin{aligned} \alpha L &\approx \frac{3\pi}{2} \\ \text{or} \quad \omega^2 &= \frac{\left(\frac{3\pi}{2}\right)^4 \left(\frac{Eb^2}{3\rho}\right)}{L^4} + \frac{\mu_o H_o^2}{(a-b)b\rho} \end{aligned} \quad (l)$$

Part c

The resonant frequencies are thus shifted upward due to the stiffening effect of the constant flux constraint.

Part d

We see that, no matter what the values of the system parameters  $\omega^2 > 0$ , so  $\omega$  will always be real, and thus stable. This is expected as the constant flux constraint imposes a force which opposes the motion.

PROBLEM 11.10

Part a

We choose a coordinate system as in Fig. 11.4.12, centered at the right end of the rod. Because  $\frac{d}{D} = \frac{1}{10}$ , we can neglect fringing and consider the right end of the rod as a capacitor plate. Also, since  $\frac{D}{\ell} = \frac{1}{10}$ , we can assume that the electrical force acts only at  $x_1 = 0$ . Thus, the boundary conditions at  $x_1 = 0$  are

$$- \int_0^b T_{21} D dx_2 + f^e = 0 \quad (a)$$

$$\text{where } T_{21} = \frac{(x_2^2 - b^2)}{2} E \frac{\partial^3 \xi}{\partial x_1^3} \quad (\text{Eq. 11.4.21})$$

since the electrical force,  $f^e$ , must balance the shear stress  $T_{21}$  to keep the rod in equilibrium,

**PROBLEM 11.10 (continued)**

and

$$T_{11}(0) = -x_2 E \frac{\partial^2 \xi}{\partial x_1^2}(0) = 0 \quad (b)$$

since the end of the rod is free of normal stresses. At  $x_1 = -l$ , the rod is clamped so

$$\xi(-l) = 0 \quad (c)$$

and

$$\delta_1(-l) = -x_2 \frac{\partial \xi}{\partial x_1}(-l) = 0 \quad (d)$$

We use the Maxwell stress tensor to calculate the electrical force to be

$$f^e = \frac{\epsilon A}{2} \left[ \frac{(v_o + v_s)^2}{[d - \xi(0)]^2} - \frac{(v_s - v_o)^2}{[d + \xi(0)]^2} \right] \quad (e)$$

$$\approx \frac{2\epsilon AV_o}{d^2} \left[ v_s + \frac{v_o \xi(0)}{d} \right]$$

The equation of motion of the beam is (example 11.4.4)

$$\frac{\partial^2 \xi}{\partial t^2} + \frac{Eb^2}{3\rho} \frac{\partial^4 \xi}{\partial x_1^4} = 0 \quad (f)$$

We write the solution to Eq. (f) in the form

$$\xi(x,t) = \text{Re } \hat{\xi}(x) e^{j\omega t} \quad (g)$$

where

$$\hat{\xi}(x) = A_1 \sin \alpha x + A_2 \cos \alpha x + A_3 \sinh \alpha x + A_4 \cosh \alpha x$$

with

$$\alpha = \left[ \omega^2 \left( \frac{3\rho}{Eb^2} \right) \right]^{1/4}$$

Applying the four boundary conditions, Eqs. (a), (b), (c) and (d), we obtain the equations

$$\begin{aligned} -A_1 \sin \alpha l + A_2 \cos \alpha l - A_3 \sinh \alpha l + A_4 \cosh \alpha l &= 0 \\ A_1 \cos \alpha l + A_2 \sin \alpha l + A_3 \cosh \alpha l - A_4 \sinh \alpha l &= 0 \quad (h) \\ -A_2 &+ A_4 = 0 \\ -\frac{2}{3} b^3 DE \alpha^3 A_1 + \frac{2\epsilon_o AV_o^2}{d^3} A_2 + \frac{2}{3} b^3 DE \alpha^3 A_3 + \frac{2\epsilon_o AV_o^2}{d^3} A_4 &= -\frac{2\epsilon_o AV_o \hat{v}_s}{d^2} \end{aligned}$$

PROBLEM 11.10 (continued)

Now  $i_s = \frac{dq_s}{dt}$  (i)

where  $q_s = \frac{\epsilon_o A}{d - \xi(0)} (v_o + v_s) + \frac{\epsilon_o A (v_s - v_o)}{d + \xi(0)}$  (j)

$$\approx \frac{2\epsilon_o A v_s}{d} + \frac{2\epsilon_o A v_o}{d^2} \xi(0)$$

Therefore

$$\hat{i}_s = j\omega \frac{2\epsilon_o A}{d} \left[ \hat{v}_s + \frac{v_o}{d} \hat{\xi}(0) \right] \quad (k)$$

where

$$\hat{\xi}(0) = A_2 + A_4$$

We use Cramer's rule to solve Eqs. (h) for  $A_2$  and  $A_4$  to obtain:

$$A_2 = A_4 = \frac{-\frac{\epsilon_o A v_o \hat{v}_s}{d^2} [\cos \alpha l \sinh \alpha l - \sin \alpha l \cosh \alpha l]}{\frac{2}{3} b^3 \alpha^3 DE (1 + \cos \alpha l \cosh \alpha l) + \frac{2\epsilon_o A v_o^2}{d^3} (\cos \alpha l \sinh \alpha l - \sin \alpha l \cosh \alpha l)} \quad (l)$$

Thus, from Eq. (k) we obtain

$$Z(j\omega) = \frac{d}{j\omega 2\epsilon_o A} \left[ 1 + \frac{3\epsilon_o A v_o^2}{d^3 (\alpha b)^3 ED} \frac{(\cos \alpha l \sinh \alpha l - \sin \alpha l \cosh \alpha l)}{(1 + \cos \alpha l \cosh \alpha l)} \right] \quad (m)$$

Part b

We define a function  $g(\alpha l)$  such that Eq. (m) has a zero when

PROBLEM 11.10 (Continued)

$$(\alpha L)^3 g(\alpha L) = \frac{(1 + \cosh \alpha l \cos \alpha l)(\alpha l)^3}{\sin \alpha l \cosh \alpha l - \cos \alpha l \sinh \alpha l} = \frac{3l^3 V_o^2 A \epsilon_o}{DEb^3 d^3} \quad (n)$$

Substituting numerical values, we obtain

$$\frac{3l^3 V_o^2 A \epsilon_o}{DEb^3 d^3} \approx \frac{3 \times 10^{-3} (10^6) 10^{-4} (8.85 \times 10^{-12})}{10^{-2} (2.2 \times 10^{11}) 10^{-9} 10^{-9}} \approx 1.2 \times 10^{-3} \quad (o)$$

In Figure 1, we plot  $(\alpha l)^3 g(\alpha l)$  as a function of  $\alpha l$ . We see that the solution to Eq. (n) first occurs when  $(\alpha l)^3 g(\alpha l) \approx 0$ . Thus, the solution is approximately

$$\alpha l = 1.875$$

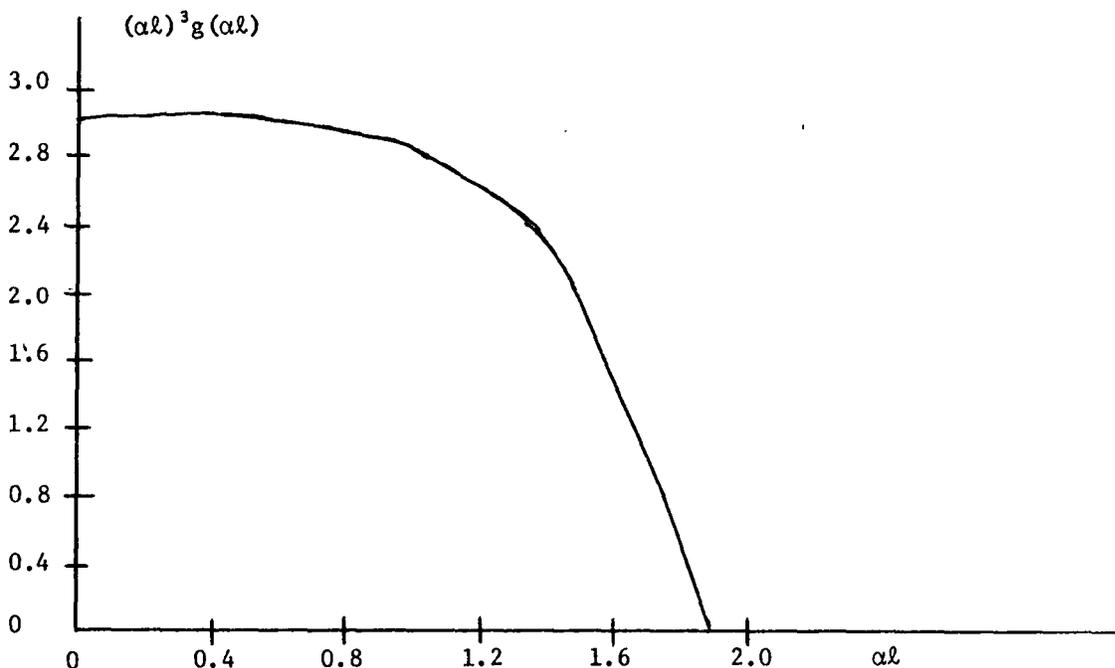


Figure 1

PROBLEM 11.10 (Continued)

From Eq. (g)

$$\alpha l = \left[ \omega^2 \frac{3\rho}{Eb^2} \right]^{1/4} \quad l = 1.875$$

Solving for  $\omega$ , we obtain

$$\omega \approx 1080 \text{ rad/sec.} \quad (p)$$

Part c

The input impedance of a series LC circuit is

$$Z(j\omega) = \frac{1 - LC\omega^2}{j\omega C} \quad (q)$$

Thus the impedance has a zero when

$$\omega_o^2 = \frac{1}{LC} \quad (r)$$

We let  $\omega = \omega_o + \Delta\omega$ , and expand (q) in a Taylor series around  $\omega_o$  to obtain

$$Z(j\omega) \approx + j \frac{2L\Delta\omega}{C\omega_o^2} = + 2j L\Delta\omega \quad (s)$$

(m) can be written in the form

$$Z(j\omega) = \frac{1}{2j\omega C_o} [1 - f(\omega)] \quad \text{where } f(\omega_o) = 1 \quad (t)$$

$$\text{and } C_o = \frac{\epsilon_o A}{d}$$

For small deviations around  $\omega_o$

$$Z(j\omega) \approx \frac{j}{2\omega C_o} \left. \frac{\partial f}{\partial \omega} \right|_{\omega_o} \Delta\omega$$

Thus, from (q), (r) (s) and (t), we obtain the relations

$$2L = \frac{1}{2\omega C_o} \left. \frac{\partial f}{\partial \omega} \right|_{\omega_o} \quad (u)$$

and  $C = \frac{1}{\omega_o^2 L} \quad (v)$

now  $f(\omega) = \frac{K}{(\alpha l)^3 g(\alpha l)} \quad (w)$

where  $K = \frac{3l^3 \epsilon_o AV_o^2}{d^3 (EDb^3)} = 1.2 \times 10^{-3}$

PROBLEM 11.10 (Continued)

$$\text{and } g(\alpha l) = \frac{1 + \cos \alpha l \cosh \alpha l}{\sin \alpha l \cosh \alpha l - \cos \alpha l \sinh \alpha l}$$

Thus, we can write

$$\left. \frac{df(\omega)}{d\omega} \right|_{\omega_0} = \left\{ \frac{d}{d(\alpha l)} \left[ \frac{K}{(\alpha l)^3 g(\alpha l)} \right] \frac{d(\alpha l)}{d\omega} \right\} \Big|_{\omega_0} \quad (y)$$

Now from (g),

$$\left. \frac{d(\alpha l)}{d\omega} \right|_{\omega_0} = \left( \frac{3\rho}{Eb^2} \right)^{1/4} \frac{l}{2\omega_0^{1/2}} \quad (z)$$

and

$$\begin{aligned} \left. \frac{d}{d(\alpha l)} \left[ \frac{K}{(\alpha l)^3 g(\alpha l)} \right] \right|_{\omega_0} &= \frac{-K}{[(\alpha l)^3 g(\alpha l)]^2} \left. \frac{d}{d(\alpha l)} [(\alpha l)^3 g(\alpha l)] \right|_{\omega_0} \\ &\approx -\frac{1}{K} \left. \frac{d}{d(\alpha l)} [(\alpha l)^3 g(\alpha l)] \right|_{\omega_0} \end{aligned} \quad (aa)$$

since at  $\omega = \omega_0$

$$(\alpha l)^3 g(\alpha l) = K. \quad (bb)$$

Continuing the differentiating in (aa), we finally obtain

$$\begin{aligned} \left. \frac{d}{d(\alpha l)} \left[ \frac{(\alpha l)^3 g(\alpha l)}{-K} \right] \right|_{\omega_0} &= -\frac{1}{K} \left[ g(\alpha l) 3(\alpha l)^2 + (\alpha l)^3 \frac{d}{d(\alpha l)} g(\alpha l) \right] \Big|_{\omega_0} \\ &= \frac{-3}{\alpha l} \Big|_{\omega_0} - \frac{(\alpha l)^3}{K} \left. \frac{d}{d(\alpha l)} g(\alpha l) \right|_{\omega_0} \end{aligned} \quad (cd)$$

Now

$$\frac{d}{d(\alpha l)} g(\alpha l) = \frac{-\sin \alpha l \cosh \alpha l + \cos \alpha l \sinh \alpha l}{(\sin \alpha l \cosh \alpha l - \cos \alpha l \sinh \alpha l)}$$

$$= \frac{-(1 + \cos \alpha l \cosh \alpha l) + (\cos \alpha l \cosh \alpha l + \sin \alpha l \sinh \alpha l + \sin \alpha l \sinh \alpha l - \cos \alpha l \cosh \alpha l)}{(\sin \alpha l \cosh \alpha l - \cos \alpha l \sinh \alpha l)}$$

$$= -1 - \frac{2g(\alpha l) (\sin \alpha l \sinh \alpha l)}{(\sin \alpha l \cosh \alpha l - \cos \alpha l \sinh \alpha l)} \quad (dd)$$

PROBLEM 11.10 (Continued)

Substituting numerical values into the second term of (cc), we find it to have value much less than one at  $\omega = \omega_0$ .

Thus,

$$\frac{d}{d(\alpha l)} g(\alpha l) \approx -1 \quad (\text{ee})$$

Thus, using (y), (z), (aa) (bb) and (dd), we have

$$\left. \frac{df}{d\omega} \right|_{\omega_0} \approx \left( \frac{3\rho}{Eb^2} \right)^{1/4} \frac{l}{2\omega_0^{1/2}} \left[ - \left. \frac{3}{\alpha l} \right|_{\omega_0} + \left. \frac{(\alpha l)^3}{K} \right|_{\omega_0} \right] \approx 4.8 \quad (\text{ff})$$

Thus, from (v) and (w)

$$L \approx \frac{4.8 \times 10^{-3}}{4(1080)(8.85 \times 10^{-12})(10^{-4})} = 1.25 \times 10^9 \text{ henries}$$

and

$$C \approx \frac{1}{1.25 \times 10^9 (1080)^2} = 6.8 \times 10^{-16} \text{ farads.}$$

PROBLEM 11.11

From Eq. (11.4.29), the equation of motion is

$$\rho \frac{\partial^2 \delta_3}{\partial t^2} = G \left( \frac{\partial^2 \delta_3}{\partial x_1^2} + \frac{\partial^2 \delta_3}{\partial x_2^2} \right) \quad (a)$$

We let

$$\delta_3 = \text{Re } \hat{\delta}(x_2) e^{j(\omega t - kx_1)} \quad (b)$$

Substituting this assumed solution into the equation of motion, we obtain

$$-\rho \omega^2 \hat{\delta} = G \left( -k^2 \hat{\delta} + \frac{\partial^2 \hat{\delta}}{\partial x_2^2} \right) \quad (c)$$

or

$$\frac{\partial^2 \hat{\delta}}{\partial x_2^2} + \left( \frac{\rho \omega^2}{G} - k^2 \right) \hat{\delta} = 0 \quad (d)$$

$$\text{If we let } \beta^2 = \frac{\rho \omega^2}{G} - k^2 \quad (e)$$

the solutions for  $\hat{\delta}$  are:

$$\hat{\delta}(x_2) = A \sin \beta x_2 + B \cos \beta x_2 \quad (f)$$

The boundary conditions are

$$\hat{\delta}(0) = 0 \quad \text{and} \quad \hat{\delta}(d) = 0 \quad (g)$$

This implies that  $B = 0$

and that  $\beta d = n\pi$ .

Thus, the dispersion relation is

$$\omega^2 \frac{\rho}{G} - k^2 = \left( \frac{n\pi}{d} \right)^2 \quad (h)$$

Part b

The sketch of the dispersion relation is identical to that of Fig. 11.4.19. However, now the  $n=0$  solution is trivial, as it implies that

$$\hat{\delta}(x_2) = 0$$

Thus, there is no principal mode of propagation.

PROBLEM 11.12

From Eq. (11.4.1), the equation of motion is

$$\rho \frac{\partial^2 \delta}{\partial t^2} = (2G + \lambda) \nabla(\nabla \cdot \delta) - G \nabla \times (\nabla \times \delta) \quad (a)$$

We consider motions

$$\delta = \delta_{\theta}(r, z, t) \bar{i}_{\theta} \quad (b)$$

Thus, the equation of motion reduces to

$$\rho \frac{\partial^2 \delta_{\theta}}{\partial t^2} - G \left[ \frac{\partial^2 \delta_{\theta}}{\partial z^2} + \frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial}{\partial r} r \delta_{\theta} \right) \right] = 0 \quad (c)$$

We assume solutions of the form

$$\delta_{\theta}(r, z, t) = \text{Re } \hat{\delta}(r) e^{j(\omega t - kz)} \quad (d)$$

which, when substituted into the equation of motion, yields

$$\frac{\partial}{\partial r} \left[ \frac{1}{r} \frac{\partial}{\partial r} r \hat{\delta}(r) \right] + \left( \frac{\rho \omega^2}{G} - k^2 \right) \hat{\delta}(r) = 0 \quad (e)$$

From page 207 of Ramo, Whinnery and Van Duzer, we recognize solutions to this equation as

$$\hat{\delta}(r) = A J_1 \left[ \left( \frac{\rho \omega^2}{G} - k^2 \right)^{1/2} r \right] + B N_1 \left[ \left( \frac{\rho \omega^2}{G} - k^2 \right)^{1/2} r \right] \quad (f)$$

On page 209 of this reference there are plots of the Bessel functions  $J_1$  and  $N_1$ . We must have  $B = 0$  as at  $r = 0$ ,  $N_1$  goes to  $-\infty$ . Now, at  $r = R$

$$\hat{\delta}(R) = 0 \quad (g)$$

This implies that

$$J_1 \left[ \left( \frac{\rho \omega^2}{G} - k^2 \right)^{1/2} R \right] = 0 \quad (h)$$

If we denote  $\alpha_1$  as the zeroes of  $J_1$ , i.e.

$$J_1(\alpha_1) = 0$$

we have the dispersion relation as

$$\frac{\rho}{G} \omega^2 - k^2 = \frac{\alpha_1^2}{R^2} \quad (i)$$